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# **Vehicle Technology Assessment, Model Development, and Validation of a 2018 Honda Accord LX With a 1.5L I4 and Continuously Variable Transmission**

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## Definitions and Abbreviations

<b>Acronyms</b>	<b>Description</b>
°C or C	degrees Celsius
°F or F	degrees Fahrenheit
2WD	two-wheel drive
4WD	four-wheel drive
AC	air conditioning
AKI	anti-knock index
AMTL	Advanced Mobility Technology Laboratory (Argonne)
Autonomie	Argonne full vehicle simulation software <a href="https://www.autonomie.net/">https://www.autonomie.net/</a>
Argonne	Argonne National Laboratory
ASR	absolute speed change rating
AVTE	Advanced Vehicle Testing Evaluation (Previous U.S. DOE activity)
BEV	battery electric vehicle
BTU	British thermal unit
CAN	controller area network
CAFE	Corporate Average Fuel Economy
cc	cubic centimeter
ccps	cubic centimeters per second
CEd	positive driven cycle energy
CFR	Code for Federal Regulation
cm	centimeter
CO	carbon monoxide
CO <sub>2</sub>	carbon dioxide
D3	Downloadable Dynamometer Database ( <a href="http://www.anl.gov/d3">www.anl.gov/d3</a> )
DAQ	data acquisition system
deg	degree
DFCO	deceleration fuel cutoff
DFI	direct fuel injected
DI	direct Injection
DOHC	double overhead cam
DOT	U.S. Department of Transportation
DR	distance rating
EGR	exhaust gas recirculation system
EPA	U.S. Environmental Protection Agency
ER	energy rating
EER	energy economy rating
FTP	Federal test procedure (EPA defined)
gps	grams per second
HC	hydrocarbon
HEV	hybrid electric vehicle
hp	horsepower
Highway or HWFET	EPA certification testing: Highway dynamometer driving cycle

<b>Acronyms</b>	<b>Description</b>
Hz	hertz
inH <sub>2</sub> O	inches of water
inHg	inches of mercury
kPa	kilopascal
kph	kilometer per hour
kW	kilowatt
L	liter
LA92	California unified driving schedule
lb/lbs	pound(s)
lb-ft	Foot pounds
lbm	pound-mass
LHV	lower heating value
m	meter
MBT	maximum brake torque
mg	milligrams
mpg	mile(s) per gallon
mph	miles per hour
N	newton
NA	naturally aspirated
Nm	newton-meters (torque)
NO <sub>x</sub>	oxides of nitrogen
PFI	port fuel injected
RMS	root mean squared
rpm	rotations per minute
RWD	Rear wheel drive
s	second
SAE	Society of Automotive Engineers
SC03	EPA certification test (air conditioning test)
scfm	standard cubic feet per minute
SSS	steady speed stairs
TCC	torque converter clutch
TCU	transmission control unit
UDDS	EPA certification test: Urban dynamometer driving schedule (FTP-72)
US06	EPA certification test: US06 dynamometer driving schedule
Volpe	Volpe National Transportation Systems Center
VSR	Vehicle Systems Research
VTC	valve timing control
VTEC	variable valve timing and lift electronic control
V	volts

<b>Symbols</b>	<b>Description</b>
$F_{chassis}$	force obtained from the dynamometer
$J_{TC}$	inertia of torque converter
$P_{acc_{mech}}$	power of accessory load
$r_t$	radius of tire
$R_{xy}(\tau)$	cross-correlation over the range of lags between two signals $(x, y)$
$T_{acc_{mech}}$	torque of accessory load
$T_{eng}$	torque of engine
$T_{fd,in}$	torque in of final drive
$T_{fd,out}$	torque out of final drive
$T_{gb,in}$	torque in of gearbox
$T_{gb,out}$	torque out of the gearbox
$T_{ratio}$	torque ratio of torque converter
$T_{TC,in}$	torque in of torque converter
$T_{TC,out}$	torque out of torque converter
$T_{trq_{cpt},out}$	torque out of torque-coupling
$T_{wheel,brake}$	brake torque of wheel
$T_{wheel,loss}$	torque loss of wheel
$T_{wheel,out}$	torque out of wheel
$v_{chassis}$	linear speed of vehicle
$\gamma_{fd}$	ratio of the final drive
$\eta_{fd}$	transfer coefficient of final drive
$\tau$	displacement, also known as lag
$\omega_{eng}$	rotational speed of engine
$\omega_{gb,out}$	rotational speed out of gearbox
$\omega_{ratio}$	speed ratio of turbine speed to impeller speed for torque converter
$\omega_{TC}$	rotational speed of impeller for torque converter
$\omega_{wheel}$	rotational speed of wheel

## 1. Introduction

The U.S. Department of Transportation's National Highway Traffic Safety Administration sets Corporate Average Fuel Economy (CAFE) standards for passenger cars, light trucks, and medium-duty vehicles. NHTSA has contracted Argonne National Laboratory to conduct full vehicle simulations using Argonne's Autonomie software ([www.autonomie.net/](http://www.autonomie.net/)), to provide input into the CAFE model to determine the contribution of vehicle technologies on fuel economy. To model and simulate the new technologies employed in vehicles, it is critical to consistently update and validate Autonomie using vehicle and component test data. For the past 20 years Argonne's Advanced Mobility Technology Laboratory (AMTL) has been providing the laboratory vehicle test data to support Autonomie modelling and validation.

The vehicle benchmarked in this report is a 2018 Honda Accord equipped with the 1.5 liter, inline four cylinder "Earth Dreams" engine coupled to a continuously variable automatic transmission. This powertrain configuration is acclaimed for providing favorable fuel economy results while still providing capable vehicle performance [1]. The focus of the evaluation is to understand the use of critical powertrain components and their impact on vehicle energy use and efficiency. The test vehicle was instrumented to characterize the operating conditions of critical components as well as understand the overall control strategy. Standardized tests were performed on a chassis dynamometer in a controlled laboratory environment across a range of certification tests, and other testing conditions relevant for model development and validation. Furthermore, additional testing focused on characterization of different powertrain components performance and control (e.g., shifting).

The analysis in this report is separated into several sections:

- Vehicle instrumentation and setup.
- Vehicle energy consumption and efficiency testing results across a wide range of standard (e.g., U.S. certification, acceleration, steady-state) and specialized (e.g., constant pedal position) driving cycles across different thermal conditions
- System and component control analysis
- Autonomie model development and validation.

In addition to this report, the dataset will be made publicly available through the Advanced Mobility Technology Laboratory's Downloadable Driving Database (D3) at [www.anl.gov/d3](http://www.anl.gov/d3).



## 2. Test Vehicle Description

### 2.1. Vehicle Specifications

The 2018 Honda Accord, the tenth generation of the Accord model, is constructed on a refined modular platform and features a turbocharged, 1.5 L, in-line four-cylinder engine with variable valve timing and lift electronic control (VTEC) paired with a continuously variable transmission [1]. The Honda’s VTEC turbo technology is marketed as part of the cluster of new generation of powertrain technologies known as “Earth Dreams Technology.” The manufacturer states that the engine achieves the fuel economy benefits of a small engine while maintaining drivability by combining the turbo charger with a direct injection system and variable valve timing mechanism [2]. Dual valve timing control (VTC) and stable combustion further reduce pumping losses and yield lower fuel consumption across a wider load range [2]. An overview of the test vehicle’s technical specifications is shown in Table 1.

*Table 1: Technical specifications of the MY2018 Honda Accord LX test vehicle [1]*

<b>Test vehicle</b>	2018 Honda Accord sedan LX/1.5 L I4 with continuously variable transmission
<b>VIN</b>	1HGCV1F14JA056668
<b>Engine</b>	1.5 liter, I4, single-scroll turbocharger, DOHC VTEC ® 16V 143 kW (192 hp) @ 5,500 rpm, 260 Nm (192 lb-ft) @ 1,600-5,000 rpm Redline- 6,500 rpm Compression ratio 10.3 :1 Direct injection
<b>Transmission</b>	CVT with overdrive Ratio range: 2.645-0.405 Differential gear ratio: 5.36 225/50 R17 tires
<b>Climate control</b>	Dual-zone automatic climate control Belt-driven air conditioning compressor R-1234yf refrigerant
<b>EPA label fuel economy (mpg) <sup>1</sup></b>	30 city/38 hwy/33 combined

The full vehicle build details can be found in the test vehicle’s Monroney label, or window sticker, in Appendix A: Vehicle Build Sheet.

### 2.2. Key Technology Features

The 2018 Honda Accord was produced with a new generation of Honda internal combustion engines part of “Earth Dream Technology:” a direct-injection in-line 4-cylinder, 1.5 liter, 16-valve, dual overhead cam engine (Honda Motor Company, n.d.). Development objectives with this downsized, turbocharged engine were set to ensure a balance of high power and high torque while lowering fuel consumption and emissions to the top level of the vehicle class (Nakano et

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<sup>1</sup> Data from fueleconomy.gov

al., 2016). The engine uses the following technologies (Honda Motor Company, n.d.), (Nakano et al., 2016).

- Direct injection technology
  - o In-cylinder direct injection with high-pressure multi-hole injector between intake valves
  - o Reduced friction for increased efficiency and response at high torque
- Rapid combustion technology
  - o Optimized cylinder port shape for more stable combustion
  - o High tumble intake port to strengthen air flow
  - o Strong tumble flow during cold starts
  - o Higher compression ratio and optimized ignition timing to raise thermal efficiency even under high load
- Turbocharger with electronic wastegate
  - o Increased responsiveness
  - o Increased fuel economy under low-load conditions
- Dual VTC technology
  - o Low engine speed/high load range: Valve overlap is increased to leverage the scavenging effect, raise charging efficiency, and suppress knock, while as the same time the scavenging effect increases turbine speed.
  - o Medium engine speed range: The valve overlap limits are reduced, and the electric wastegate is controlled to raise the turbocharged pressure
  - o Low engine speed/low load range, and high engine speed/all load: Valve overlap is reduced to increase combustion stability by reducing the internal EGR amount, lower exhaust resistance
  - o High engine speed/all load: Valve overlap is reduced to increase combustion stability by reducing the internal EGR amount, lower exhaust resistance and heighten knock resistance
- Water-cooled exhaust manifold built into cylinder head
  - o Reduced high-load exhaust gas temperature, increasing fuel economy

In addition the 2018 Honda Accord is equipped with a continuously variable transmission (CVT) that features the following technologies.

- Overview of design [1]:
  - o Pump, turbine and stator assembly in a single unit
  - o Four parallel shafts: input shaft, drive pulley shaft, driven pulley shaft, and final drive shaft
    - Input shaft connects to the torque converter turbine and integrates the forward clutch, connecting the drive pulley shaft end
    - Drive pulley shaft includes CVT drive pulley, a moveable and fixed-face pulley
    - Driven pulley shaft includes CVT driven pulley, a moveable and fixed-face pulley

- Six-position shift lever with multiple modes
    - Park/Reverse/Neutral/Drive/Sport/Low
      - Sport mode: non-stage speeds (standard stepped transmission behavior not emulated), transmission keeps engine at higher rpm than in drive mode
      - Low mode: engine braking and power for climbing, transmission shifts to lowest pulley ratio
    - Transmission lock-up: operates in drive, sport, and low positions
    - Step-shift mode: while accelerator pedal is depressed deeply, engine speed reaches maximum, and vehicle speed is high, CVT switches to an automatic transmission like multistage gear shift
  - Engine driven transmission fluid pump
- Operation highlights of Honda continuously variable transmissions [3], [4]:
- Expanded ratio range over prior generation CVTs providing:
    - Increased fuel efficiency performance (high ratio locked)
    - Guaranteed driving force (low ratio locked)
  - Advanced control vane-type oil pump with half-feed switching and expansion of half-feed operation
  - Transmission fluid with high metal friction coefficient (HCF2)
    - Increased transmission capacity
    - Reduced fuel consumption and increased fuel efficiency
  - Optimization of pulley V-surface properties
  - Protective control technologies
    - Engine speed control (restricted to max transmission fluid temperature)
    - Engine torque cooperative control
    - CVTF temperature control

### **2.3. Comparison Vehicles and Preliminary Analysis**

The Honda Accord is a top-selling model in the mid-sized vehicle category. This section provides a brief comparison of the 2018 Honda Accord with historical trends in this category and other vehicles released in the midsized non-luxury vehicle category for the 2018 model year. The non-hybrid 2018 Honda Accord was offered in four trim levels: LX, EX-L, Sport, and Touring. All trim levels are equipped with a 1.5 L turbocharged engine, with the EX-L, Sport, and Touring levels offering an optional turbocharged 2.0 L I4. Following a joint review of possible powertrain configurations with the project sponsors, the 2018 Honda Accord LX with the turbocharged 1.5 L engine and continuously variable transmission was selected for this research.

This 2018 Honda Accord LX test vehicle has a curb weight of 3131 lbs, with a gross vehicle and equivalent test weight of 3,500 lbs. To provide insight into trends for similar vehicles in this category, the test vehicle was compared with cars of a similar weight. To this end, the 2017 EPA fuel economy trends report [5] provides a glimpse into the historical trends from 1975 until 2020 for similar cars within the weight class of 3,500-4,000 lbs. The trend of average fuel economy, with the specific test vehicle shown as a star, can be seen in Figure 1.

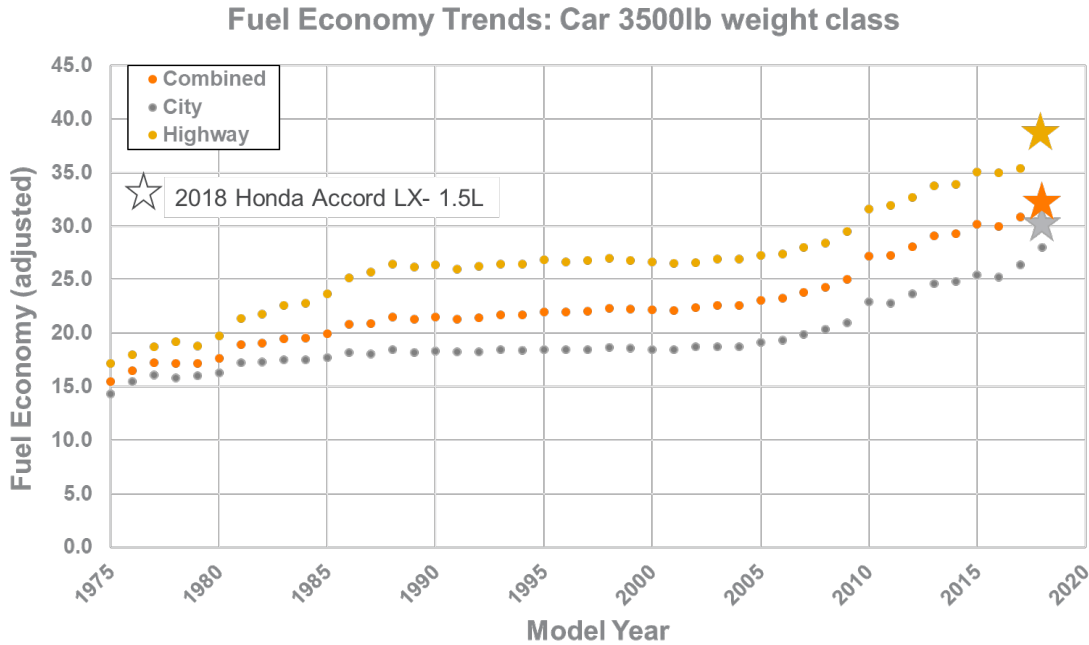


Figure 1: Fuel economy trends: car 3,500 lb weight class

Combined fuel economy for midsize passenger cars has been steadily increasing from 25 mpg since 2009. The 2018 Honda Accord 1.5 L, as the first year of a new generation of Honda powertrains, provides insight into how this trend will likely continue. Improvements were found in both city and highway fuel economy, with the highway cycle fuel economy results demonstrating the greatest increase over the historical trend.

Beyond historical trends of vehicles in a similar weight category, there are benefits in comparing the test vehicle with other vehicles within the model year (MY) 2018 midsize category. For this comparison, vehicles of similar vehicle class, with a starting manufacturer's suggested retail price (MSRP) below \$25,000, were considered. With vehicles selected based on these broad criteria, all trim levels were then considered based on data available in the EPA vehicle test car list database [6]. (A subset of selected vehicles used for this comparison can be found in Appendix B: Subset of Midsize Cars for Comparative Analysis.) The resulting list of comparable midsize sedans from the 2018 model year is summarized in the list below.

- Buick Regal
- Chevrolet Malibu
- Ford Fusion
- Kia Optima
- Honda Accord
- Hyundai Sonata
- Mazda 6
- Nissan Altima
- Subaru Legacy
- Toyota Camry
- Volkswagen Passat

The vehicle weights varied considerably, as optional powertrains (with the exception of hybrids) and trim levels were also considered. The distribution of weight and available power for the vehicles reviewed is shown in Figure 2.

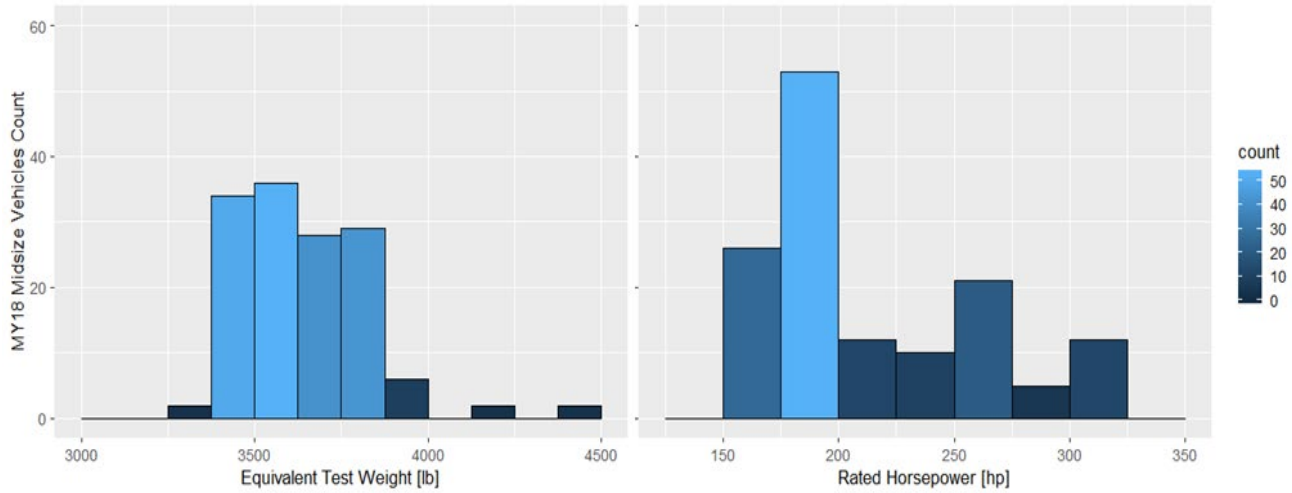


Figure 2: Summary distributions of weight and horsepower of mid-size cars reviewed

The 2018 Accord 1.5 L LX test vehicle weight was near the mean of the category, with an equivalent test weight of 3,500 lbs, and at 192 hp (143 kW), the power available was slightly below the mean of 213.2 hp (159 kW). Fuel economy in this category varies considerably by powertrain and trim selection. The fuel economy values published by manufacturer are termed “adjusted fuel economy values”, as the observed (unadjusted) fuel economy from vehicle dynamometer is adjusted downward based on established procedures [6]. A comparison of the unadjusted fuel economy (FE), separated by air induction system category, can be seen on the fuel test procedure (FTP) cycle in Figure 3.

Distribution of MY2018 Unadj. FTP FE (mpg)

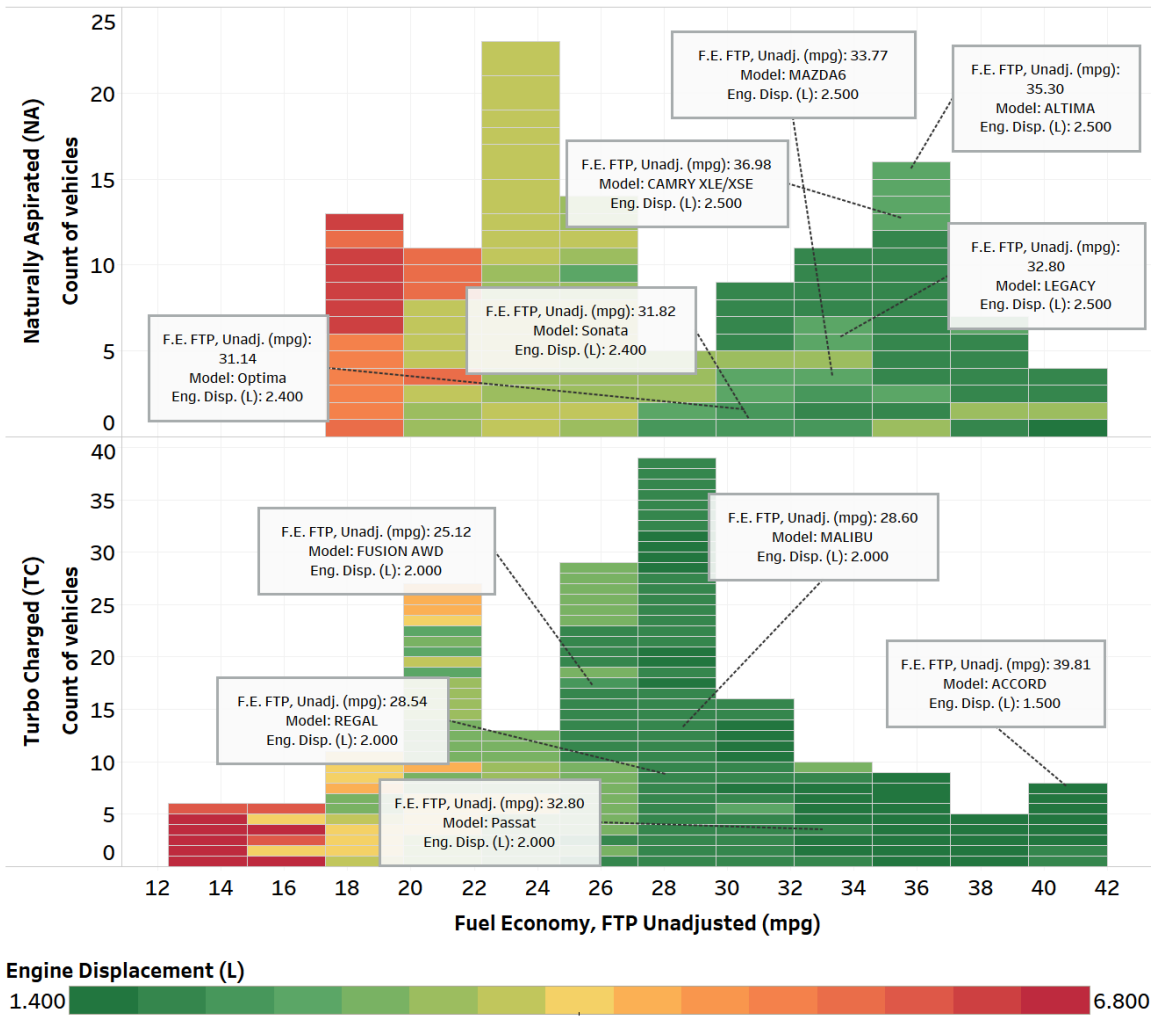


Figure 3: FTP unadjusted fuel economy (FE) of 2018 midsize vehicles

As shown in Figure 4 the 2018 Honda Accord LX with CVT is amongst the most fuel efficient vehicles in its class both on the FTP and HWFET cycles.

Distribution of MY2018 Unadj. FTP FE (mpg)

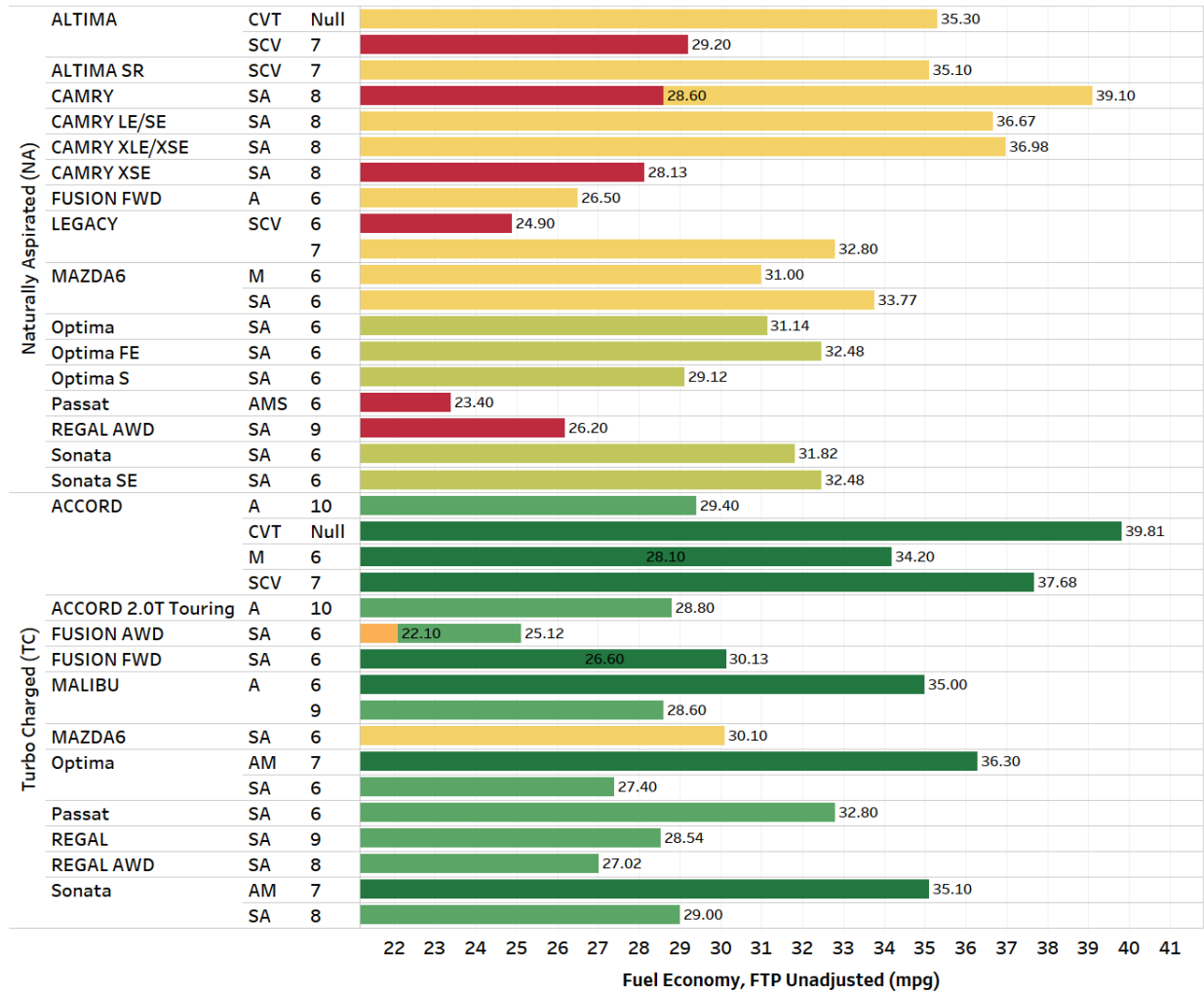


Figure 4: FTP unadjusted fuel economy of 2018 midsize vehicles by vehicle

The 2018 Honda Accord unadjusted fuel economy is also on the upper end of the sample set on the reported highway fuel economy driving (HWFET) cycle, as shown in Figure 5 and Figure 6.

### Distribution of MY2018 Unadj. HWFET FE (mpg)

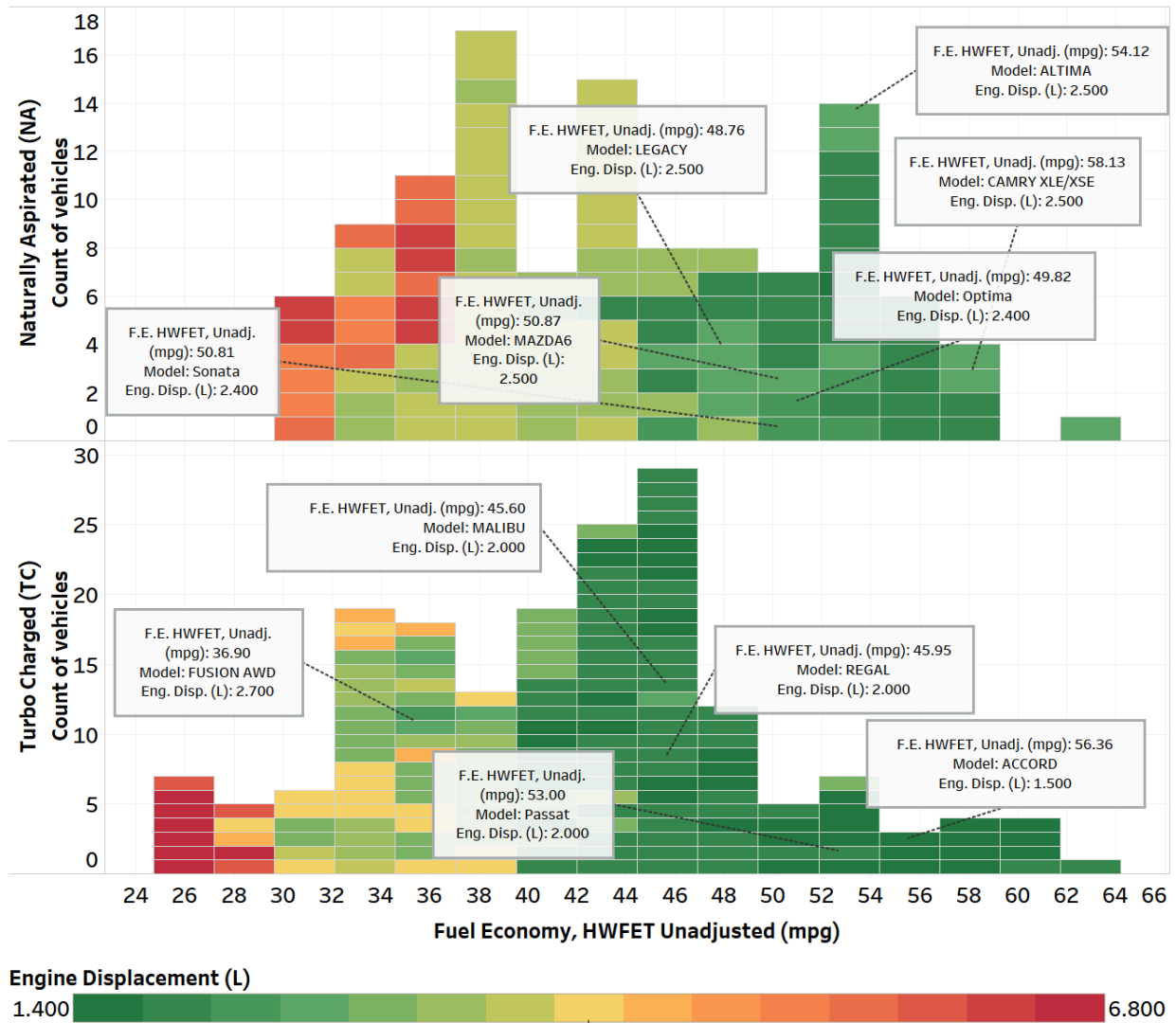
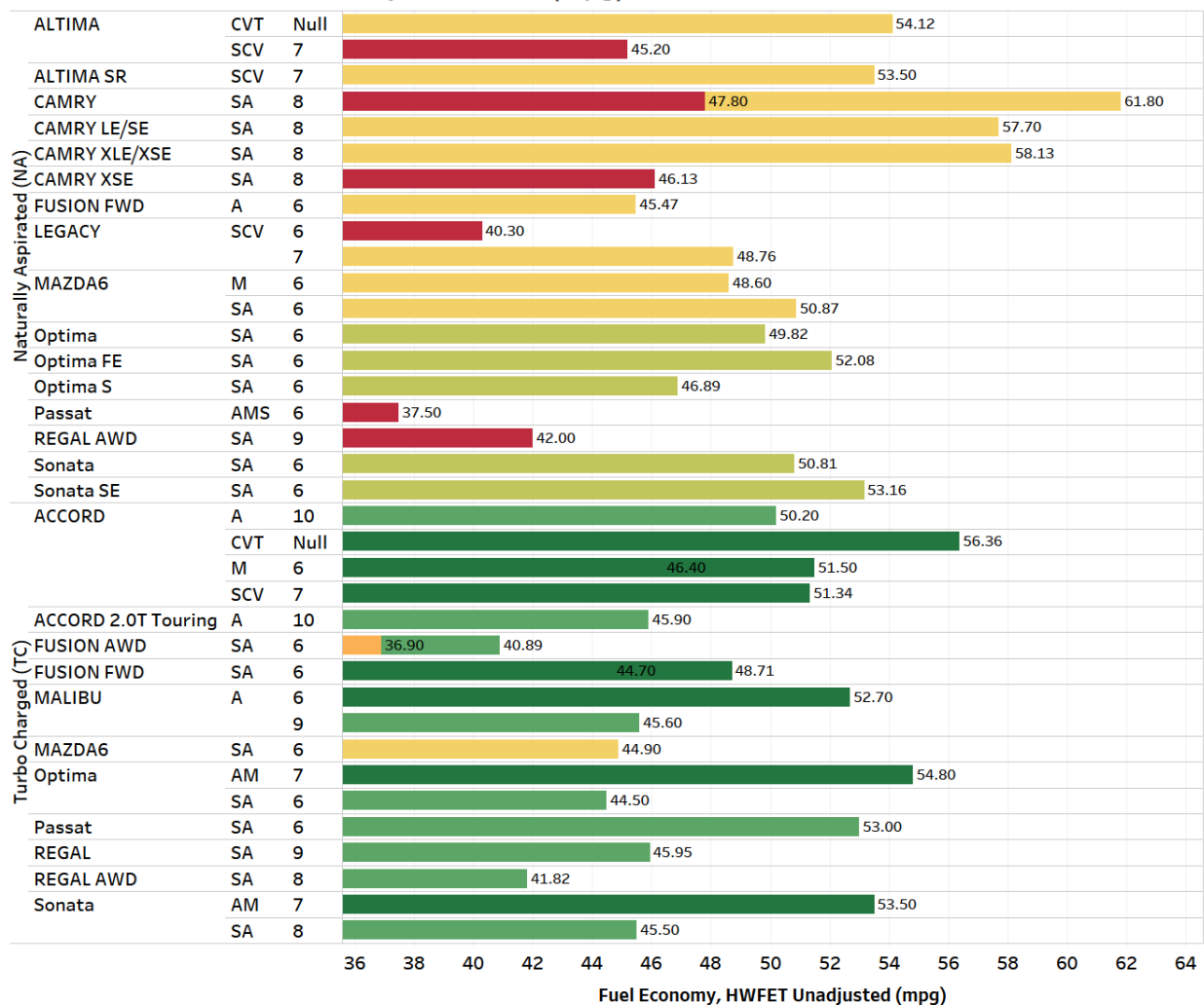


Figure 5: Highway unadjusted fuel economy of 2018 midsize vehicles



### Distribution of MY2018 Unadj. HWFET FE (mpg)



#### Engine Displacement (L)

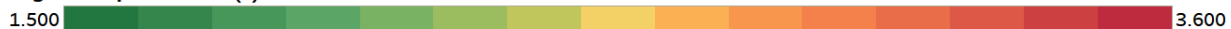


Figure 6: Highway unadjusted fuel economy of 2018 midsize vehicles by vehicle selected

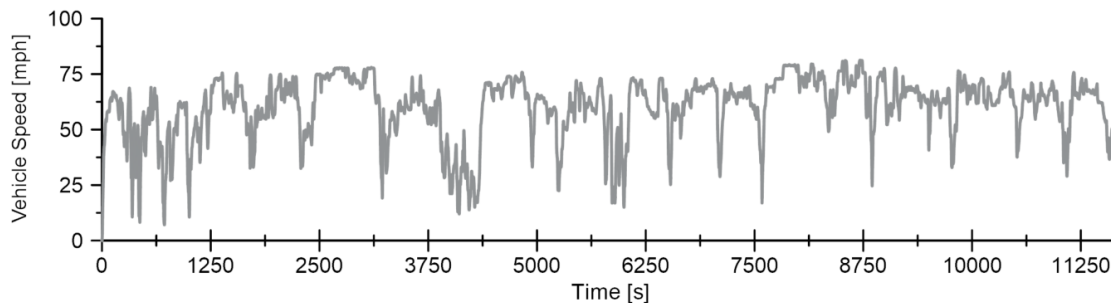
When compared to other vehicles in the market, the 2018 Honda Accord demonstrates high fuel economy on both the HWFET and FTP cycles with the vehicle weight and engine power slightly lower than the mean of the vehicles compared in the category. The following sections describe the vehicle and component operating conditions that led to those vehicle fuel economy values.

### 3. Testing Overview

#### 3.1. Vehicle Break-In

A new vehicle must be “broken in” for stability, for consistent vehicle losses to tires and moving and rotating components, and to ensure catalyst “degreening.” An established industry standard for proper vehicle break-in is 4,000 miles, as required in the Code for Federal Regulations, Title 40, Part 86 [7][8]. On the test vehicle, this preliminary 4,000 miles were completed through a combination of on-road and on-dynamometer operation. Controller area network (CAN)-based vehicle instrumentation was completed prior to break-in, providing data for preliminary results and instrumentation validation and refinement. The preliminary vehicle mileage accumulation up to 2,500 miles was completed on transient drive cycles on a chassis dynamometer, in order to expedite the vehicle evaluation. Following this, on-road mileage accumulation of 1,500 miles ensured proper break-in of vehicle tires and other rotating components, in addition to collecting on-road data on vehicle operation.

A key component of an effective break-in is variation in powertrain speed and loading. Break-in miles accumulated on-road inherently provide this variability, but operation on a chassis dynamometer depends on the driving cycle completed. To ensure variability while accumulating miles on a dynamometer, several custom drive cycles were created based on collected on-road data, with varying acceleration rates and speeds. An example of a custom drive trace is shown in Figure 7.



*Figure 7: Drive cycle developed from on-road data for on-dyno mileage accumulation*

Vehicle operation during the on-dynamometer mileage accumulation was performed by a custom-built robot driver, allowing for consistent mileage accumulation while reducing project burden. The test vehicle during mileage accumulation can be seen in Figure 8.



*Figure 8: Vehicle mounted for mileage accumulation on the AMTL 2WD chassis dynamometer*

### **3.2. Vehicle Dynamometer Setup**

The following sections provide details of the vehicle setup and an overview of the test methodology specific to this test vehicle. Further information regarding the methods of vehicle testing, please review the general procedures document for the facility [9]. The test vehicle in the 4WD chassis dynamometer during testing at the AMTL can be seen in Figure 9.



*Figure 9: Vehicle mounted for full testing inside the AMTL 4WD chassis dynamometer*

### 3.3. Instrumentation

#### 3.3.1 Facility Signal Overview

Figure 10 shows the general instrumentation process for technology evaluation of conventional vehicles such as the 2018 Honda Accord. This process integrates data streams from several sources.

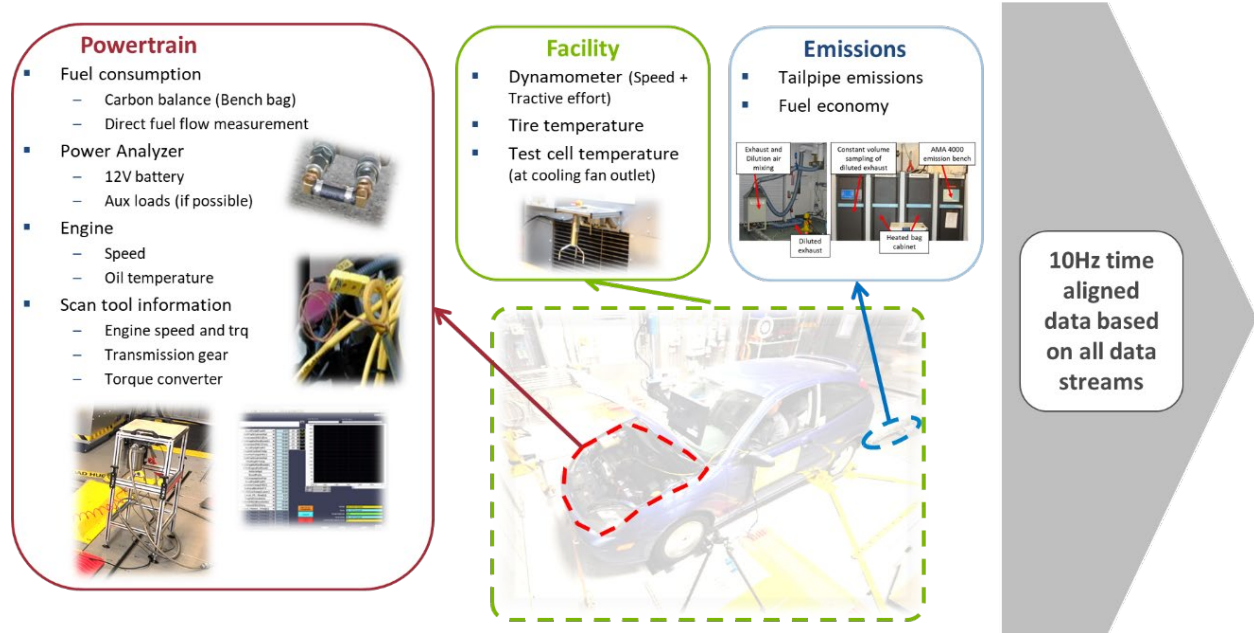


Figure 10: Overview of general instrumentation for conventional vehicle

The facility data (Table 2) captures the test cell conditions (ambient test cell temperature and relative humidity), the dynamometer data (vehicle speed and tractive effort) and emissions data (bag and modal bench data: HC, CO, NO<sub>x</sub>, and CO<sub>2</sub>). Fuel consumption is measured in several different ways. A carbon balance fuel economy result from the emissions bench (bag and modal) is used to provide a standard method of fuel economy calculation. To provide greater insight during transient operation, the vehicle was equipped with additional inline fuel flow meters.

Table 2: Standard data streams collected for all vehicles tested at Argonne’s Advanced Mobility Technology Laboratory

Facility data	Drive cycle input	Emissions data	Generic vehicle data
Dyno_Spd (mph)	Drive_Schedule_Time (s)	Dilute_CH4 (mg/s)	Engine_Oil_Dipstick_Temp (C)
Dyno_TractiveForce (N)	Drive_Trace_Schedule (mph)	Dilute_NOx (mg/s)	Cabin_Temp (C)
Dyno_LoadCell (N)	Exhaust_Bag	Dilute_COlow (mg/s)	Tire_Rear_Temp (C)
DilAir_RH (%)		Dilute_COmid (mg/s)	Tire_Front_Temp (C)
Tailpipe_Press (in H <sub>2</sub> O)		Dilute_CO2 (mg/s)	
Cell_Temp (C)		Dilute_HFID (mg/s)	
Cell_RH (%)		Dilute_NMHC (mg/s)	
Cell_Press (inHg)		Dilute_Fuel (g/s)	

Additional analog signals include thermocouples measuring the air temperature behind the radiator and the engine bay temperature.

### 3.3.2 Fuel Flow Measurements (Scale, Coriolis, Modal, Bag)

The 1.5 L I4 Earth Dreams engine uses a direct injection (DI) system for fuel injection. On the test vehicle, total fuel flow was measured using two fuel flow meters routed in series at the fuel connection on the engine firewall. Fuel was first routed to a Coriolis flow meter and then to a positive displacement fuel scale before returning to the fuel rail at the high-pressure fuel pump inlet. It should be noted that the addition of the hosing required for the direct fuel flow measurements results in some delay and dynamic effect on the direct fuel flow measurements. These effects are taken into account during post-processing. Figure 11 and Figure 12 illustrate the fuel system instrumentation of the test vehicle.

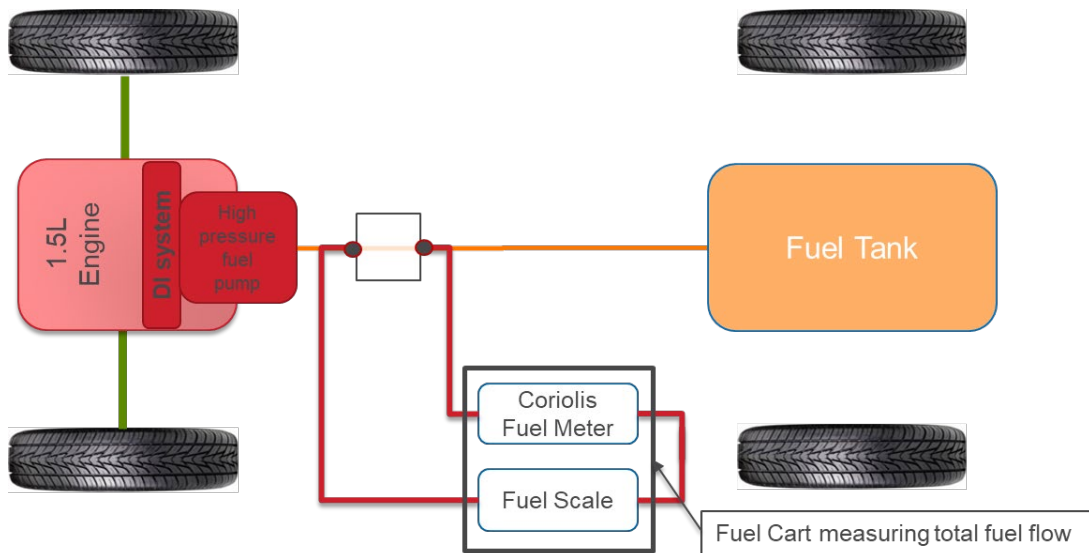


Figure 11: Instrumentation overview of direct fuel injection system on 2018 Honda Accord



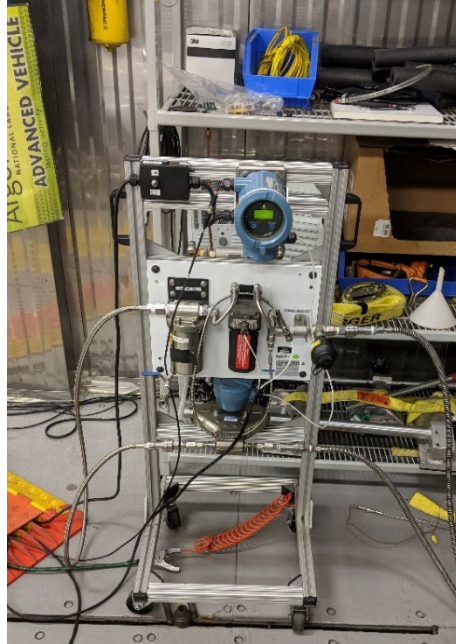


Figure 12: Direct fuel flow measurements via fuel scale and Coriolis flow meters

### 3.3.3 Hioki Power Analyzer Setup

Vehicle electrical system measurements were captured with a four-channel Hioki 3390-10 power analyzer. Three channels were instrumented, each with a direct current measurement with Hioki CT6843 200A current probes. These current probes were located to capture the current of the alternator, 12V loads, and the 12V battery negative terminal. Voltage for the low voltage bus was measured across the 12V battery, which was then bridged to act as the voltage source for all three channels. From the measured current and voltage channels, power and energy use were calculated within the analyzer. An overview of vehicle wiring can be seen in Figure 13.

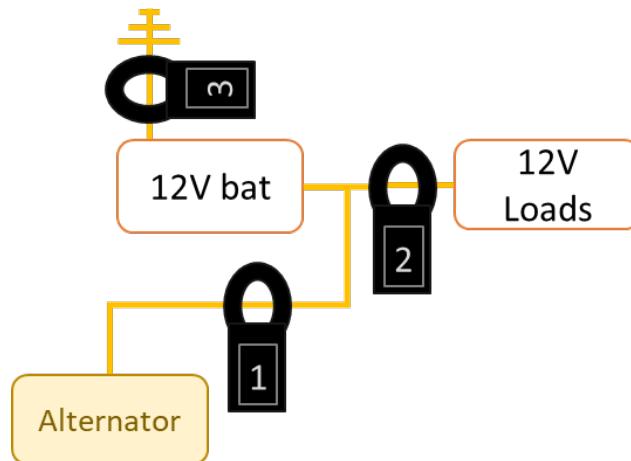


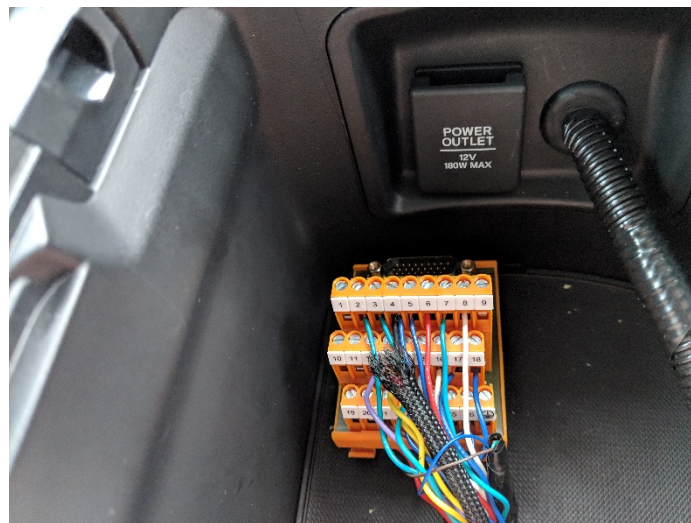
Figure 13: Wiring of Hioki power analyzer on the 2018 Honda Accord test vehicle

### 3.3.4 CAN Signals

A core capability of the AMTL staff is the ability to decode the vehicle and powertrain internal communication messages (CAN messages). Over the past few years, AMTL staff have developed powerful tools that enable the decoding of both broadcast and diagnostic CAN messages. These tools rely on an understanding of CAN messaging structure, the correlation of changes in CAN messages to known instrumentation signals, and the ability to use the chassis dynamometer environment to safely control planned scenarios to enable the decoding of certain signals.

Capturing communication signals, whether broadcast or diagnostic, directly from the vehicle can provide a considerable amount of data that would otherwise be unattainable due to the challenges of instrumentation and the high costs associated. Once determined, these signals provide key insight into component control and operation. Though these signals offer the mentioned benefits, they do have a higher level of signal specific uncertainty as the data is developed internally at the manufacturer and varies based on the specific signals and sensors. Due to this, Argonne staff validate signals to the greatest extent possible through independent instrumentation and calculation of correlating results of similar signals.

The team decoded a significant list of vehicle messages for the vehicle, which is detailed in Appendix C: 2018 Honda Accord LX Test Signals. This instrumentation included the determination and probing of eight separate CAN networks across the vehicle. Each connection was then routed to an accessible location with a single connection in the center console for external instrumentation, as shown in Figure 14: CAN breakout on the 2018 Honda Accord LX.



*Figure 14: CAN breakout on the 2018 Honda Accord LX*

The corresponding logging and communication of CAN messages was completed through a combination of custom scripting with Intrepid Control Systems Vehicle Spy software and National Instruments LabVIEW software located on the AMTL custom built data acquisition (DAQ) system. The following is a categorized list of critical signals decoded on the vehicle communication bus, from either diagnostic or broadcast CAN messaging:

- Driver input:
  - Accelerator pedal position (multiple signals)
  - Brake pedal (multiple signals)
  - Eco mode selection
  - Transmission PRNDL selection
  - HVAC system settings and states
- Engine:
  - Engine torque
  - Engine speed
  - Intake air temp
  - Throttle valve angle
  - Knock retard
  - Spark advance
  - Air fuel ratio
  - Deceleration fuel cutoff state
  - Turbocharger boost pressure
  - Turbocharger bypass state
  - Exhaust catalyst temperature
- Cooling system
  - Engine coolant temperature
- Transmission
  - Transmission temperature
  - Pulley ratio
  - Turbine speed
  - Secondary shaft speed
  - Torque converter lockup operation

The complete signal list can be found in Appendix C: 2018 Honda Accord LX Test Signals, and the datasets can be found at Argonne’s Downloadable Dynamometer Database at [www.anl.gov/D3](http://www.anl.gov/D3).

### **3.4. Test Plan**

#### **3.4.1. Lists of Tests Conducted**

A test plan was developed to provide a broad base of vehicle operation at varying driving conditions and test temperatures. To capture test to test variability, specific drive cycles such as the UDDS, HWFET, and US06 were repeated. Custom tests for mapping, those with a focus on specific ambient temperatures, or those on Tier 2 certification fuel, were only performed a single time to allow for a greater range of testing within the allocated project budget. A summary of the testing performed can be found in Table 3, though more details on the specific test cycles can be found in CFR[7][8], or the reference AMTL testing overview report [9].



Table 3: Summary of the number of standard test cycles in the general test plan

		35 °C + Solar Emulation*	-7 °C	23 °C Tier 2 fuel
<b>UDDS x 3 (cold start/hot/hot)</b>	3x	1x	1x	1x
<b>HWFET (pair- prep/test)</b>	3x	2x	1x **	1x
<b>US06 (pair- prep/test)</b>	3x	2x	1x	1x
<b>SC03 (pair- prep/test)</b>		2x		
<b>Steady state speed testing at 0%, 3% 6% grade</b>	1x	1x		1x
<b>Passing 0%, 3%, 6% grade</b>	1x			1x
<b>Wide open throttle (WOT) x 3</b>	1x			1x
1x- test was conducted a single time 2x- 2 tests were completed 3x – 3 tests were completed *: Solar loading during all tests set to the level of 850 W/m <sup>2</sup> **: Highway cycles were completed as a series of three to ensure thermal stability at low temperature				

In addition to the test matrix listed in Table 2, focused testing was included to provide further insight into vehicle energy consumption and operation. The additional testing included the following.

- 23 °C cold-start idle: mapping out the idle fuel flow as a function of powertrain temperature
- 23 °C cold start LA92
- 23 °C hot start JC08
- 23 °C cold start US06
  - Varying engine and transmission mapping cycles through:
  - Constant accelerator tip-ins tests
  - Accelerator tip-ins with vehicle locked at constant speed
- Additional 23 °C testing:
  - Vehicle maximum acceleration at an emulate road grade of 25%
  - Cold start engine idle fuel flow at idle

The table in Appendix D: Test Summary, summarizes all tests performed.

### 3.4.2. U.S. Standard Driving Cycles

The fuel economy testing focused on the UDDS, HWFET, and US06 (high acceleration aggressive) drive cycles at 23 °C ambient temperature condition. The test sequence includes a cold start UDDS, a hot start UDDS, a third UDDS, a HWFET pair, and a US06 pair. The preparation for the cold start test consists of completing a UDDS cycle at 23 °C and leaving the vehicle to thermally soak at 23 °C for over 12 hours. As the vehicle remains in the test cell for the duration of testing, the overnight soak prior to each sequence of tests is completed with the vehicle already mounted to the chassis dynamometer. The graph in Figure 15 shows the sequence of drive cycles executed. Note that there is a 10-minute soak period between the first two UDDS cycles, while a soak time of slightly over 10 minutes occurs between the second and third UDDS cycles due to variations of emissions bag analysis and DAQ processing. As described in Table 2, this series was repeated three times to capture test-to-test variability at the test temperature of 23 °C on the Tier 3, low octane fuel.

Unless otherwise noted, the fuel economy numbers in this report are based on analysis of the test phases highlighted by the pink boxes in Figure 15. The test phases follow a prep phase of the same cycle (not highlighted), which acclimates the vehicle to the test conditions, a process described in the CFR [7]. The US06 drive cycle phases are the split city and highway phases needed to calculate the EPA five-cycle fuel economy label.

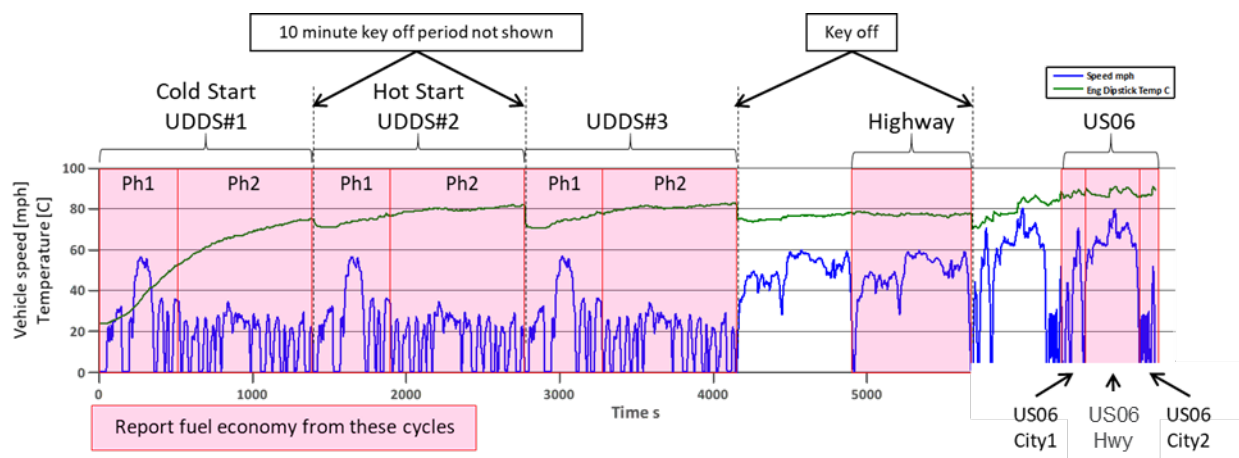


Figure 15: Daily drive cycle test sequence executed in the morning

### 3.4.3. Additional Testing

Determination of component and controls operation and limitations is best realized by focused testing in which vehicle operation can be controlled. This section will provide an overview of the methods and tests developed specifically for the 2018 Honda Accord. Additional operational testing discussion, and further details on the development of these custom cycles, can be found in a supplemental report [9].

#### 3.4.3.1. Steady State Speeds

Steady-state speed tests evaluate vehicle operation while the vehicle operates at a constant speed and load point. Steady-state cycles are conducted by following a driving schedule with a minimum 30-second hold at each speed. Vehicle speed is increased in 10 mph increments up to 80 mph while held at each speed for a set period of time, and then decreased from 80 mph

repeating the measurements. By holding each speed following both an acceleration and deceleration, one captures variability in powertrain operation (such as in commanded gear ratio) and thermal state. Additionally, these steady-state cycles may be repeated at varying grades to capture variations in vehicle powertrain loading.

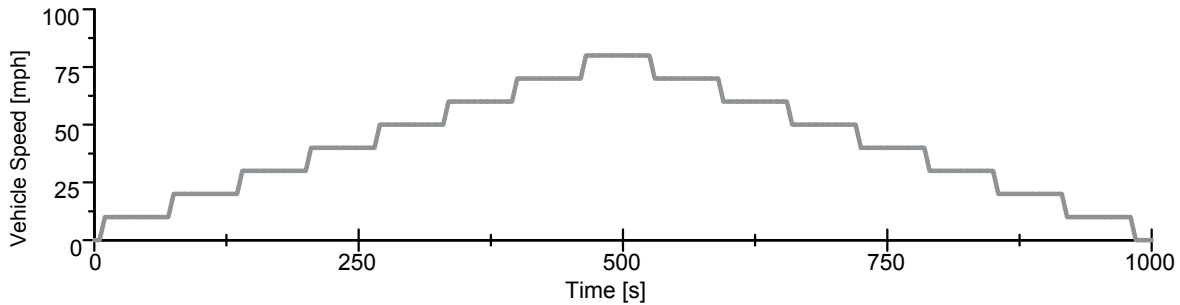


Figure 16: Overview of steady state drive cycle with preparation

Prior to each steady state speed cycle, the vehicle is warmed to an engine oil temperature of over 80 °C, or to a temperature recorded on a prior transient drive cycle. The 2018 Honda Accord, steady state speed cycles were performed at the ambient test temperatures of 23 °C (0% grade, both fuels), and 35 °C (0%, 3%, and 6% grade), seen in Appendix D: Test Summary.

### 3.4.3.2. Powertrain Mapping Cycles

Full vehicle powertrain operation across its speed and load range are not commonly seen during operation on standard transient drive cycles. To fully map powertrain operation, supplemental custom cycles, a robotic driver, and feedback from focused instrumentation are used to control the vehicle and precisely map component operation. This mapping was performed using several tests. The first test consisted of the dynamometer being placed in road load simulation mode (the same dynamometer mode used on certification drive cycles), and accelerated with fixed accelerator pedal inputs, as can be seen in Figure 17 below. It should be noted that a limit on the chassis dynamometer limited maximum vehicle speed to 85 mph, which can be seen at test time beyond ~3,600s.

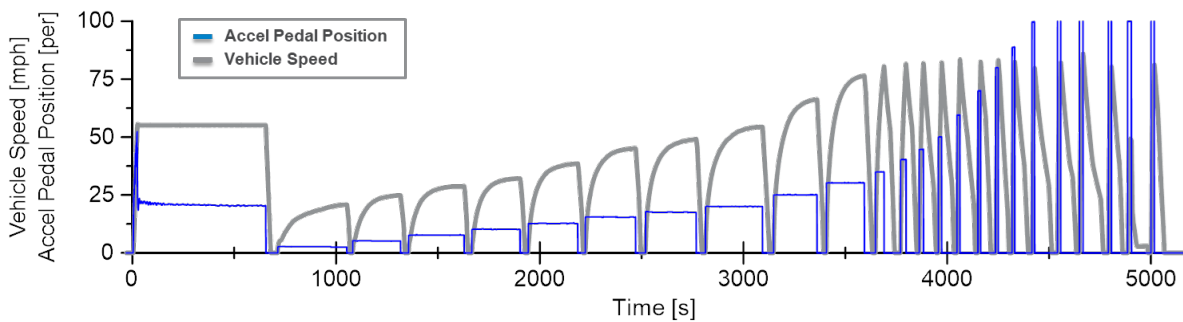


Figure 17: Vehicle acceleration with varying constant pedal inputs

This test provides a map of load demand and gear ratio selection strategy for the full range of powertrain operation. Accelerator pedal inputs were incremented a small amount at low pedal positions to provide higher granularity in torque demand, while increments increased at higher pedal inputs. The desired accelerator pedal position in this test is described in Table 4.

Table 4: Accelerator pedal position selection during constant pedal tip-in test

<b>Event</b>	<b>1</b>	<b>2</b>	<b>3</b>	<b>4</b>	<b>5</b>	<b>6</b>	<b>7</b>	<b>8</b>	<b>9</b>	<b>10</b>
<b>Accel Pedal Position (%)</b>	2.5	5	7.5	10	12.5	15	17.5	20	25	30
<b>Event</b>	<b>11</b>	<b>12</b>	<b>13</b>	<b>14</b>	<b>15</b>	<b>16</b>	<b>17</b>	<b>18</b>	<b>19</b>	
<b>Accel Pedal Position (%)</b>	35	40	45	50	60	70	80	90	100	

Although transmission operation during acceleration is captured during the constant pedal position test discussed above, additional mapping is required to capture transmission operation during deceleration. As a result, another test was conducted with the dynamometer placed in a mode that provides a sequence of constant acceleration and deceleration at a rate of 2 mph/s. This rate was chosen as a compromise between the need for a low enough acceleration rate: low enough to provide adequate quantity of data at each speed/load point but a need for a high enough rate to avoid component overheating at prolonged high loads. During these ramp cycles, the accelerator pedal position is held constant while vehicle speed is varied between 3 mph and 85 mph. An overview of the cycle is shown in Figure 18 below.

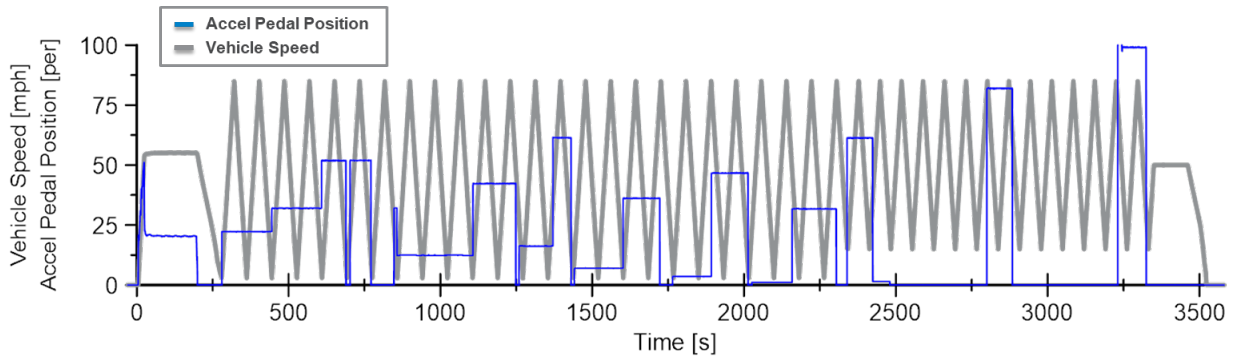


Figure 18: Constant acceleration ramp cycles with varying accelerator pedal inputs

### 3.4.4. Fuel Selection

Test fuel used during chassis dynamometer testing is an important factor affecting the determined fuel economy. Test fuels vary in many ways, such as: energy content, octane value, and other characteristics. The 2018 Honda Accord specifies the use of fuel with an octane rating of 87 (RON 91) or higher. Manufacturer certification testing was performed using a high-octane (RON 93) Tier 2 fuel [6]. Since a low-octane fuel is likely to be often used by consumers, while certification testing was conducted on the high-octane fuel, both high and low octane fuels were evaluated to provide an understanding of the impact of each fuel on vehicle operation.

The low-octane fuel chosen was EPA Tier 3 EEE certification fuel with an octane rating of 88 AKI and 10% ethanol content. The fuel was procured from Haltermann Solutions under the product code HF2021. Table 5 provides the major specifications for the low-octane Tier 3 certification fuel used. The complete fuel specifications sheet for each fuel can be found in Appendix E: Cert Fuel Specifications.

Table 5: Main specifications of the EPA Tier 3 EEE fuel for test for Test ID 61809017-61809052

<b>Fuel Name:</b>	<b>HF2021 EEE Tier 3 (Batch FH3021HW10)</b>
<b>Ethanol content</b>	10%
<b>Carbon weight fraction*</b>	0.827
<b>Density</b>	0.744 (g/ml)
<b>Net heating value</b>	17958 (BTU/lbm)
<b>Research octane number</b>	91.9
<b>Motor octane number</b>	83.3
<b>R+M/2</b>	87.6
<b>Sensitivity</b>	8.6

\*Carbon weight fraction value based on ASTM D5291 results

The low-octane Tier 3 fuel used at the start of testing was depleted prior to the end of low-octane fuel testing. As certification fuel is held to tight tolerances, a second supply of Tier 3 HF2021 fuel was directly used, though it is worth noting it was from a separate batch with slight variations. This fuel change began on test number 61809053 (test numbering is described in the collaborating report [8]), which can be referenced in Appendix D: Test Summary. The fuel specifications are listed in Table 6 below.

Table 6: Main specifications of the EPA Tier 3 EEE fuel for test for Test ID 61809053-61809066

<b>Fuel Name:</b>	<b>HF2021 EEE Tier 3 (Batch GH1621LT10)</b>
<b>Ethanol content</b>	10%
<b>Carbon weight fraction*</b>	0.8252
<b>Density</b>	0.745 (g/ml)
<b>Net heating value</b>	17994 (BTU/lbm)
<b>Research octane number</b>	92.0
<b>Motor octane number</b>	84.3
<b>R+M/2</b>	88.2
<b>Sensitivity</b>	7.7

\*Carbon weight fraction value based on ASTM D5291 results

A high-octane fuel was used to provide comparative data for engine operation as well as comparison with certification testing. The fuel used during certification testing was a high-octane, Tier 2 EEE high-octane certification fuel, and a similar fuel was procured from Haltermann Solutions with the product code HF0437. Table 7 provides the major specifications for the Haltermann Solutions Tier 2 certification fuel used for this test.

Table 7: Main specifications of the EPA Tier 2 EEE fuel for Test ID 61809067-61810004)

<b>Fuel Name:</b>	<b>HF0437 EEE Tier 2 (Batch FC2421BE10)</b>
<b>Ethanol content</b>	0%
<b>Carbon weight fraction</b>	0.8678
<b>Density</b>	0.743 (g/ml)
<b>Net heating value</b>	18627 (BTU/lbm)
<b>Research octane number</b>	96.8
<b>Motor octane number</b>	89.1
<b>R+M/2</b>	93.0
<b>Sensitivity</b>	7.7

\*Carbon weight fraction value based on ASTM D5291 results

The certification fuel used for each test can be referenced in Appendix D: Test Summary.

The high-octane fuel has a 3.7% higher energy content by mass than the low octane fuel, which was accounted for in post-processing for all fuel economy calculations. Vehicle efficiency calculations use the fuel energy content and density, taking into account fuel variability. The specification sheets for each batch of fuel used during testing are listed in Appendix E: Cert Fuel Specifications.

### 3.4.5. Vehicle Setup

Argonne’s testing goal is research fidelity and data capture for the purpose of direct analysis and model development. Due to this, Argonne testing may deviate from certification testing, though standard certification drive cycles are conducted. The staff often purposefully chose to change specific aspects of the test procedures to prioritize vehicle operation in real-world conditions. The standard vehicle and test setup, as well as specifics on these changes, are discussed in the collaborating report [9]. Additionally, for specific details on how a specific test was performed, please consult Appendix D: Test Summary.

All the chassis dynamometer testing was conducted with the vehicle driver assistance systems (aka Honda Sensing) disabled. In addition, a manufacturer-specific tool known as the “Honda Handyman” was provided by the manufacturer to disable systems for operation on the chassis dynamometer. Following several preliminary tests, the data was reviewed to ensure consistent vehicle operation with on road data.

Argonne used the test weight and road load coefficients published by the EPA in 2018 (U.S. Environmental Protection Agency, 2018)[6]. As the vehicle was front wheel drive, it was tested in 2WD mode using only the front rolls of the 4WD test cell. The vehicle was restrained on the chassis dynamometer from lateral motion using chains attached to straps affixed to the front sub-frame of the vehicle. The chains were connected to towers at the front corners of the vehicle. Longitudinal movement of the vehicle was restrained with specialized wheel chocks applied to the rear wheels. The team performed the vehicle coast-down and vehicle loss determination evaluations before formal testing began. Table 8 provides the chassis dynamometer setup parameters for the Honda Accord, where the target coefficient originated from the previously mentioned EPA database [6], while the dynamometer set coefficients were derived from the dynamometer coast down evaluation. Figure 19 shows a picture of the test vehicle mounted to

the chassis dynamometer. Further details on vehicle dynamometer coefficients used for specific tests can be found in Appendix D: Test Summary.

Table 8: Chassis dynamometer target parameters for the 2018 Honda Accord LX test vehicle

<b>Test weight</b>	3,500 (lb)	
<b>Chassis dyno setup</b>	2WD on rolls with dyno mode	
	<b>Target</b>	<b>Set</b>
<b>Road load A term</b>	43.75 (lb)	-1.4 (lb)
<b>Road load B term</b>	-.6042 (lb/mph)	0.3825 (lb/mph)
<b>Road load C term</b>	0.02619(lb/mph <sup>2</sup> )	0.01443 (lb/mph <sup>2</sup> )

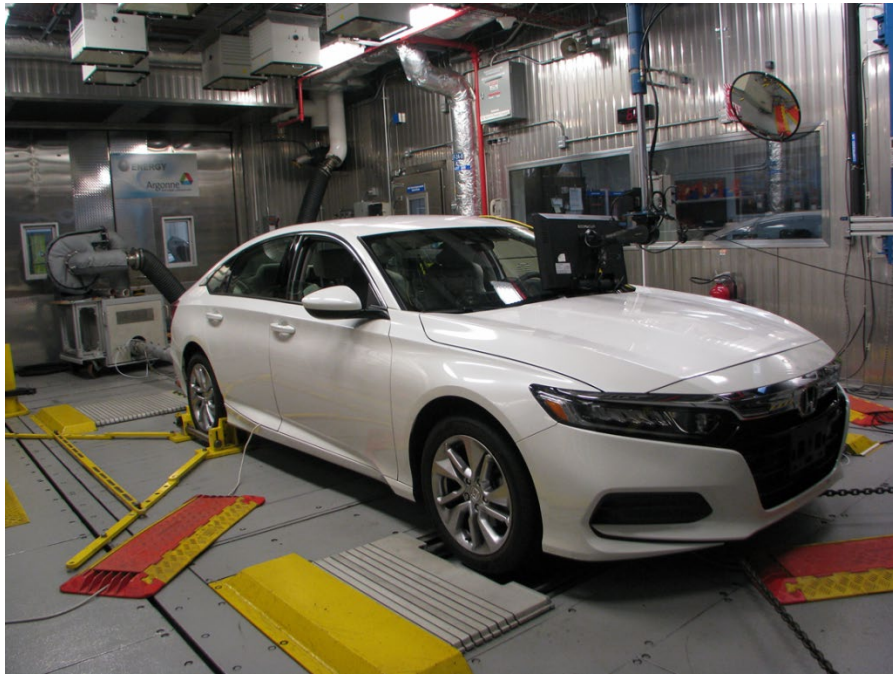


Figure 19: Honda Accord test vehicle mounted to the chassis dynamometer inside the test cell.

### 3.4.6. Driver Selection (Human vs Robotic)

Argonne personnel include experienced dynamometer test drivers with decades of experience operating vehicles on the chassis rolls over test cycles. Vehicle operation on all drive cycles was completed with the use of a human drive. To supplement their efforts in mapping or steady-state speed tests, Argonne uses a robot driver. These unique tests are best performed when step change inputs for braking or acceleration can be executed and subsequently held constant, an operation which is better performed by an actuator. The drivers utilized for specific tests can be found in the test plan in Appendix D: Test Summary.

## 4. Vehicle Testing Analysis

### 4.1. Comparison with EPA CAFE Fuel Economy Results

An initial validation step for vehicle operation can be completed by comparing the fuel economy results from testing with those provided from manufacturer (MFR), or EPA, certification testing [6]. The manufacturer certification testing results, published by the EPA [6], provide unadjusted fuel economy results for phases 1, 2, and 3 of the UDDS, otherwise known as the FTP, as well as the HWFET cycle. Figure 20 and Table 8 compare the manufacturer and EPA published fuel economy results to the results from AMTL testing. AMTL tests were separated into two categories; the single high-octane test, and the average of the three low-octane test sequences performed on the low-octane fuel.

The fuel economy results from the EPA published, and resulting tests are within 2% on all certification cycles. Several factors which influence this variability, which are noted in the testing reference [9], which include AMTL testing performed with the vehicle hood closed and the test cell fan in vehicle speed match mode. Some additional factors which influence this variability are how closely a driver follows a prescribed cycle, the specific test vehicle state (tire wear, etc.), and ambient conditions beyond test control (absolute pressure). In addition to providing the specific test data for future comparison ([www.anl.gov/d3](http://www.anl.gov/d3)), and calculating SAE J2951 [10] driver metrics for the testing, repeat testing of the certification cycles was performed.

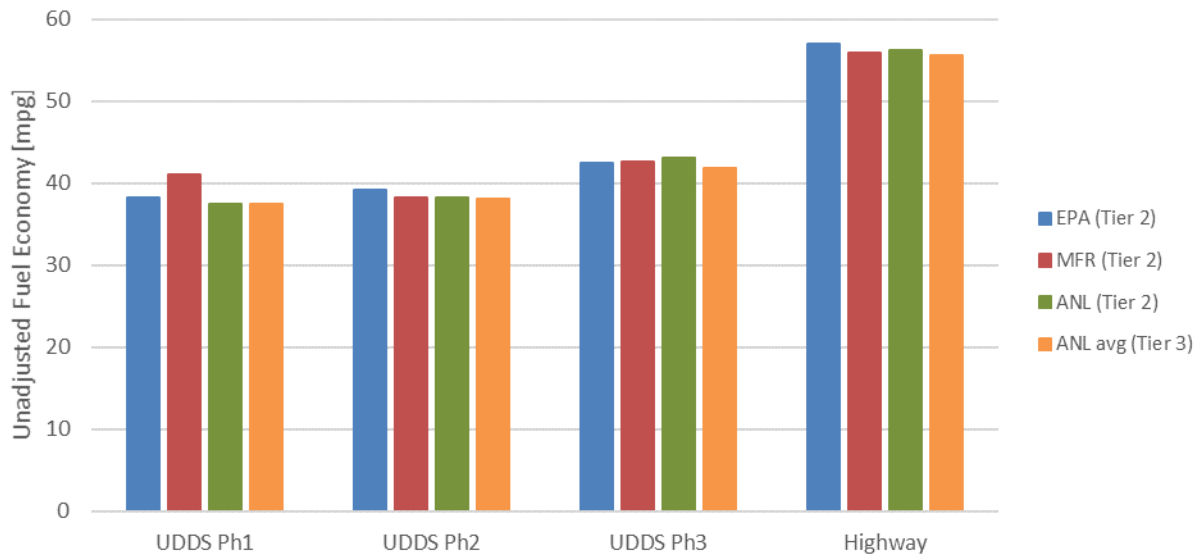


Figure 20: Raw fuel economy results: UDDS and HWFET certification cycles from EPA and Argonne

### 4.2. Test to Test Repeatability

Three separate repeats of the certification cycles were performed on the low-octane Tier 3 fuel at the ambient test temperature of 23 °C. The test results on the low-octane fuel demonstrate acceptable levels of repeatability. Figure 21 and Table 9 compare the three test sequences completed at the AMTL. From the test process described in Figure 15, the results differ by less than 1.3% from the average value for UDDS testing and less than 1% for HWFET testing.



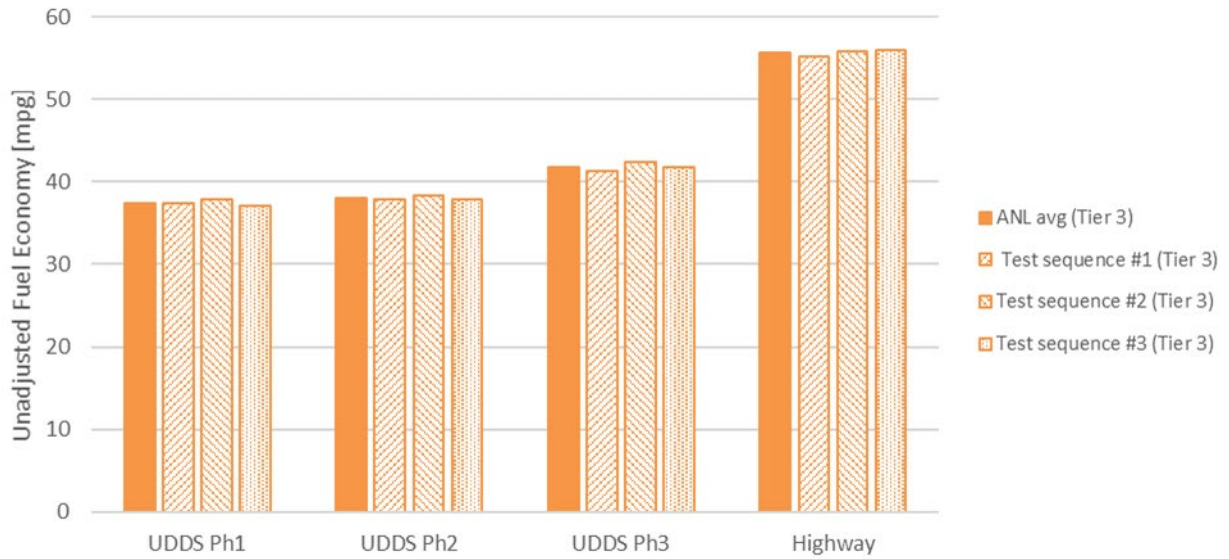


Figure 21: Test to test repeatability (UDDS AND HWFET raw fuel economy results)

Table 9: Raw fuel economy results: UDDS and HWFET certification cycles from EPA and Argonne

	EPA (Tier 2)	MFR (Tier 2)	ANL (Tier 2)	ANL avg (Tier 3)
<b>UDDS Ph1</b>	38.1	41.0	37.5	37.4
<b>UDDS Ph2</b>	39.1	38.2	38.2	38.0
<b>UDDS Ph3</b>	42.5	42.6	43.0	41.8
<b>HWFET</b>	57.0	55.9	56.1	55.6

### 4.3. U.S. Standard Drive Cycles

#### 4.3.1. Vehicle Fuel Economy

The fuel economy results for standard drive cycles are presented in Table 10. The drive cycles include the cold start UDDS (Phase 1 and 2), the hot start UDDS (Phase 3 and 4), a third UDDS cycle, the HWFET cycle and the US06 cycle. The third UDDS cycle is not part of the certification testing; it is performed to further understanding of the fuel economy changes at higher powertrain temperature. Both the HWFET and US06 drive cycles included 2 phases of testing: a prep cycle and the test cycle. The fuel economies presented in Table 9 for HWFET and US06 are determined from the second test, as was described previously in Figure 19.

Table 10: Raw Tier 3–88 AKI average fuel economy results for drive cycle results

	Fuel economy (mpg)
<b>UDDS #1 Cold Start</b>	37.7
<b>UDDS#1 Ph1</b>	37.4
<b>UDDS#1 Ph2</b>	38.0
<b>UDDS#2 Hot</b>	39.8
<b>UDDS#2 Ph3</b>	41.8
<b>UDDS#2 Ph4</b>	38.1
<b>UDDS#3</b>	39.3
<b>UDDS#3 Ph1</b>	41.3
<b>UDDS#3 Ph2</b>	37.7
<b>HWFET</b>	55.6
<b>US06</b>	32.3
<b>US06 City</b>	21.2
<b>US06 Highway</b>	37.9

Figure 22 shows an example of general vehicle operation on a section of the urban dynamometer driving schedule (UDDS) cycle. The Honda Accord idles its internal combustion engine when the vehicle is stopped. When the vehicle accelerates, the CVT reduces the transmission ratio to maintain a low engine speed. At a speed of 35 mph and a low accelerator pedal position, the transmission enables engine speeds as low as 1,200 rpm. During deceleration the engine is not fueled, seen where the equivalence ratio reaches a maximum lean value. During deceleration, the engine is instead motored through the transmission using vehicle kinetic energy. Fueling is resumed and the engine transitions from a fuel cutoff mode to idle mode prior to the vehicle arriving at a full stop.

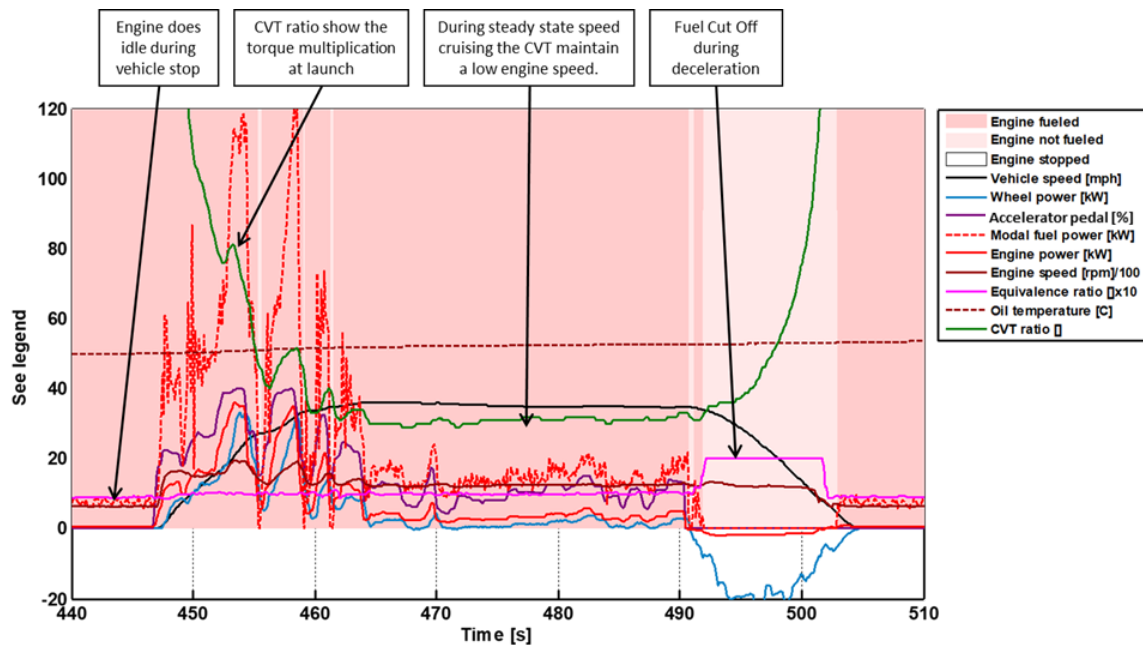


Figure 22: Honda Accord powertrain operation on cold start UDDS

### 4.3.2. Vehicle Efficiency

Vehicle efficiency, calculated as per the SAE standard for drive cycle metrics, SAE J2951[10], is calculated by dividing positive driven cycle energy (CEd) by the fuel energy used over the drive cycle. Table 11 provides the calculated vehicle efficiencies for the drive cycles in each test sequence.

Table 11: Powertrain efficiencies based on J2951 positive cycle energy

	Test Sequence #1	Test Sequence #2	Test Sequence #3	Average
<b>UDDS #1 Cold Start</b>	23.8%	23.7%	24.2%	23.9%
<b>UDDS#2 Hot Start</b>	25.2%	25.1%	25.5%	25.2%
<b>UDDS#3</b>	24.7%	24.7%	25.3%	24.9%
<b>HWFET</b>	32.0%	31.8%	32.2%	32.0%
<b>US06</b>	31.0%	30.8%	31.2%	31.0%

The lowest average vehicle efficiency occurs on the UDDS cycle, which is typical for conventional vehicles. The UDDS cycle is a stop-and-go drive cycle with very mild power requirements. On the UDDS cycle, the engine operates at low load with a relatively low throttle opening, which increases the pumping losses. The powertrain efficiency increases by 1% from the cold start cycle to the third cycle, where the powertrain has reached its operating temperature. This efficiency increase is due to a combination of factors, largely driven by a reduction in friction typical of increasing temperatures in all components of the powertrain. The increase in efficiency due to increasing powertrain temperature is partially offset on the third UDDS cycle, as powertrain temperature reaches a point that requires additional cooling which results in increased loading of the alternator.

Average powertrain efficiency is highest on the HWFET drive cycle. On this cycle, the powertrain can take full advantage of the CVT, increasing engine loads and reducing the engine speed of the small displacement boosted engine to enable the vehicle to achieve over 30% vehicle efficiency. The average powertrain efficiency on the US06 drive cycle is also over 30%, mainly driven by increased engine loading.

### 4.3.3. Thermal Impact on Fuel Economy and Vehicle Efficiency

The UDDS cycles, the highway cycle, and the US06 cycle were also tested at -7 °C and at 35 °C with 850 W/m<sup>2</sup> of solar load, the two extreme temperature conditions established for the EPA five-cycle fuel economy label [6]. Figure 23 provides the test results for those conditions and drive cycles.

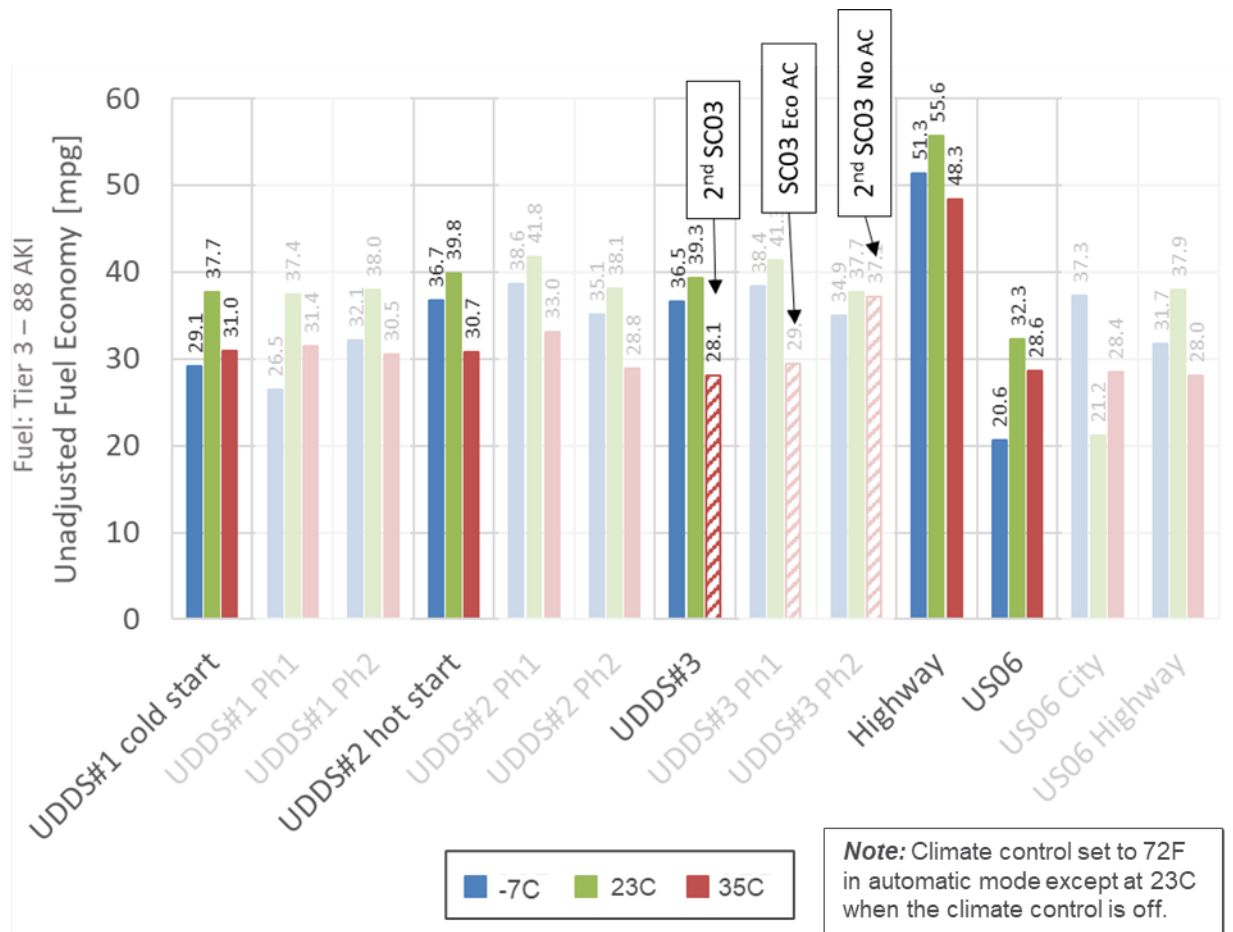


Figure 23: Raw, uncorrected, fuel economy results for certification cycles across different temperature conditions

The fuel economy for the cold start UDDS at -7 °C is 23% lower when compared to the same test at 23 °C. This impact decreases to 8% for the second UDDS cycle. The powertrain has to overcome significantly increased drive train friction losses during the cold start at -7 °C cycle, but once the powertrain reaches a steady operating temperature, those friction losses become less significant.

Vehicle fuel economy at the elevated 35 °C with solar load emulation (SCO3 testing conditions as described in the CFR [7]) is also less than at the 23 °C test condition. The fuel economy decreases by 18% and 24% for the cold start UDDS and the hot start UDDS cycles, respectively, in comparison to the 23 °C test condition. The fuel economy reduction is driven by the additional power required to operate the air conditioning system. In addition, the ability to perform deceleration fuel cutoff (DFCO) is significantly reduced (16.5% of cycle time in DFCO for UDDS cold start at 23 °C compared to 3.1% of cycle time at 35 °C) as the additional compressor load requires the engine to restart fueling sooner. Note that for the 35 °C testing, the third UDDS cycle was replaced by an SCO3 (the air conditioning "supplemental FTP") drive cycle. This change was made to capture vehicle operation on the SCO3 cycle, while the first two UDDS cycles remained to provide a comparison for other ambient temperatures.

Table 12 provides the calculated vehicle efficiencies for the different ambient test conditions. The impact of the cold powertrain temperatures is apparent in the -7 °C cold start efficiency. As

the powertrain temperatures rise throughout the tests in the test sequence, the vehicle efficiencies at  $-7\text{ }^{\circ}\text{C}$  start to approach the vehicle efficiencies at  $23\text{ }^{\circ}\text{C}$  ambient temperature. The impact of the auxiliary load from the air conditioning compressor at  $35\text{ }^{\circ}\text{C}$  is also apparent in this table. It is noteworthy that the efficiency impact of the air conditioning compressor is lower on the high-power US06 drive cycle, because the ratio between the air conditioning power to the average wheel power is lower than the same ratio for the UDDS cycle.

Table 12: Powertrain efficiencies across different ambient test conditions based on Tier 3 fuel

	$-7\text{ }^{\circ}\text{C}$	$23\text{ }^{\circ}\text{C}$	$35\text{ }^{\circ}\text{C}$
<b>UDDS #1 Cold Start</b>	18.5%	23.9%	18.0%
<b>UDDS#2 Hot Start</b>	23.4%	25.2%	19.2%
<b>UDDS#3</b>	23.4%	24.9%	N/A
<b>HWFET</b>	29.8%	32.0%	28.0%
<b>US06</b>	30.5%	31.0%	27.5%

Figure 24 shows the engine operational areas for the cold start and hot start UDDS at each of the three ambient temperature conditions. The  $23\text{ }^{\circ}\text{C}$  plot in the middle serves as the reference. At  $-7\text{ }^{\circ}\text{C}$ , the engine operation is slightly shifted to higher speeds, but not higher torque. At  $35\text{ }^{\circ}\text{C}$ , the engine torque is shifted upwards throughout the map due to the additional power required to run the air conditioning compressor.

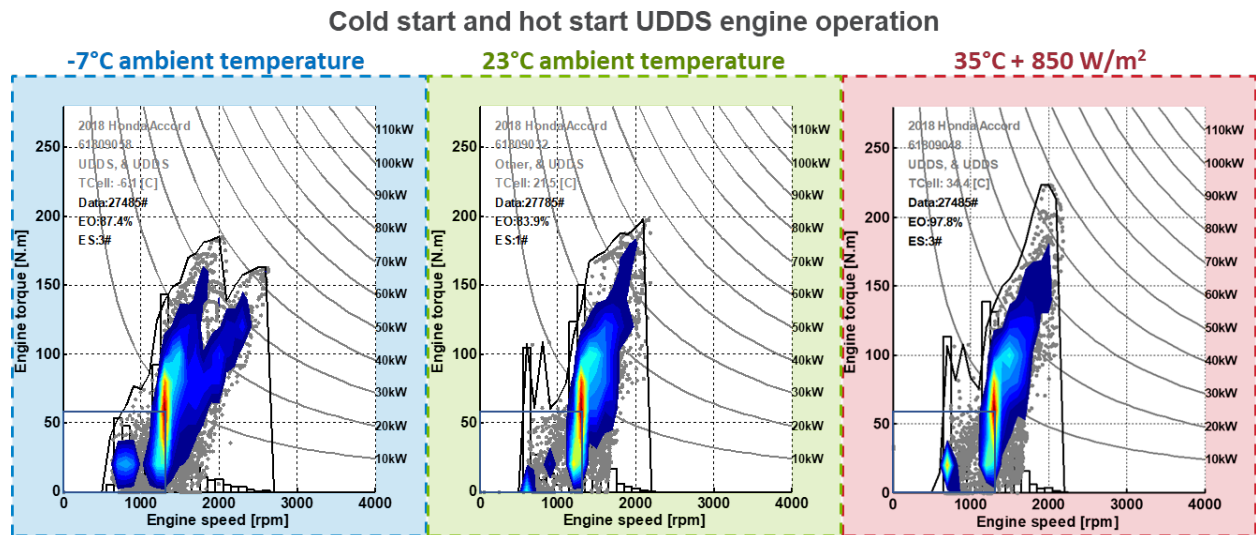


Figure 24: Engine operation on the UDDS across different temperatures

Figure 25 shows some relevant powertrain and ambient temperature profiles over the completion of the test sequence. To obtain a thermally stable result, three consecutive HWFET drive cycles were performed at  $-7\text{ }^{\circ}\text{C}$ . Additionally, at the test temperature of  $35\text{ }^{\circ}\text{C}$ , the SC03 test cycle replaced the third UDDS cycle to capture operation on the air conditioning test cycle. These graphs also show the targeted  $23\text{ }^{\circ}\text{C}$  cabin temperature that the automatic climate control system tries to achieve in the  $-7\text{ }^{\circ}\text{C}$  and  $35\text{ }^{\circ}\text{C}$  test condition.

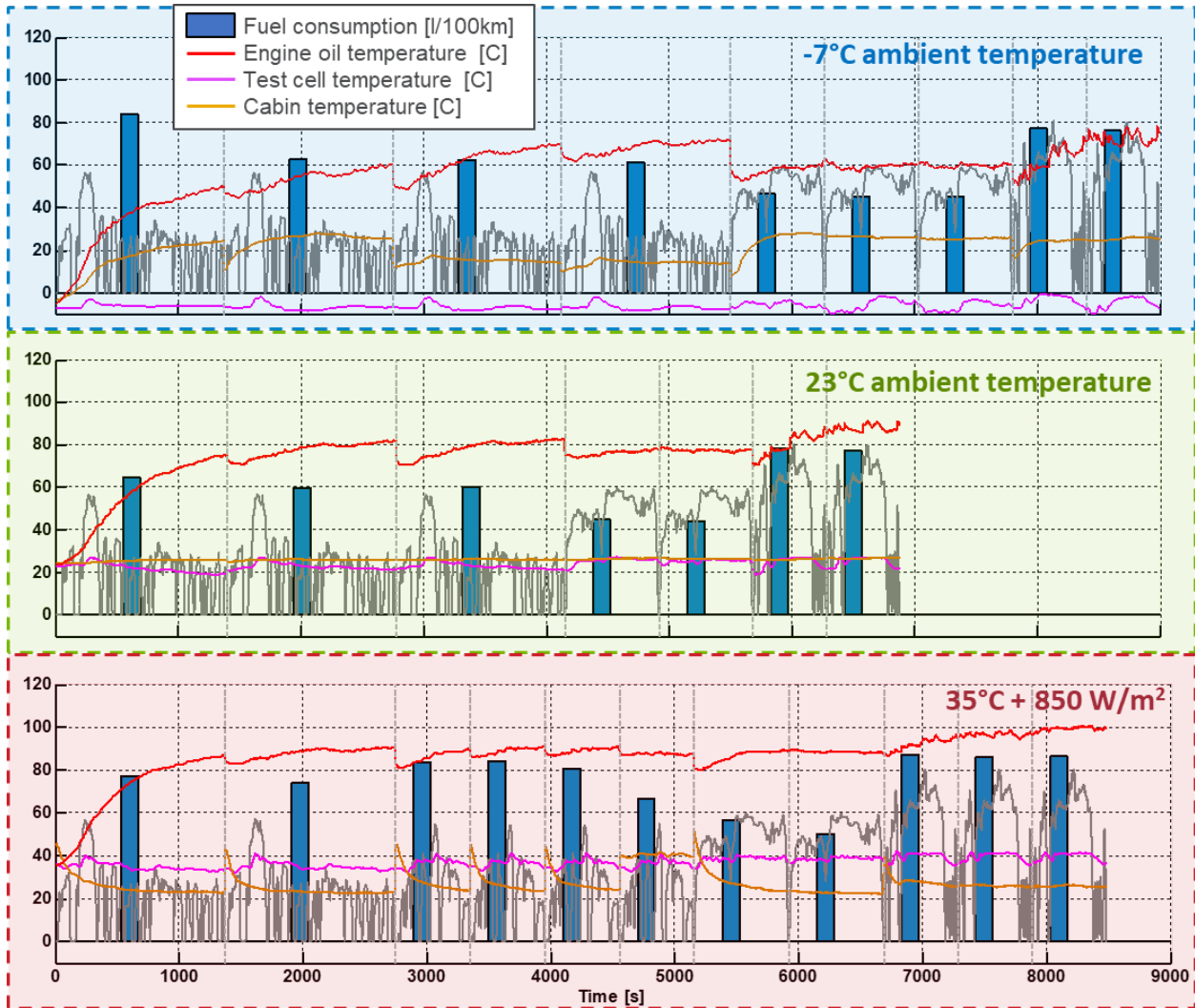


Figure 25: Powertrain and cabin temperature profiles across different temperature

The engine oil temperature is representative of the powertrain temperature. For the 23 °C and 35 °C ambient temperature conditions, the final engine oil temperature for the US06 is 95 °C to 100 °C. The engine temperature during the -7 °C test conditions required a longer duration to reach a steady state level, and once at a stable temperature remains 20 °C lower, increasing pumping losses and decreasing efficiency.

#### 4.4. Steady State Speed

One characterization test run is the steady state speed drive cycle, which holds vehicle speed for one minute at speeds from 10 mph to 80 mph in increments of 10 mph. The vehicle is accelerated then decelerated through the set speed points in order to capture any effects that may be seen in powertrain operation. This test was conducted at several varying vehicle states and loads, including:

- Low-octane fuel at 0% grade in Drive mode at 23 °C
- High-octane fuel at 0% grade in Drive mode at 23 °C
- Low-octane fuel at 0% grade in Drive mode at 35 °C with 850 W/m<sup>2</sup> solar emulation
- Low-octane fuel at 0% grade in Drive mode at -7 °C



- Low-octane fuel at 0% grade in Sport mode at 23 °C
- Low-octane fuel at 0%, 3%, and 6% grade in Drive at 23 °C

Vehicle fuel economy results, along with vehicle efficiency, the power required at the wheel, engine speed, and the transmission ratio, were calculated. Results for the 0% grade test on low-octane fuel, are presented in Figure 26.

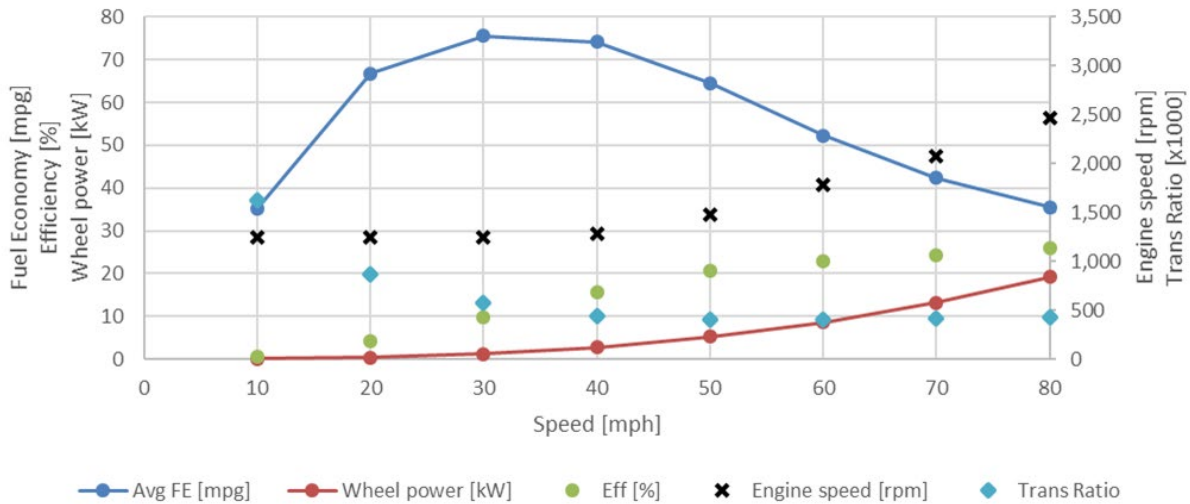


Figure 26: Steady state speed operation at 23 °C, 0% grade and Tier 3 low-octane fuel

The highest fuel economy occurred at a vehicle speed of 30 mph. Vehicle efficiency increases as vehicle speed increases due to increased powertrain loading. In Figure 24, this can be seen by a progression of engine speed and engine load into a higher efficiency band at the engine speeds of 1,100-1,400 rpm and up to 100 Nm. Vehicle efficiency continuously increases to a maximum of 26% at the maximum test speed of 80 mph. By constantly varying the CVT ratio at speeds of 10–30 mph, the engine speed is held at 1,250 rpm. A slight increase in engine speed, to 1,300 rpm, is seen at 40 mph. At speeds over 50 mph, the CVT ratio is held constant at 0.41:1, allowing the engine speed to increase with increasing vehicle speed. The maximum engine speed, at 80 mph, recorded at 2,500 rpm. Though efficiency does increase with vehicle speed, vehicle losses increase as well due to aerodynamic drag (emulated on dyno) and rotational losses. These additional losses offset improvements to vehicle efficiency, ultimately resulting in a lower fuel economy.

As discussed in greater detail in section 4.8, steady-state speed operation testing was also performed using a high-octane Tier 2 fuel. Testing on both fuels was conducted with the dynamometer in 2WD mode, with the vehicle remaining mounted to the dynamometer during the fuel swap. Figure 27 shows vehicle operation with high-octane fuel. Fuel economy, vehicle efficiency, and general vehicle operation all mirrored the behavior observed with low-octane fuel, showing typical test-to-test variability. This behavior was expected because vehicle and engine loading at 0% grade is low, and any variations to ignition timing due to low-octane fuel are likely reduced.

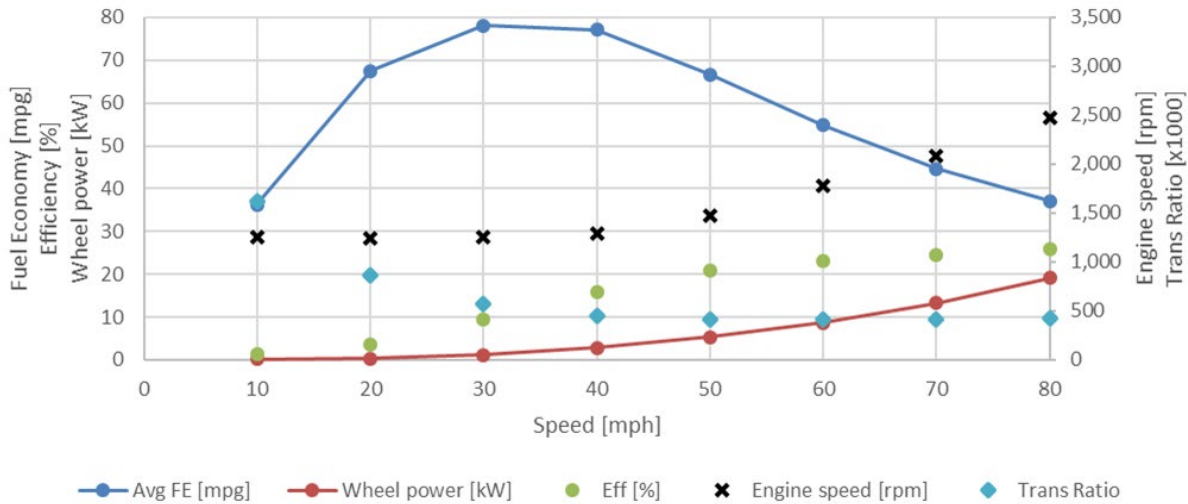


Figure 27: Steady state speed operation at 23 °C, 0% grade and Tier 2 high-octane fuel

Additional steady state testing was completed to capture comparison data at operating conditions, including:

- An elevated temperature of 35 °C with solar emulation
- A low temperature of -7 °C
- Sport mode at 23 °C
- Grades of 0%, 3%, and 6% at 23 °C

The results of the 0% grade and elevated temperature condition are shown in Figure 28.

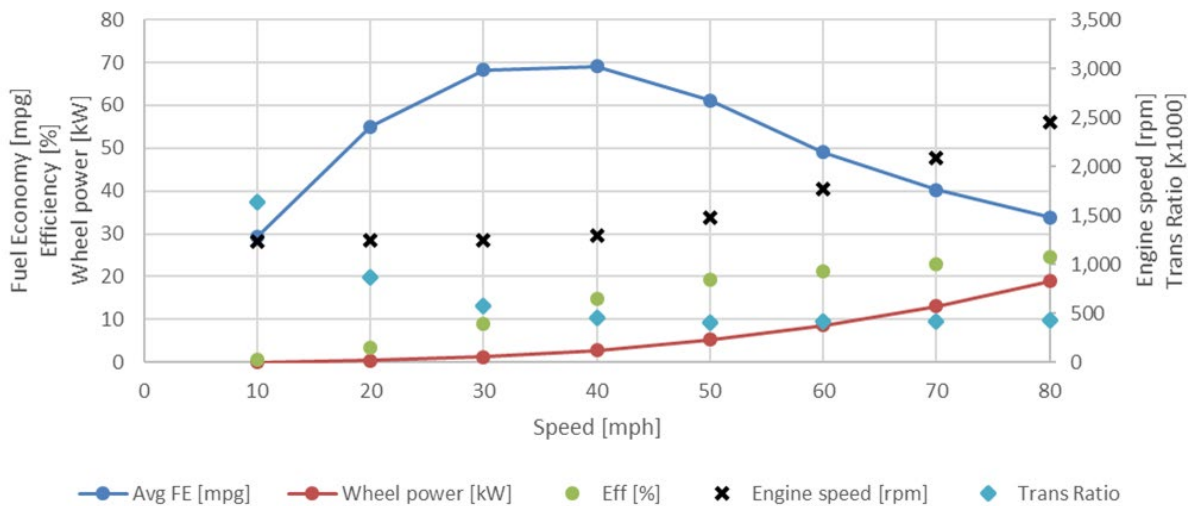


Figure 28: Steady state speed operation at 35 °C and 0% grade

Transmission operation at the elevated test temperature was similar to transmission operation during the 23 °C baseline test. Dynamometer target coefficients were held constant regardless of temperature, resulting in an equal wheel power requirement. It should be noted that though this results in equal wheel power requirements, this will result in a variation from real world operation as changes in air density were not adjusted which affect vehicle loading. Though



rotating losses are reduced due to higher component temperatures, AC compressor loads result in a reduced peak fuel economy of approximately 68 mpg. This peak fuel economy remained consistent at the 30 to 40 mph speeds.

Vehicle efficiency is reduced by HVAC operation. This effect is greater at lower speeds due to the relative power requirement of HVAC compared to other loads. It should be noted the vehicle cabin reached steady state before start of test to avoid variable impacts of the “pulldown” of cabin temperature on the results.

An additional steady state speed test was completed at -7 °C to capture the impacts of cold temperatures; the results can be seen in Figure 29. As in the elevated temperature test, dynamometer target coefficients were held constant regardless of temperature to provide an equal wheel power requirement across speeds. Fuel economy and vehicle efficiency were reduced across the vehicle speed range during the low temperature test. This decrease was due to lower temperature components and fluids causing increased rotational losses, with the greatest impact demonstrated at low speeds.

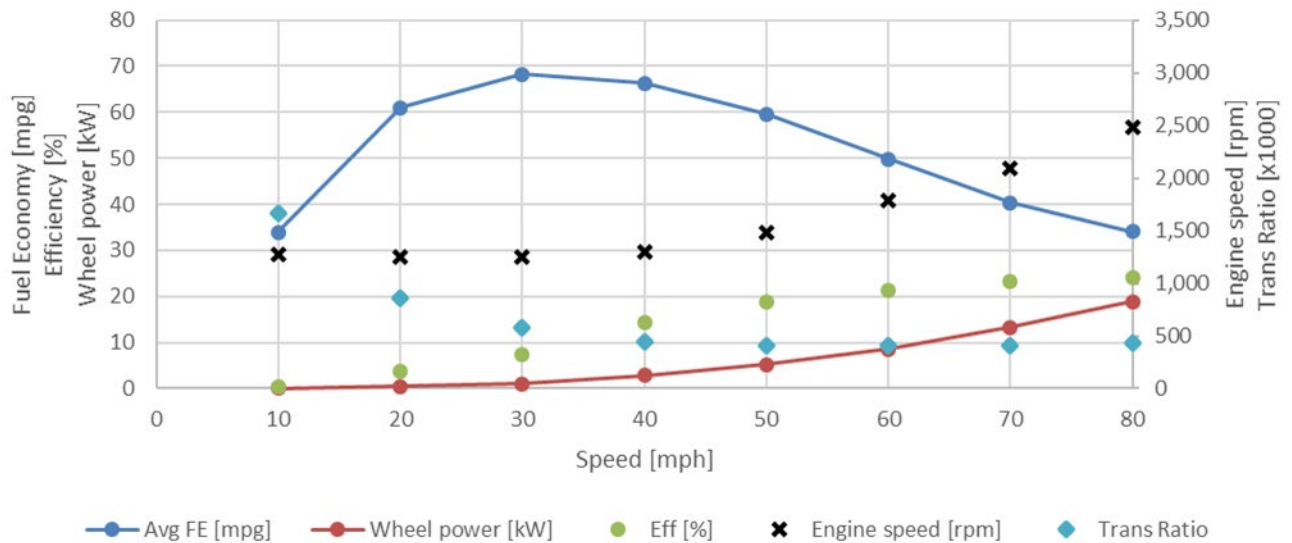


Figure 29: Steady state speed operation at -7 °C and 0% grade

Operating the transmission in Sport mode resulted in a significant change in powertrain performance. This change in operation can be seen in Figure 30. For the test speed, the CVT ratio was increased, resulting in a higher engine speed. Higher powertrain speeds resulted in higher powertrain losses, reducing fuel economy and vehicle efficiency across the speed range. The fuel economy at speeds of 30–40 mph, the speeds that yielded the highest results, was approximately 35% lower than in Drive.

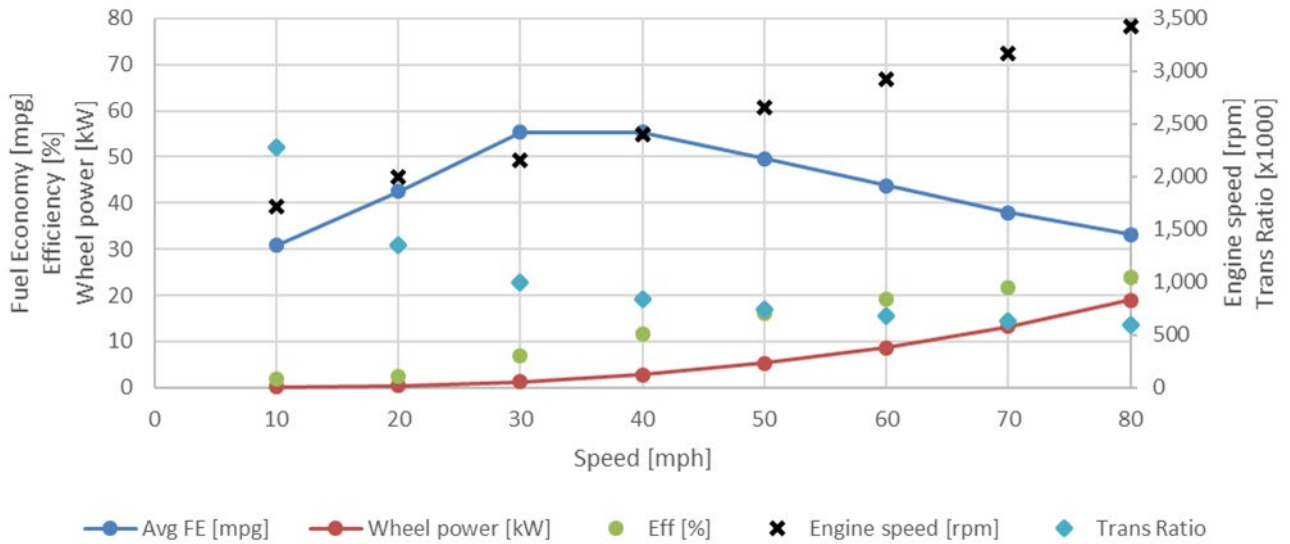


Figure 30: Steady state speed operation at 23 °C and 0% grade in Sport mode

Figure 31 and Figure 32 display vehicle operation while the vehicle is in Drive on a constant grade of 3% and 6% respectively. A change in vehicle loading can be seen as an increase in wheel power as the grade increases from baseline 0% grade to 3% and 6% grade tests. At 3% grade, the increase in required wheel power has no impact on engine speed or transmission ratio selection until a speed of 40 mph is reached. At over 40 mph, engine speed is increased, and the transmission ratio remains higher to deliver greater wheel power. The additional loading results in a considerable reduction in fuel economy, though efficiency is increased due to the change in powertrain load point. Operation at 6% grade further demonstrates these trends, with decreases in fuel economy but increased efficiency.

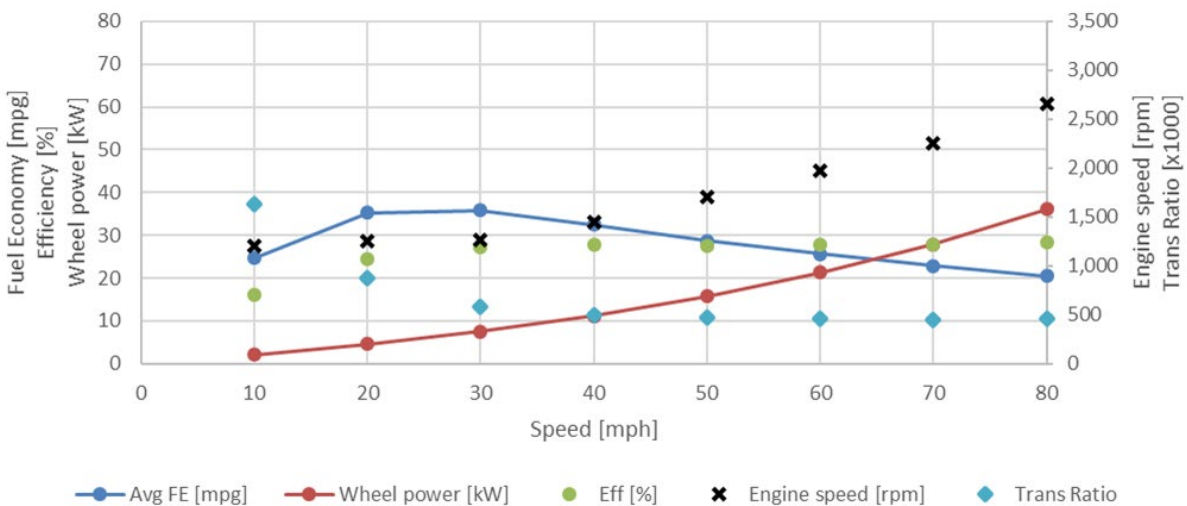


Figure 31: Steady state speed operation at 23 °C and 3% grade

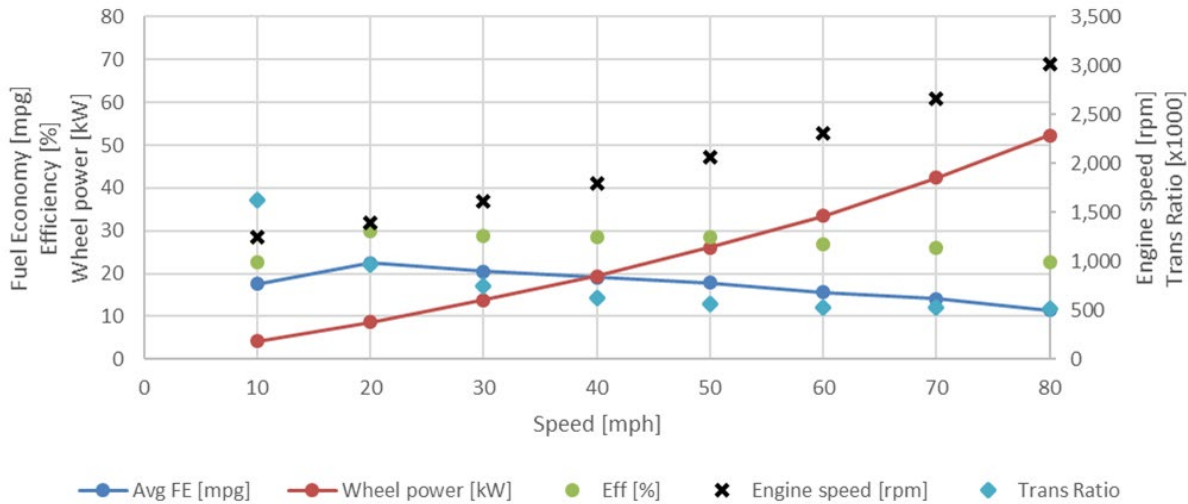


Figure 32: Steady state speed operation at 23 °C and 6% grade

#### 4.5. Passing Maneuver

Specific tests were performed to characterize a vehicle overtaking another on a highway. This passing maneuver drive cycle includes accelerations from 35 to 55 mph, 55 to 65 mph, 35 to 75 mph and 55 to 80 mph. Additionally, to determine vehicle operation at higher loads, such as on an incline, this test is repeated at dynamometer grade settings of 0%, 3%, and 6%. For each passing maneuver, the vehicle is held at an initial steady-state speed, then the driver applies 100% accelerator pedal until the vehicle passes the desired end speed.

Table 13 summarizes the time to complete each passing maneuver on both high- and low-octane fuels and for a passing maneuver performed in Sport mode.

Table 13: Time duration for acceleration events in seconds

Passing Maneuver Times				
	mph	0% grade	3% grade	6% grade
<b>Low-Octane Drive Mode</b>	35-55	4.1	4.5	4.9
	55-65	3.1	3.5	3.9
	35-70	7.2	8.2	9.8
	55-80	6.7	8.2	10.3
<b>Low-Octane Sport Mode</b>	35-55	3.9		
	55-65	2.8		
	35-70	6.8		
	55-80	6.4		
<b>High-Octane Drive Mode</b>	35-55	4.1	4.4	4.7
	55-65	3.0	3.3	3.6
	35-70	7.1	7.8	8.8
	55-80	6.7	7.6	8.8

A plot of powertrain details for the passing maneuver from 55 mph to 80 mph, with low-octane fuel and the vehicle in Drive, is shown in Figure 33.

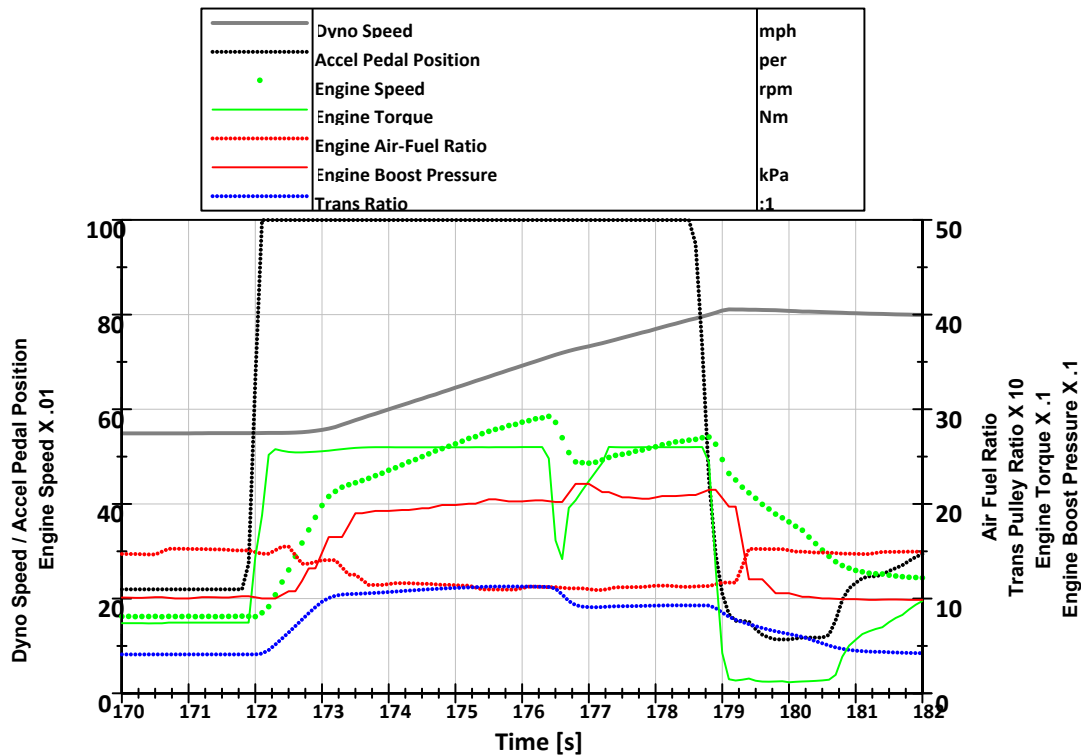


Figure 33: Powertrain operation during the 55 mph to 80 mph passing maneuver with low-octane fuel in Drive at 0% grade

In this case, the powertrain required slightly more than 0.4 seconds after application of the accelerator pedal to reach a peak torque of 250 Nm. The CVT begins to transition from an initial ratio of just below 0.5 immediately after pedal application and reaches a maximum rate of ratio change slightly after 1 second. This maximum rate of change continues until a speed of 70 mph, where a ratio change occurs, reducing the ratio to around 0.92. Fuel enrichment begins around 0.5 seconds after the 100% pedal movement and blends to a ratio of 10:1 to 11:1 once the turbocharger boost pressure rate has stabilized. The acceleration on the low-octane fuel in drive takes about 6.7 seconds.

A separate test was conducted with the vehicle in Sport mode. The same section of this test, the 55 mph to 80 mph acceleration at 0% grade, is displayed in Figure 34.

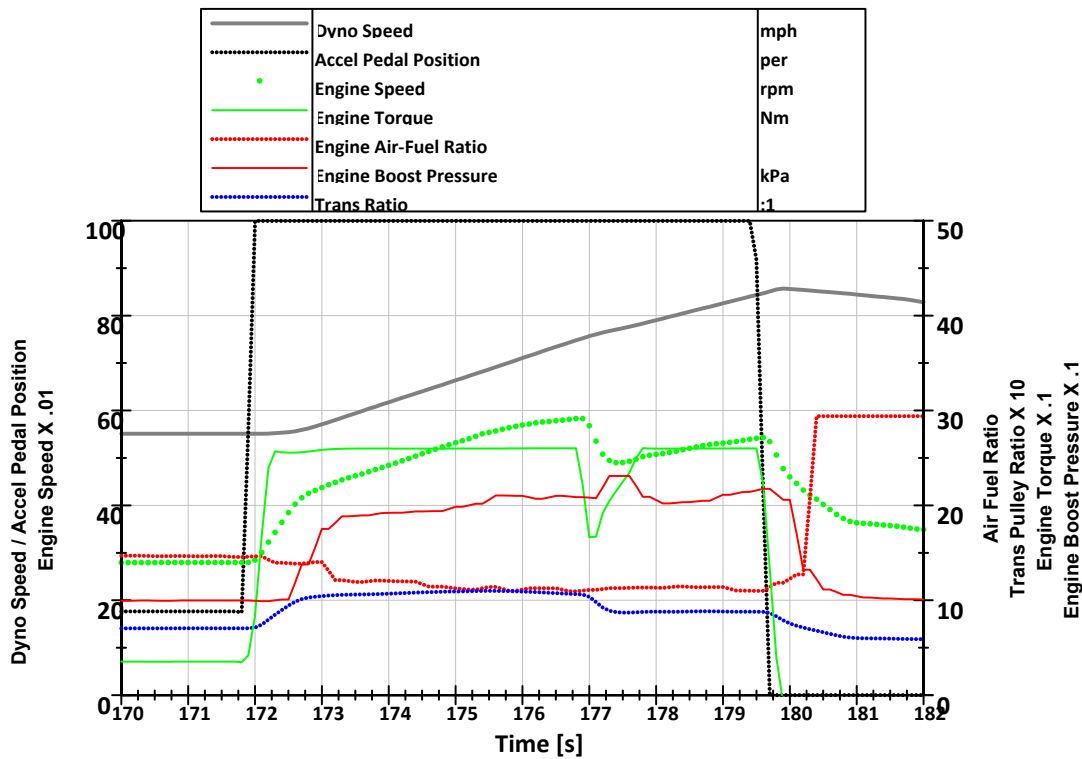


Figure 34: Powertrain operation during the 55 mph to 80 mph passing maneuver with low-octane fuel in Sport at 0% grade

The vehicle steady state operation at 55 mph changes while in Sport mode. The plot shows that the engine speed before the passing maneuver is held at 2,800 rpm, as compared to an engine speed in Drive mode of 1,625 rpm. As the engine was at a higher speed due to the higher transmission ratio, the delivered engine torque at 55 mph was reduced from 75 Nm to 35 Nm while the engine power remained constant at just above 10 kW. Following the driver's movement to 100% pedal, engine power and intake manifold pressure increased at a comparably faster rate than in Drive. This reduced the time for the passing maneuver by 0.3 s.

A third passing maneuver test was performed after the vehicle fuel swap from low-octane fuel to high-octane fuel. The same section of this test, the 55 mph to 80 mph acceleration at 0% grade, is shown in Figure 35.

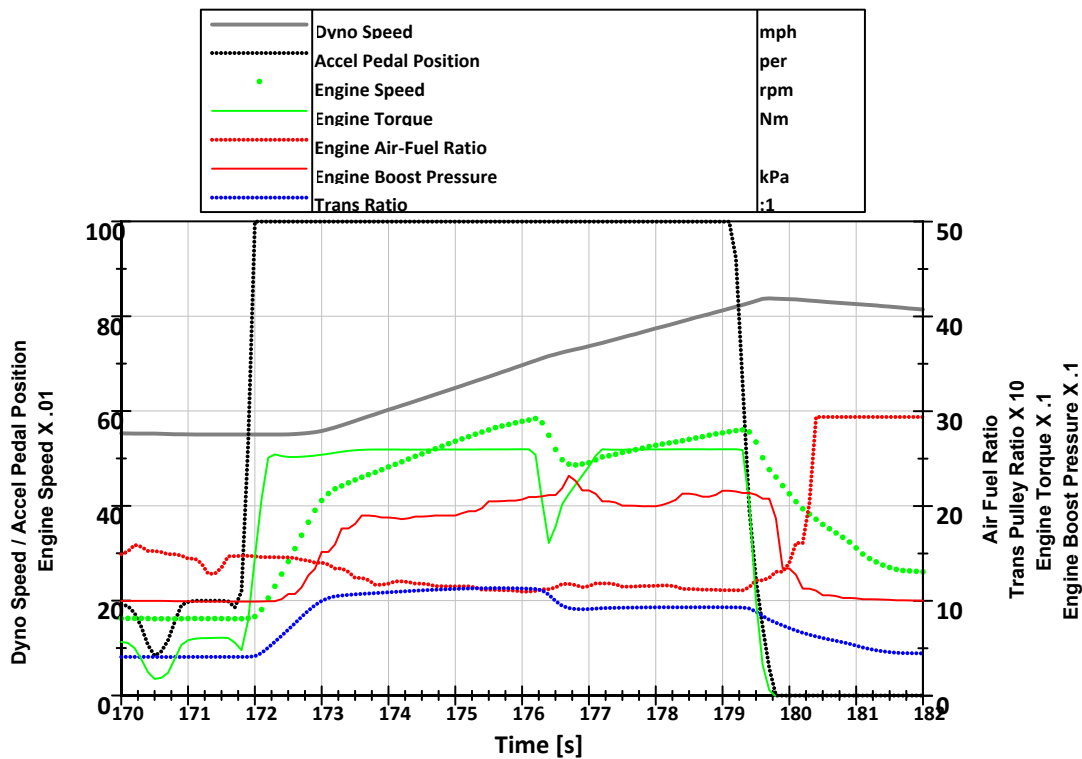


Figure 35: Powertrain operation during the 55 mph to 80 mph passing maneuver with high-octane fuel in Drive at 0% grade

At 0% grade, no significant difference in the time needed for passing maneuvers is seen between the two fuels. At a higher vehicle load, as seen on the 3% and 6% grade passing maneuver, a notable increase of 1.5 s is found. A 4-degree difference in the knock retard on the 6% grade 55 mph to 80 mph acceleration when comparing the two fuels, with the high-octane fuel holding at 6 degrees. As a result, engine boost pressure and throttle position are increased with the high-octane fuel, allowing for a reduced transmission pulley ratio. The combination of these factors results in a higher dynamometer power and a reduced acceleration time.

#### 4.6. Maximum Acceleration

Maximum acceleration performance tests were performed on the chassis dynamometer. The test is performed from a rolling start to alleviate traction issues of the tire on a steel roll. During this testing two powertrain behaviors were observed.

In the first type of powertrain behavior, the CVT simulates fixed gear ratios at speeds above 40 mph while the accelerator is at 100%, as seen in Figure 36. The engine accelerates up to 6,300 rpm and produces 156 kW (209 hp). From vehicle background research [1], this emulated shifting is the standard behavior a driver could expect in full acceleration on the road.

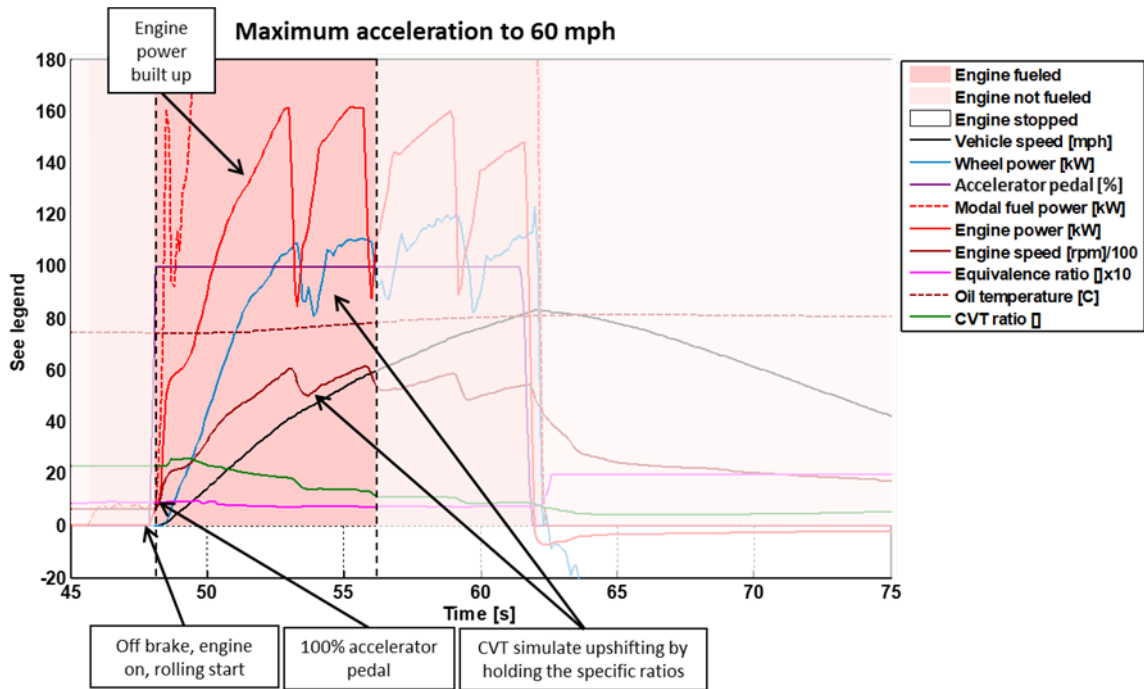


Figure 36: Powertrain operation during maximum acceleration, with a focus area highlighted

Note that in the maximum acceleration runs, the engine power of 165 kW (221 hp) is higher than the SAE rated horsepower of 143 kW (192 hp) during this short acceleration event. Argonne also tested the vehicle on a simulated 25% grade test, during which the continuous power settled at 146 kW (196 hp) with 6,300 rpm and 220 Nm, as shown in Figure 37.

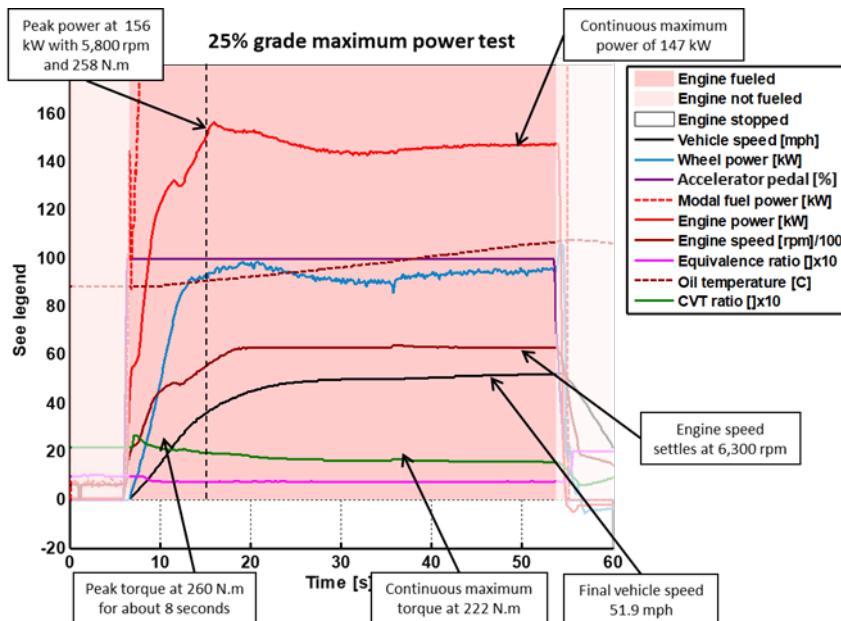


Figure 37: Honda Accord continuous power test on simulated 25% grade, with a focus area highlighted



The second powertrain behavior observed in full acceleration testing, limited the engine power while the CVT smoothly and continuously adjusts the transmission ratio, as seen in Figure 38. In this state, though no malfunctioning indicator lamp (MIL) was displayed, the engine is maintained at a roughly constant speed of around 4,000 rpm and at power levels of 110-130 kW. This limit continued over five consecutive acceleration cycles and is notable. Though the specific cause of this powertrain operation was not found, nor later repeated, it demonstrates vehicle operation in a state in which vehicle powertrain performance is reduced.

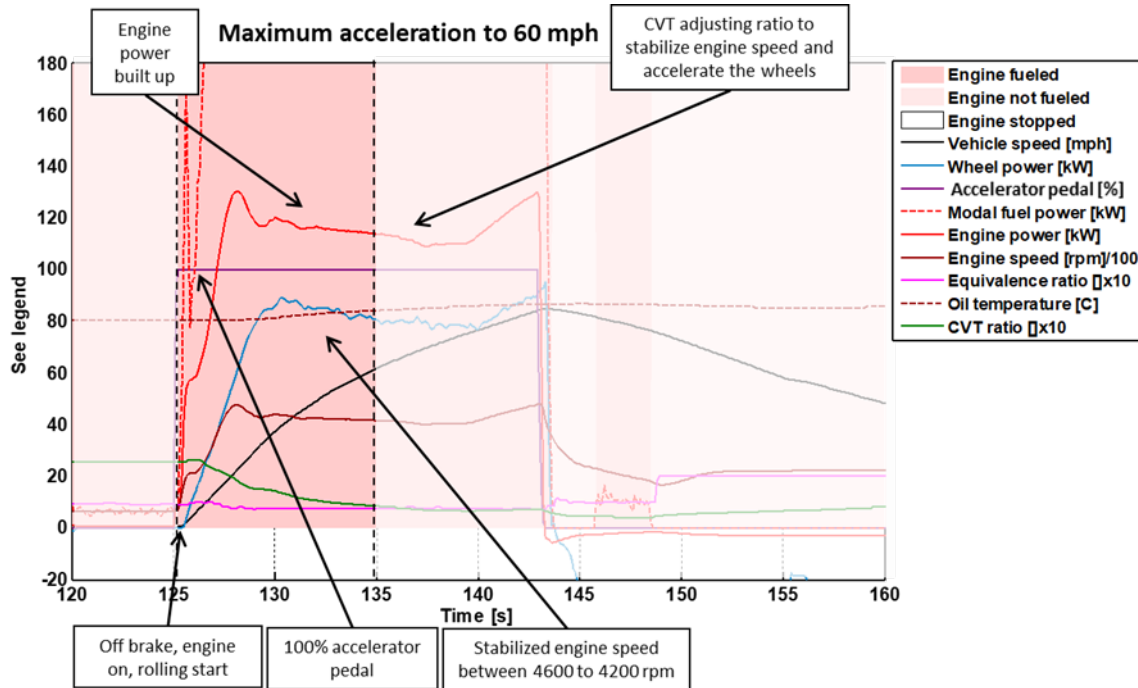


Figure 38: Powertrain operation during maximum acceleration with simulated constant gear ratios, with a focus area highlighted

#### 4.7. Idle Fuel Flow Rate Test Results

A 30-minute engine idle test in cold start conditions was performed with the transmission in Park, following a 12-hour soak at the test temperature of 23 °C. This test is designed to characterize engine behavior and fuel flow rate as the powertrain warms up at idle.

Figure 39 shows the first 120 seconds of the cold start engine idle test. The engine is started at 23 seconds into the test. On starting, the engine speed increases to over 1,750 rpm, and gradually settles to 1,100 rpm before transitioning to closed-loop operation about 60 seconds later. During this period, the ignition is retarded to help warm up the exhaust after-treatment system. During the transition to closed loop operation, ignition timing advances and stabilizes, and gradually engine speed and engine fuel power (calculated from fuel flow rates and fuel properties) decrease. At the transition, the catalyst temperature from diagnostics (vehicle estimated) is reported at about 200 °C.



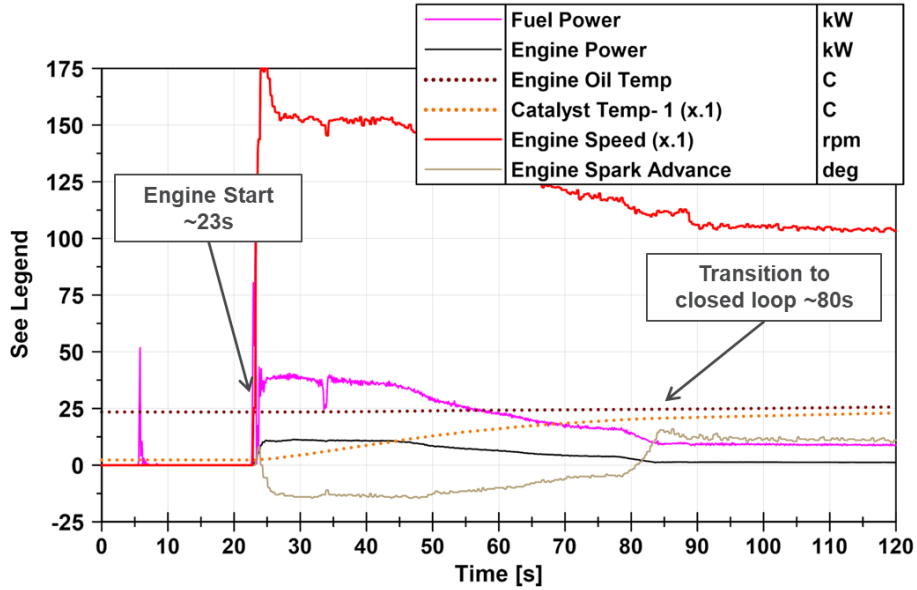


Figure 39: Initial 120 s of the idle fuel flow test

The full 30 minutes of the idle fuel flow test can be seen in Figure 40. It should be noted that the fuel spike, seen at ~5 sec is a result of the fuel system pressurizing, causing a temporary flow through the fuel scale though not used by engine operation. The engine oil temperature continues to increase over the duration of the test, ending at slightly over 73 °C. The catalyst temperature increases over the duration of the test as well, reaching a near steady state temperature at the end of the test of just under 410 °C.

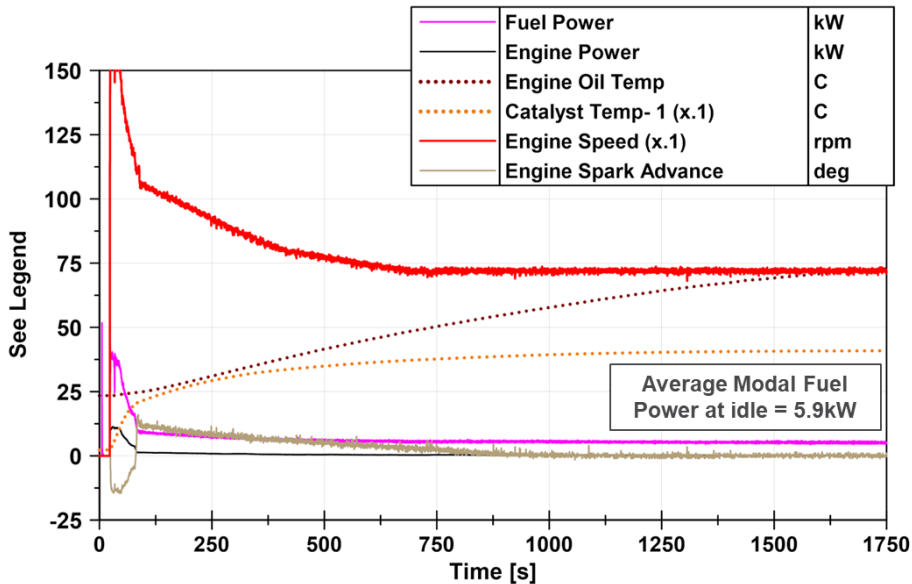


Figure 40: Idle fuel flow test—full duration

#### 4.8. Tier 3–88 AKI (Low-Octane) to Tier 2–93 AKI (High-Octane) Fuel Comparison

Fuels with octane ratings of 87 (RON 91) and higher are recommended for the 2018 Honda Accord, and due to lower price, the lower octane fuel is expected to be the dominant fuel used by consumers. Argonne tested the vehicle on low and high octane certification fuel to investigate the effects of octane rating on fuel economy and performance. The Tier 2 certification fuel has an octane rating of 93 AKI and the Tier 3 certification fuel has an octane rating of 88 AKI. The Tier 2 fuel represents premium fuel, and the Tier 3 fuel represents regular fuel in this investigation.

The specifications for the fuels are in Table 4, Table 5, and Table 6, with full fuel specification sheets in Appendix E: Cert Fuel Specifications, and further details on these fuels can be found in section 3.4.4. Though both fuels are standard test fuels, several differences should be noted, including octane, energy content, and ethanol content. The Tier 3 – 88 AKI has a volumetric energy content that is 3.7% lower than the Tier 2 – 93 AKI’s.

The unadjusted fuel economy results comparing the bench reported fuel consumption of the two fuels on certification cycles are shown in Table 14. The impacts of the lower-octane fuel on bench reported fuel economy are seen across all drive cycles, though at a higher amount on cycles with higher powertrain temperatures and powertrain loading.

*Table 14: Octane impact on fuel economy (MPG) on standard drive cycles at 23 °C*

	<b>Tier 3 – 88 AKI avg. fuel economy (mpg)</b>	<b>Tier 2 – 93 AKI fuel economy (mpg)</b>	<b>Difference of low and high octane fuels on mpg</b>
<b>UDDS#1 Cold Start</b>	37.7	37.9	<b>-0.3%</b>
<b>UDDS#2 Hot Start</b>	39.8	40.6	<b>-2.0%</b>
<b>UDDS#3 Hot Start</b>	39.3	40.9	<b>-4.0%</b>
<b>HWFET</b>	55.6	56.1	<b>-0.9%</b>
<b>US06</b>	32.3	34.4	<b>-6.5%</b>

Vehicle efficiency based on SAE J2951™[10] calculations are shown in Table 15. The vehicle efficiencies for the Tier 2 – 93 AKI fuel are lower than for the Tier 3 – 88 AKI fuel. It is not possible to determine the reasons (octane, energy content, other fuel specifications) for the shift without further testing, which was outside the scope of this effort.

Table 15: Octane impact on vehicle efficiency

Vehicle Efficiency	Tier 3 – 88 AKI	Tier 2 – 93 AKI
UDDS#1 Cold Start	23.9%	23.2%
UDDS#2 Hot Start	25.2%	24.8%
UDDS#3	24.9%	25.0%
HWFET	32.0%	31.3%
US06	31.0%	31.9%

The value of knock feedback correction on a pair of UDDS cycles, with the first being a cold start, is shown in Figure 41 for the series of tests for both fuels. The correction value on the high-octane Tier 2 – 93 AKI fuel is consistently higher than the low-octane Tier 3 – 88 AKI fuel.

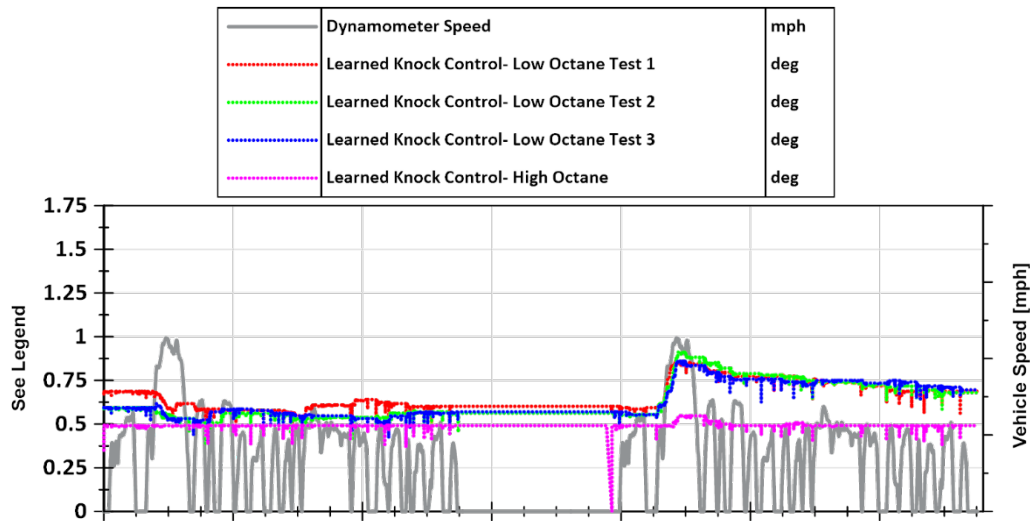


Figure 41: Knock feedback signals on UDDSx2, cold start cycles

Figure 42 shows the ignition timing for both fuels for the UDDS, HWFET, and US06 cycles as well as the passing maneuver test and the maximum acceleration test. The maximum vehicle reported engine torque was found to be equal for both fuels. With the lower octane fuel, the spark ignition timing is retarded by a few degrees at these higher loads to prevent engine knock from occurring.

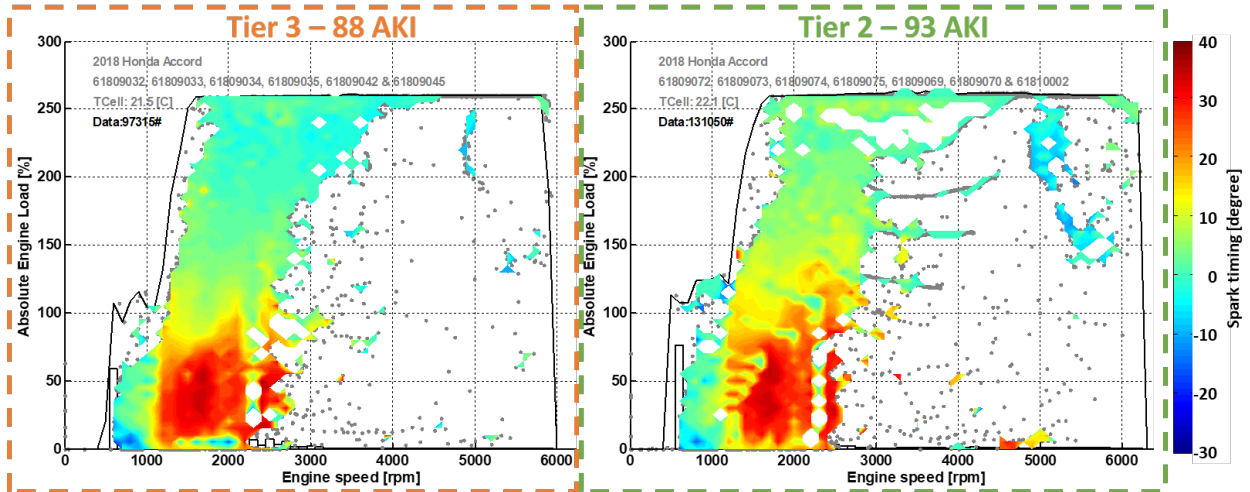


Figure 42: Spark advance comparison between Tier 2 and Tier 3 fuels

Further evaluation was completed on vehicle acceleration from a standstill. The vehicle performed better using the 93 AKI fuel compared to the 88 AKI fuel. The vehicle accelerated to 80 mph 1 second faster under maximum acceleration with the 93 AKI fuel. The performance tests suggest that the engine torque is increased with the high-octane fuel, due to spark advance. Additionally, the difference between the two fuels was outside of test-to-test uncertainty, most notably at 6.5% on the higher load US06 cycle. At the lower load cycles of the UDDS and HWFET the impact to fuel economy was reduced.

## 5. Component and Control Analysis

This section describes the vehicle component controls, including transmission shifting, torque converter lockup, engine fuel cutoff, and detailed component control concepts. Models and control calibrations developed through this analysis have been implemented in Autonomie.

### 5.1. Signal Calculations for Control Analysis

The vehicle component control analysis is conducted using Autonomie “Import Test Data” process. This process automatically changes signal names and test data units to match Autonomie nomenclature based on pre-defined conversion methods. During the test data import process, additional parameters required to analyze the component operating conditions are calculated from the test data. The vehicle configuration and signals sources are shown in Figure 43.

In Figure 43 the signals labeled in black, blue, and green are obtained directly from the test. At the energy management strategy level, the signals used to calculate the engine and battery power are critical, and directly obtained from the test. While not all signals can be recorded during testing, some can be easily calculated from the measured ones. For example, the output torque and speed of the transmission are calculated by the dyno force and speed. Transmission input signals are calculated by engine torque and speed, using assumptions of the torque converter efficiency map used in FRM (final rulemaking) study [7]. Techniques used in the process are described in the following section.

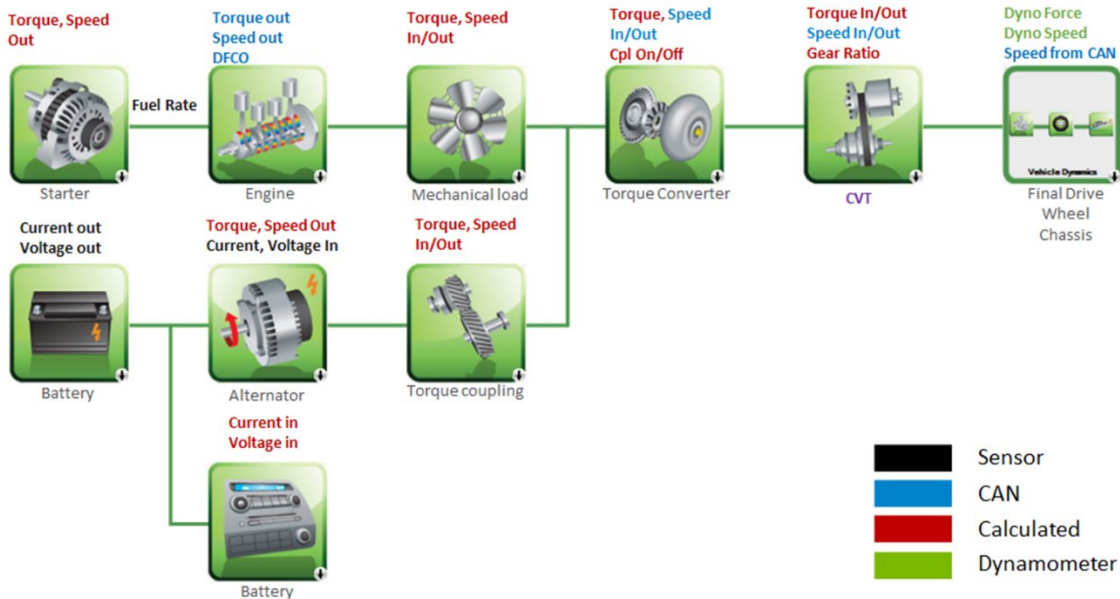


Figure 43: Schematic of the vehicle configuration

The signals marked in red in Figure 43 are calculated based on measured signals and additional information obtained from vehicle details [1], and EPA test information [6]. The details of this calculations are explained in this section. First, the time-based rotating speed and torque of each component is calculated as shown in Figure 44.

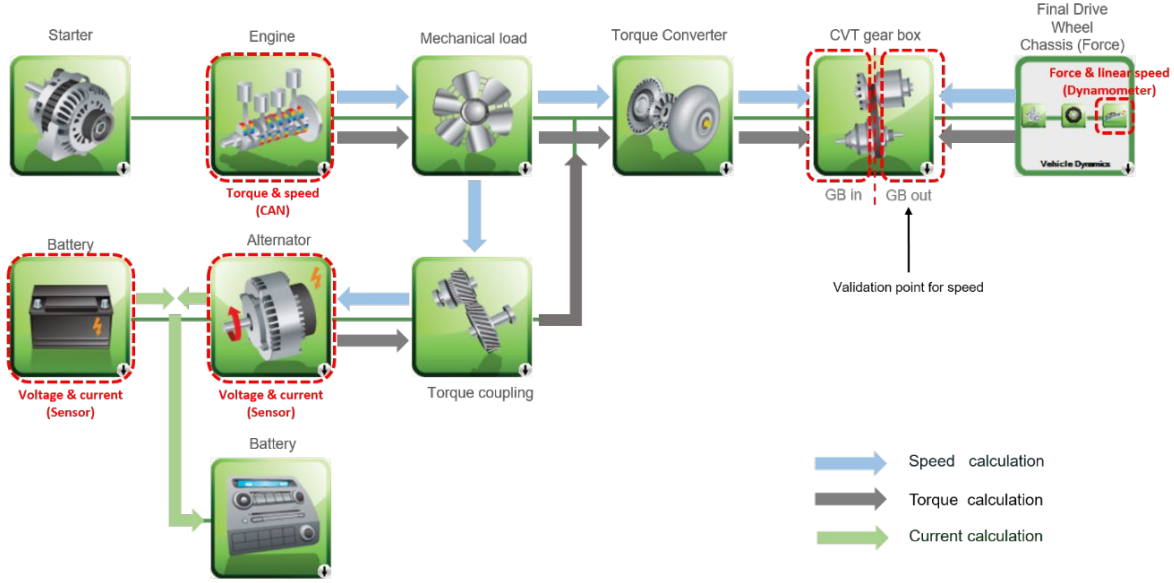


Figure 44: Calculation of missing signals for component

The wheel speed can be calculated from the chassis speed signal obtained from the dynamometer:

$$\omega_{gb,out} = \gamma_{fd} \frac{1}{r_t} v_{chassis}$$

Equation 1

where  $r_t$  is the tire radius and  $\gamma_{fd}$  is the final drive ratio. Because the exact tire radius in driving conditions is not known, the speeds can be validated by comparing the two values of  $\omega_{GB,out}$  and  $v_{chassis}$  by adjusting the tire radius. While there may be no discrepancy in speed for the wheel and chassis, the torque calculations should be carefully handled because each component torque measurements include uncertainties.

Figure 44 shows the flow of the calculation for torque signals. Because an accurate transmission efficiency map is not available, the torque calculation process is divided into two parts: from the transmission output to the wheel and from the engine to the transmission input. The output torque of the final drive is calculated from the force obtained from the dynamometer:

$$T_{fd,out} = T_{wheel,out} + T_{wheel,loss} - T_{wheel,brake} = r_t \cdot F_{chassis} + T_{wheel,loss} - T_{wheel,brake}$$

Equation 2

The gearbox output torque is calculated from  $T_{fd,out}$ , which can be expressed as:

$$T_{gb,out} = T_{fd,in} = \frac{1}{\eta_{fd}^k} \cdot \frac{1}{\gamma_{fd}} \cdot T_{fd,out}$$

Equation 3

where  $\eta_{fd}$  is the final drive transfer coefficient, and  $k$  is 1 if the power flows from the final drive to the wheel or -1 if the power flows in the other direction. These values are generic and will be applied to following calculations in this report:

$$k = \begin{cases} 1 & \text{if power flows from power sources to the wheel} \\ -1 & \text{if power flows from the wheel to power sources} \end{cases}$$

Equation 4

The torque converter torque input is calculated from the mechanical accessory load torque and the torque-coupling torque:

$$T_{acc\_mech} = T_{eng} - P_{acc\_mech}/\omega_{eng}$$

Equation 5

$$T_{TC,in} = T_{acc\_mech} + T_{trq\_cpl,out}$$

Equation 6

where  $P_{acc\_mech}$  is the mechanical accessory power the system needs.

The transmission input torque is calculated from the torque converter input torque and the torque converter characteristics. The speed ratio can be calculated from CAN signals for transmission input and output speed:

$$T_{gb,in} = T_{TC,out} = T_{TC,in} \cdot T_{ratio}(= f(\omega_{ratio}))$$

Equation 7

where  $T_{ratio}$  is the torque ratio of the torque converter, and  $\omega_{ratio}$  is the speed ratio of turbine speed to impeller speed for the torque converter.

All the equations for torque calculation are based on static equilibrium. The parameter values used in the calculations are listed in Table 16.

Table 16: Parameter values used for calculating additional signals [1]

Parameters	Values
Tire radius, $r_t$	0.317m
Gear ratio range of CVT	0.405 ~ 2.645
Gear ratio of the final drive, $\gamma_{fd}$	5.36
Vehicle test weight	1588kg

In addition to the signals introduced in this section, other signals representing efforts and flows are calculated based on component assumptions [7].

## 5.2. Transmission Operation

The 2018 Honda Accord has a continuously variable transmission (CVT). The control algorithm in Autonomie used to select the CVT gear ratio relies on multiple parameters that need to be calibrated for each individual vehicle. The transmission operation was analyzed to estimate those control parameters used in Autonomie. The details of such analysis are explained in the subsequent sections.

### 5.2.1. Gear Ratio Control

To understand the choice of reduction ratio in CVT, we divided the infinite gear range into 55 sub-ratio segments. Figure 45 shows the comparison of time spent in each gear ratio segment for each cycle.

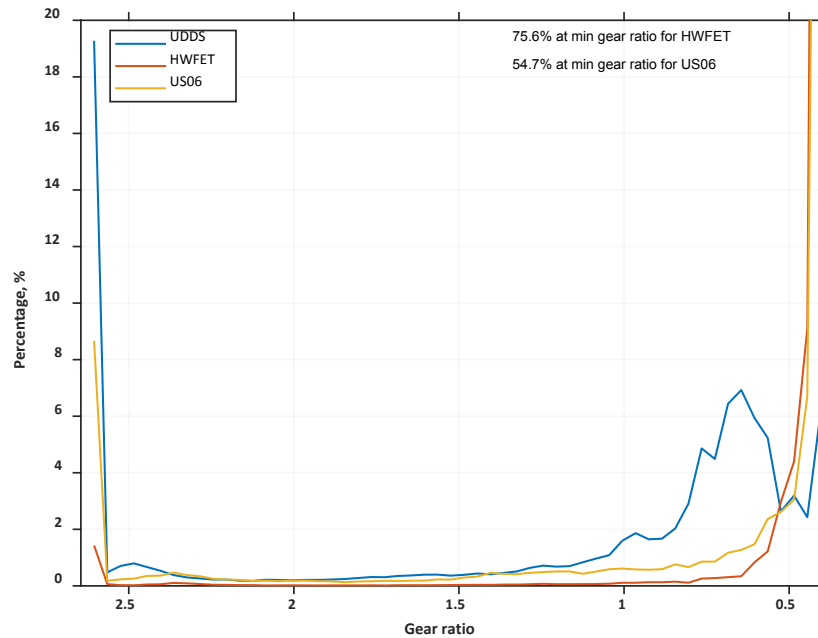


Figure 45: Time spent in each gear ratio segment for UDDS/HWFET/US06 cycles

In the urban driving cycle, the highest gear ratio (lowest gear range) is used about 19% of the time, versus just below 9% of the time in highway cycle. The low gear ratios are used more frequently, especially in the high-speed driving cycle, in which the minimum gear ratio (highest gear range) is used more than 50% of the time.

Understanding the choice of ratio at various vehicle speeds and acceleration scenario is essential in developing an accurate CVT control.

### 5.2.2. Torque Converter Lockup Control

In order to see the overall behavior of the torque converter lockup status, all operating points of the vehicle from testing are shown in Figure 46 and Figure 47. The graphs show the torque converter is locked above a certain speed regardless of wheel torque. In Figure 46, the clutch is locked at approximately 9mph or greater vehicle speed. However, in the high torque region at lower vehicle speeds, it can be seen that the torque converter is unlocked to use the torque multiplication effect.



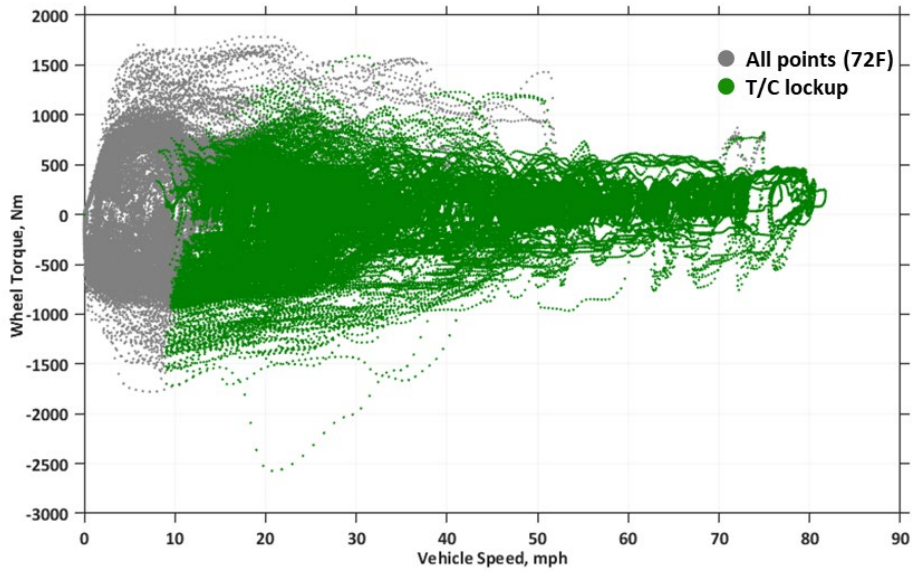


Figure 46: Torque converter lockup operation – wheel torque vs vehicle speed

In Figure 47, the torque converter lockup appears to occur when the CVT gear ratio drops below ~1.6.

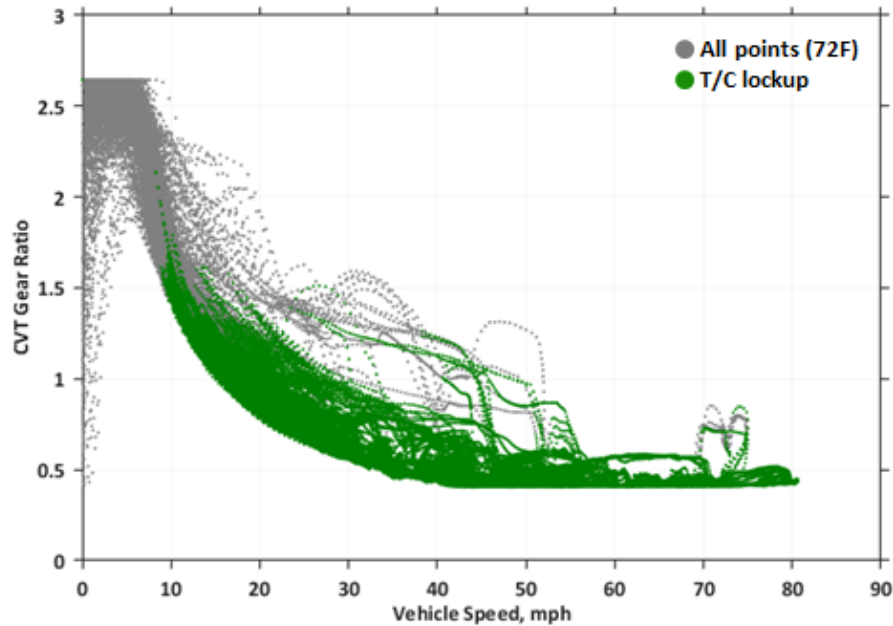


Figure 47: Torque converter lockup operation — vehicle speed vs CVT gear ratio

The percentage of torque converter lockup per test cycle is summarized in Table 17. When the vehicle is driven in the urban cycle the torque converter is locked approximately 67% of the time. During the highway cycle the torque converter can be seen locked over 97% of the time. Locking the torque converter helps improve the overall drivetrain efficiency. Hence the algorithm to govern the locking of torque converter is very important in improving fuel economy.

Table 17: Percentage time of torque converter locked per each test cycle

Test Cycle	UDDS	HWFET	US06	WLTP	JC08	LA92
%	67.01	97.95	80.49	69.87	62.12	50.54

### 5.2.3. Lockup Variability

To analyze how torque converter lockup is controlled for CVT operation, the points at which the clutch is engaged and the points at which the clutch is released were analyzed.

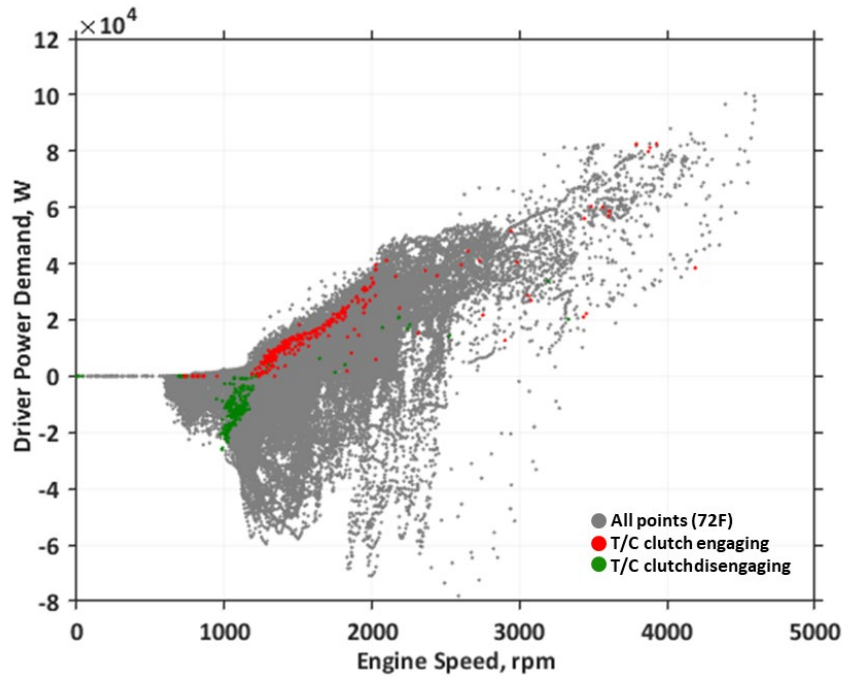


Figure 48: Torque converter operation points for clutch engaging vs. disengaging

In Figure 48 the points at which the torque converter clutch is engaging are the red points, and the points at which the clutch is disengaging are the green points, in the domain of engine speed and driver power demand. The points at which the clutch of the torque converter is engaging and disengaging are clearly visible in the form of lines. When the power demand increases, the clutch is engaged to minimize power loss from the torque converter.

### 5.3. Deceleration Fuel Cutoff

Deceleration fuel cutoff (DFCO), a feature that many modern-day engine control units (ECUs) support, detects whether the vehicle is coasting and, if certain operating conditions are met, cuts fuel to the engine and allows the wheels to keep the engine running. In this section, the DFCO enabling conditions will be determined in terms of vehicle speed, wheel torque requirement and CVT ratio choices.

In Figure 49 DFCO is active only when the wheel torque is negative, especially when the vehicle speed is above about 8 mph. In low vehicle speed cases (below about 14 mph), the wheel torque criterion of DFCO is decreased by 100 Nm or less.

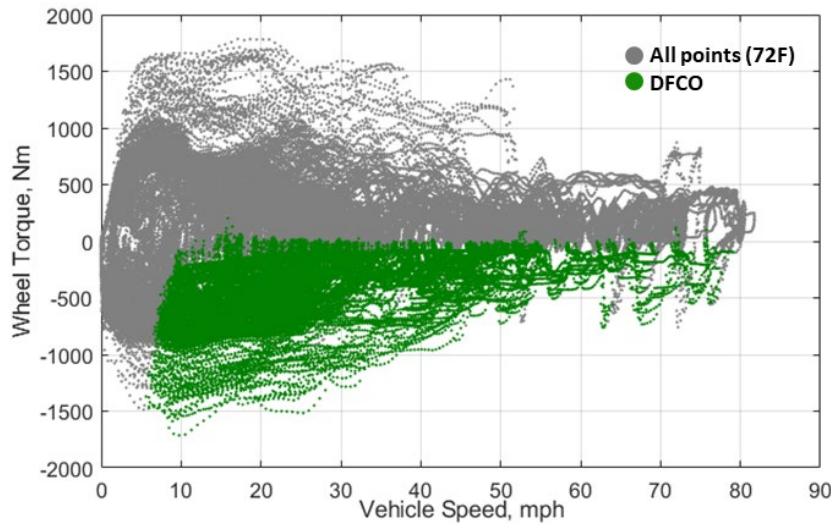


Figure 49: Operation of DFCO — vehicle speed vs wheel torque

Figure 50 shows that DFCO does not activate above a CVT gear ratio of about 2.2, even when the vehicle is braking.

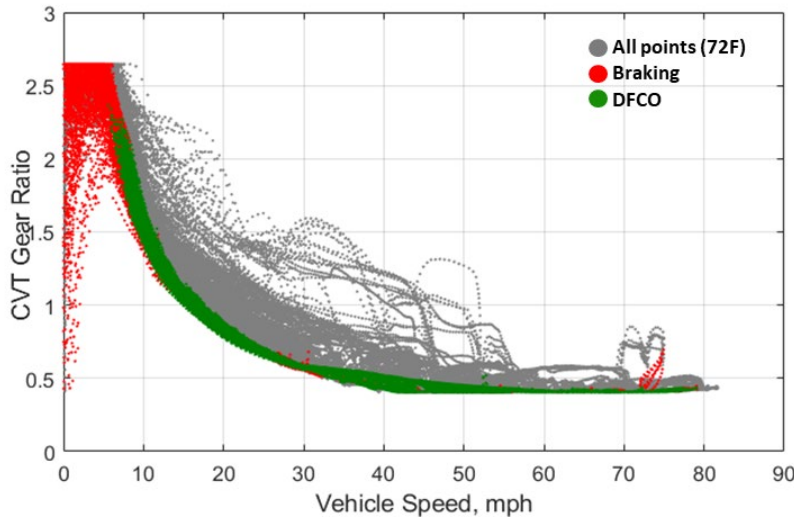


Figure 50: Operation of DFCO — vehicle speed vs CVT gear ratio

#### 5.4. Brake Energy Regeneration Systems

Unlike a mechanical braking device, Brake energy recovery systems convert otherwise dissipated energy of the vehicle during deceleration into electrical energy using an alternator. This feature is similar to regenerative braking common in electric drive vehicles, but results in lower levels of power and stores the energy within a traditional low-voltage battery. Figure 51 shows the brake energy regeneration system points in a plot of engine output power against alternator mechanical power.

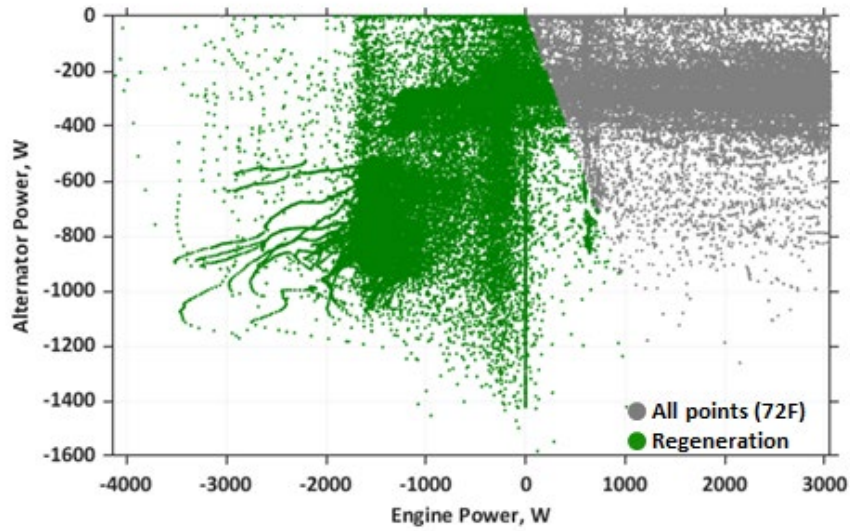


Figure 51: Brake energy regeneration system points — engine power vs alternator power

Figure 52 shows brake energy regeneration events as a function of vehicle speed against wheel torque. It can be seen that this system works in most deceleration situations except at low vehicle speeds (below about 3 mph).

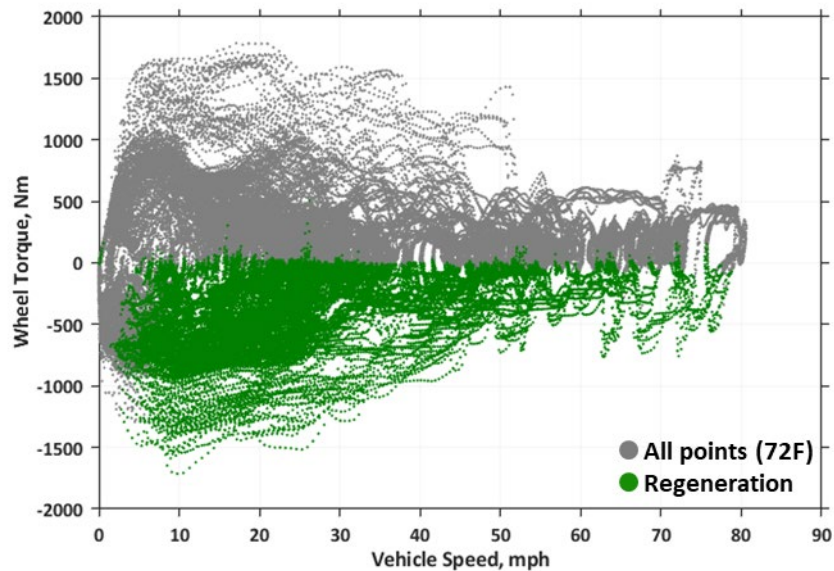


Figure 52: Regenerative braking points — vehicle speed vs wheel torque

Figure 53 shows mechanical braking power and alternator power on the UDDS cycle. The alternator power ranges from about 300 to 1,000 W when the vehicle is decelerating. The electric machine of the Honda Accord is not designed to maximize this energy capture as a means to reduce the overall energy consumption.

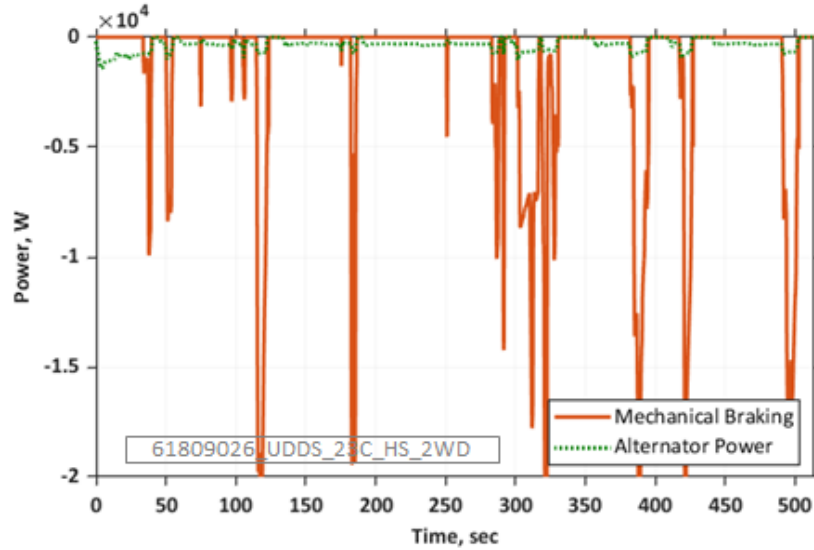


Figure 53: Mechanical braking power and alternator power on UDDS cycle

## 5.5. Engine Operation

### 5.5.1. Fuel Rate Map

The engine fuel rate map was generated from the engine mapping test data, shown in Figure 54. Since the components modeled in Autonomie were assumed to be in their warmup state, data where the engine coolant temperature was above  $60\text{ }^{\circ}\text{C}$  is used for Autonomie. Figure 54 only shows the points at which the time derivative of the acceleration pedal is below  $0.1/\text{s}$ , and the engine coolant temperature is above  $60\text{ }^{\circ}\text{C}$ , from which it is assumed that the points are obtained under quasi-steady operating conditions.

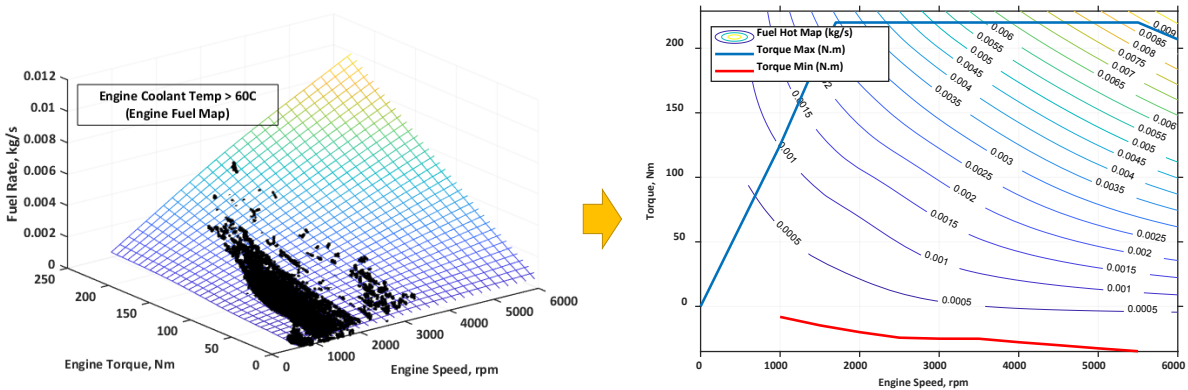


Figure 54: Engine fuel rate map according to engine speed and torque



### 5.5.2. Torque Pedal Map

The accelerator pedal is not a simple way of directly moving the throttle on the engine, because with an ECU, the traditional Bowden cable between the pedal and throttle is replaced with a pedal position sensor and a map. The torque pedal map does not depend on conditions like engine speed or transmission gear ratio. Instead, the engine throttle has a linear correlation with the middle of accelerator pedal in positions 0.25 to 0.6. A given pedal position and a given engine speed generate an engine torque demand, which is fed to the ECU to deliver the required amount of torque. In the low and high accelerator pedal positions, the engine throttle responds in a more gradual manner as shown in Figure 55.

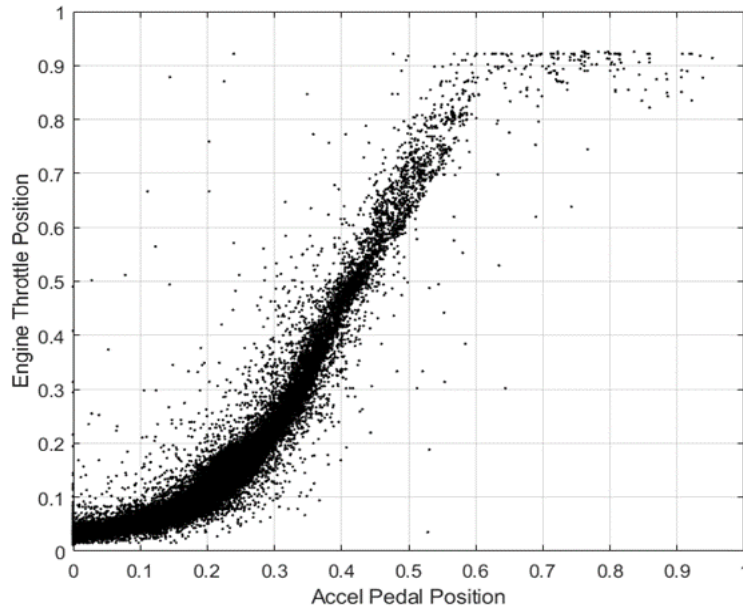


Figure 55: Torque pedal map

## 5.6. Thermal Management Impact on Vehicle Controls

In this section, we will focus on additional vehicle controls and how they are affected by thermal conditions. The impact of thermal conditions on performance and on the vehicle control are important issues. The effect of thermal conditions on control behavior will be discussed first, followed by performance analysis in different thermal conditions.

### 5.6.1. Engine Operation in Cold Conditions

Engine thermal management systems are designed to warm up the engine as quickly as possible with advanced techniques. However, it is difficult to completely avoid operating the engine at a low temperature. After startup and while engine is still idling, the coolant temperature is still low and more fuel than normal is needed until the engine warms up to operating temperatures.

Figure 56 shows three different control behaviors based on different engine coolant temperatures:

- The engine operates normally and the coolant temperature is warm at start-up (hot start).
- When the coolant temperature is between 20 °C and 70 °C, the engine stays at a higher than normal idle speed, even if without power demand. This control behavior is specific to vehicle start-up, and the engine operates normally once the coolant temperature rises above approximately 70 °C.
- When the engine coolant temperature is very low (below 0 °C) in cold ambient temperatures, the engine operates at an higher speed until the engine coolant temperature warms up as shown in the figure below.

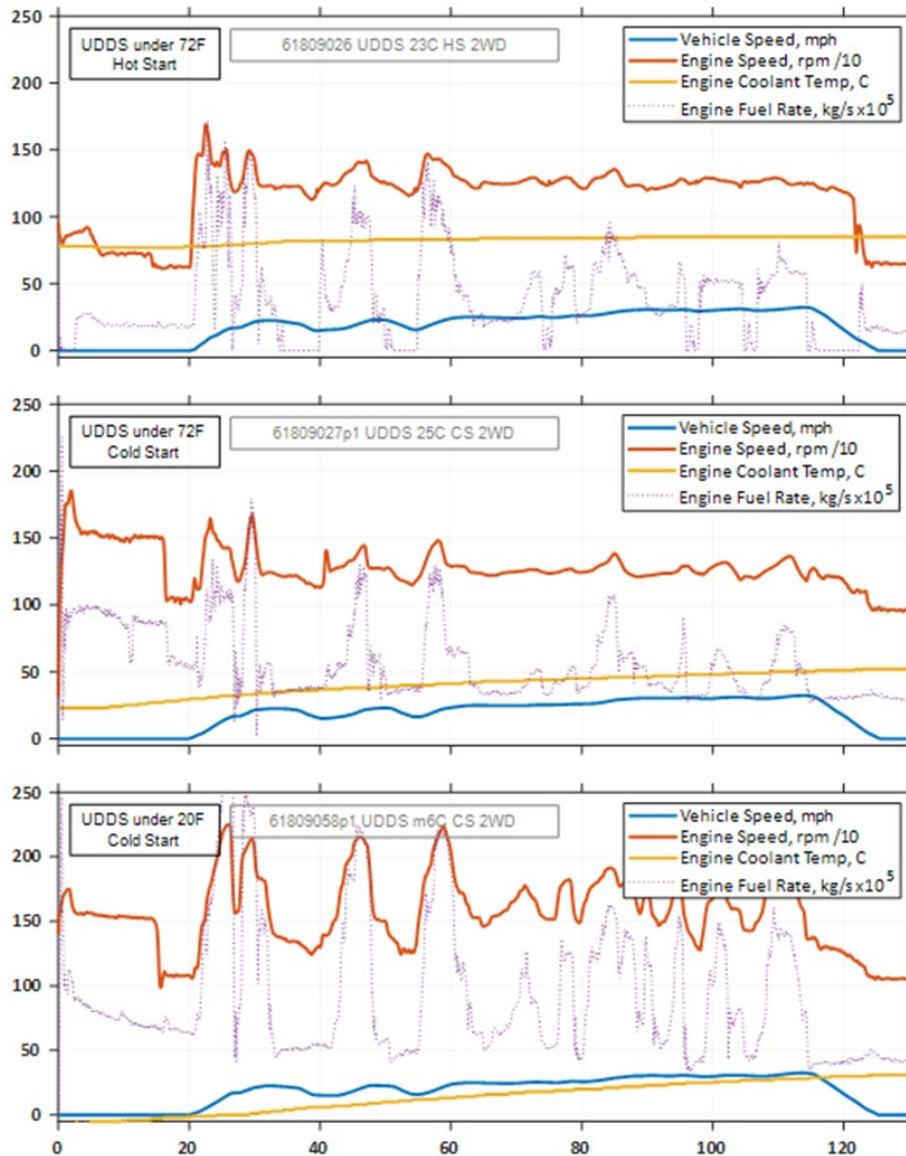


Figure 56: Engine operation at vehicle start-up differs according to the engine coolant temperature

As shown in Figure 57, the engine speed is controlled based on the engine coolant temperature, which means that the idle speed has a strong correlation with the engine coolant temperature in cold conditions.

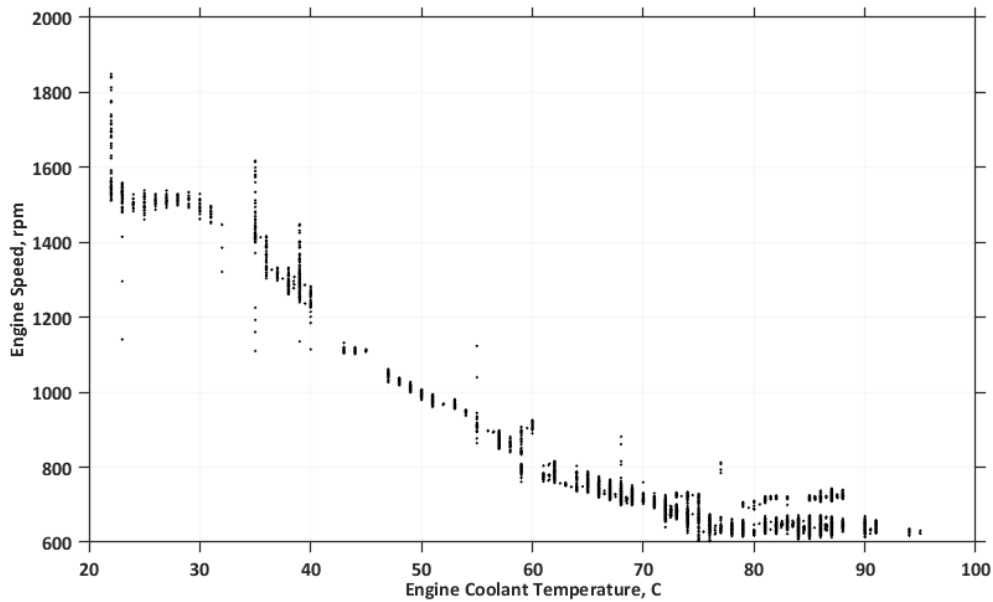


Figure 57: Engine idle speed is controlled by the coolant temperature

On the other hand, Figure 58 shows the effect of start-up coolant temperatures in driving conditions. The coolant does not reach its optimal coolant temperature within 1,200 second data window, as shown in the figure, when the vehicle is operated with the heater on in cold ambient temperatures.

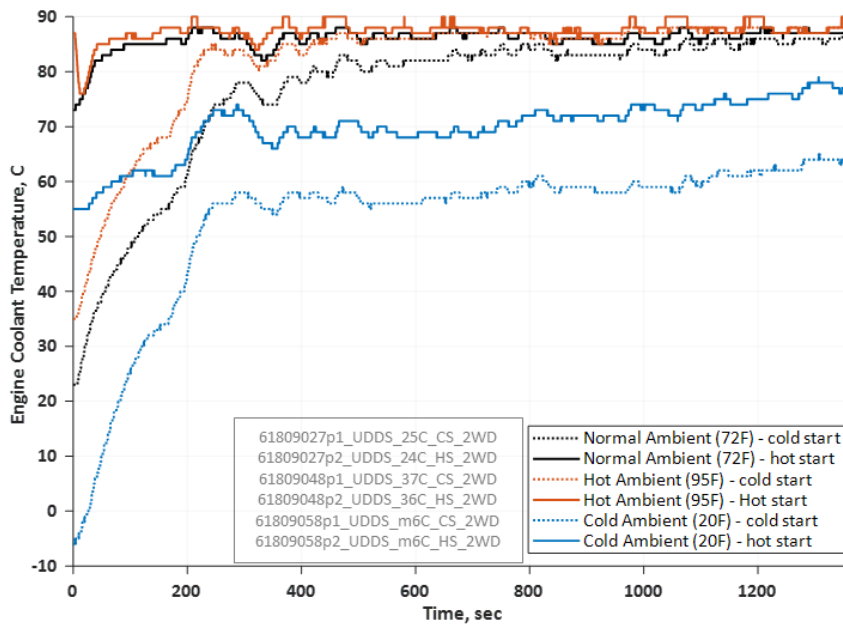


Figure 58: Behaviors of engine coolant temperatures on UDDS in different test conditions



### 5.6.2. Engine Performance

Thermal conditions affect not only components control but their performances as well. Engine performance noticeably deteriorates in very cold conditions. While we do not have complete component test data for different steady thermal conditions, the performance degradation caused by thermal conditions can be analyzed from the vehicle test data.

An engine generates a lot of heat. Approximately one third of the input power is converted to mechanical work, another third is removed through exhaust, and the last third contributes to heating the engine block and cooling system. Therefore, the engine temperature increases very quickly as long as the engine is operating, but a high coolant temperature is not sustained if the ambient temperature is very cold. Figure 59 shows that the fuel consumption rate is significantly affected by the thermal conditions.

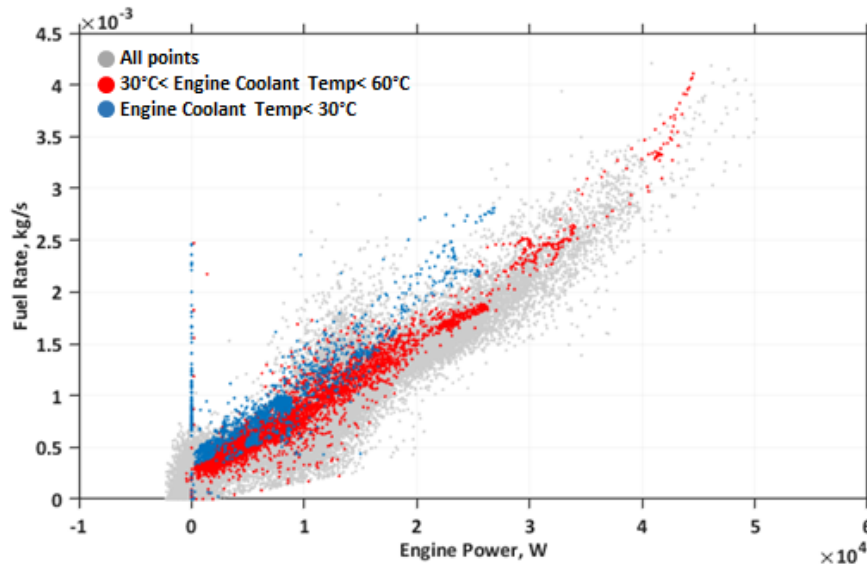


Figure 59: Fuel rate of engine by engine power for different coolant temperatures

The operating points in Figure 59 are grouped by engine coolant temperature range and show meaningful trends in fuel consumption. Although cylinder temperature might have a stronger correlation with engine efficiency than coolant temperature, it is not measured in our tests, and coolant temperature can be considered as the temperature closest to the heat source.

### 5.6.3. Fuel Consumption Analysis

Changes in engine temperature affects the vehicle fuel consumption. Figure 60 shows the fuel consumption in several UDDS tests under different test conditions.

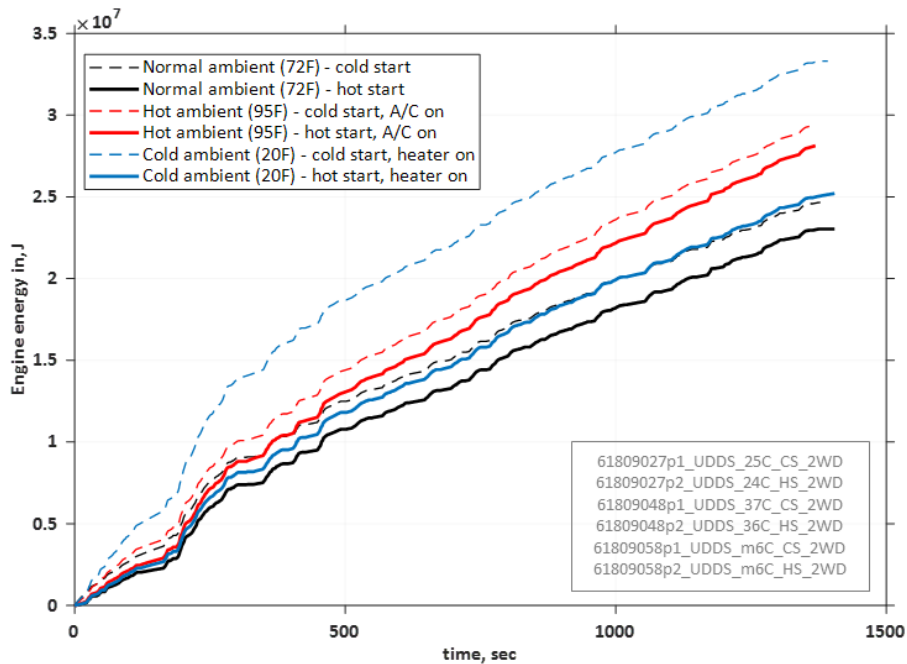


Figure 60: Accumulated fuel consumption trajectories on UDDS under different test conditions

The colors of the lines in Figure 62 indicate the ambient temperature, and a dotted line means that the engine was started at a cold temperature (cold-start). The results show that the car operated in normal ambient temperatures with the HVAC off shows the best fuel economy and that fuel economy decreases when the AC system is operating, although there are variations with initial engine temperature and transmission temperature. However, the vehicle operated at cold ambient temperatures consumed only about 10% more fuel if the engine was started in a “hot” condition. At certain hot ambient temperatures, fuel consumption is higher than at cold ambient temperatures, since the AC system consumes more energy than the heating system. Fuel consumption is dramatically increased if the engine starts at a cold temperature and the cabin heater is turned on, because the engine cannot use all the waste heat to increase the engine temperature. When the engine temperature is not well maintained, the engine consumes more fuel, which leads to lower fuel economy.

Figure 61 shows the effect of ambient temperature and engine coolant temperature on engine efficiency. If we look at the losses in engine, computed as the difference between fuel energy input and the mechanical energy output, we can see this impact. In cold ambient and cold start conditions, the engine loses more energy in the energy conversion process. However, in normal ambient conditions, if the engine temperature rises, even after a cold start, the energy loss becomes similar to that with a hot start.

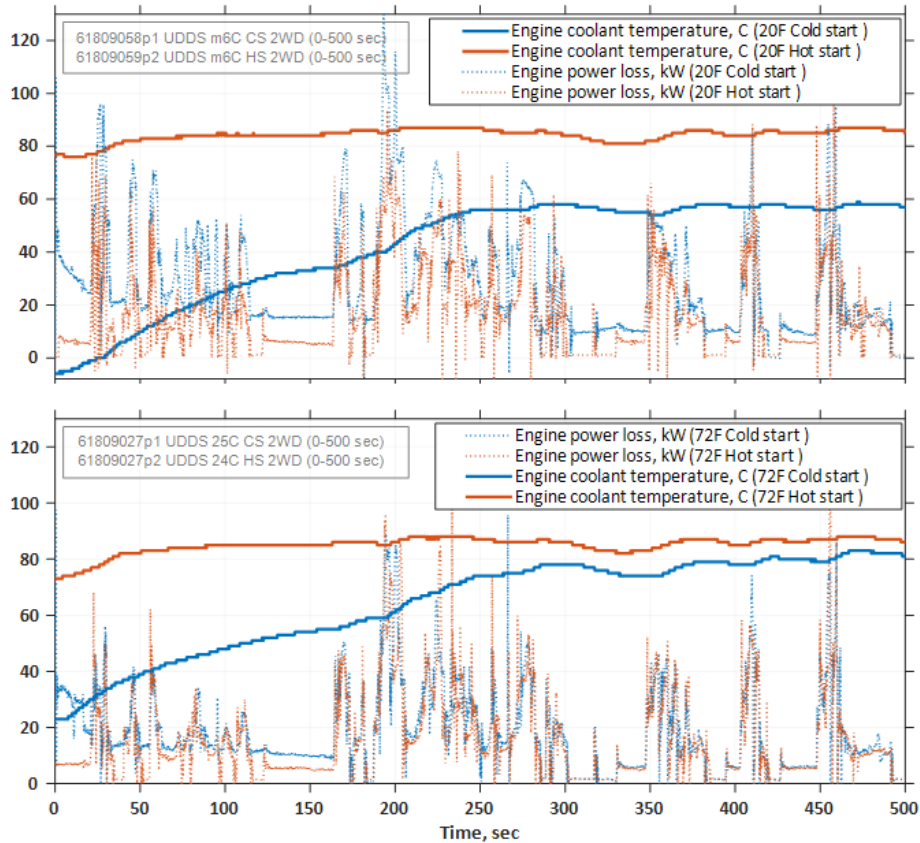


Figure 61: Engine power loss and engine coolant temperature according to driving conditions

### 5.7. Accessory Load

There is no electrical heater for the cabin in 2018 Honda Accord, so the most significant impact on the accessory load is caused by the HVAC system in hot ambient conditions. There are two kinds of accessory loads for HVAC: the first is the electrical accessory load from the battery for operating the ventilation fan, and the second is the mechanical accessory load from the engine for operating the water pump and compressor. While we have data for the electrical accessory load in our test data, we do not have the specific mechanical accessory load. However, we can deduce it from the information shown in Figure 62, which shows the engine output power when the vehicle is fully stopped in both hot and cold ambient conditions. More energy is consumed in hot ambient conditions than in cold ambient condition, and with the vehicle stopped it can be deduced that the AC compressor consumes about 280 W.

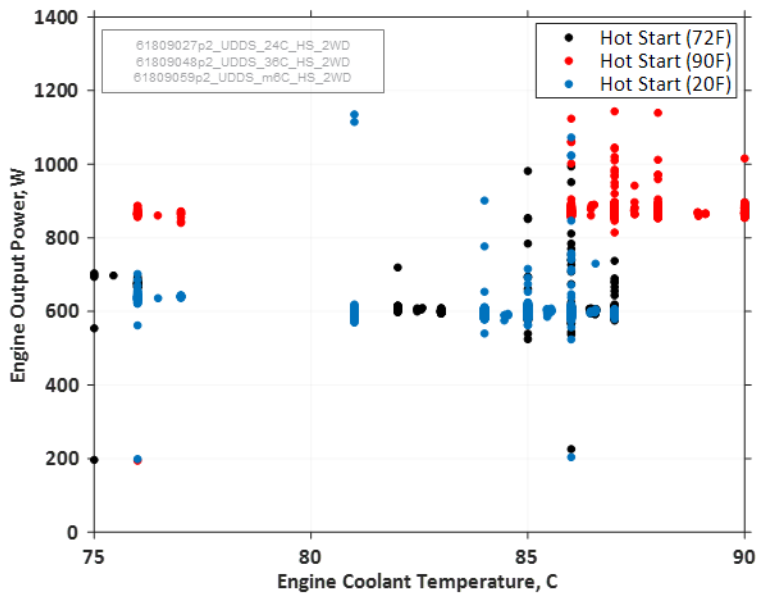


Figure 62: Engine output power when the vehicle is fully stopped

Figure 63 shows the accessory power when the vehicle is fully stopped. The operating points are grouped according to operating conditions: The black points are the accessory power consumed when the AC and heater are turned off. The required power without HVAC is about 230 W, regardless of the thermal conditions. The battery power consumption increases 180 W when the AC system in the passenger compartment is turned on in hot conditions (red points). In cold conditions (ambient temperature below 0 °C) when the heater is turned on, the battery power consumption increases by about 80 W (blue dots). Because only the fan operates, blowing hot air from the engine into the cabin, the power required for heating is relatively small compared to that for the AC system.

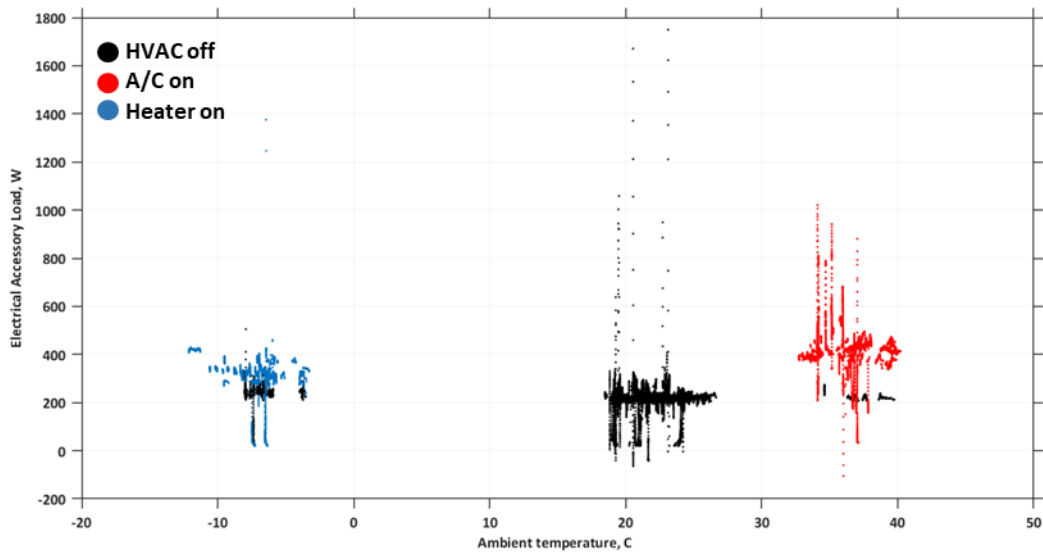


Figure 63: Electrical consumption when the vehicle is fully stopped

## 5.8. Energy Balance Diagram

In section 5.1 we saw that the signals which were not recorded in test data were calculated based on other signals. This is based on vehicle characteristics and other component assumptions. Based on these additional signals for each component, the total amounts of energy in and out are computed by post-processing in Autonomie. The terms “input” and “output” can be confusing, because their roles can be exchanged. Therefore, in this discussion, each port means one power flow, and all components have two ports in Autonomie. For example, Figure 64 shows the energy in and out for two ports, and the efficiency values for the final drive component.

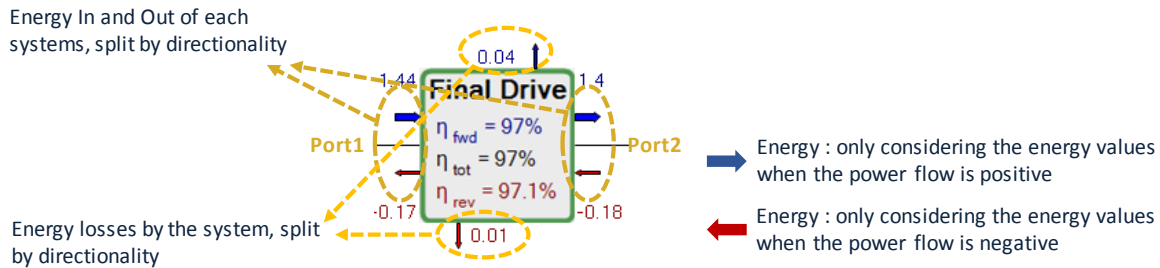


Figure 64: Example of energy calculation for one component on Autonomie

Where the total efficiency is computed on a given port in different ways, the following is the definition of efficiency values:.

- $\eta_{fwd}$  Total efficiency when the power on port 1 and 2 is positive (positive positive)
- $\eta_{tot}$  Total aggregate efficiency
- $\eta_{rev}$  Total efficiency when the power on port 1 and 2 is negative (negative negative)

For each component, the total energy consumption and efficiency are calculated based on the effort and flow signals recorded or estimated for those components. Figure 65 and Figure 66 show the final diagrams from the Autonomie graphical interface after post-processing for the energy balance on UDDS and HWFET cycles.

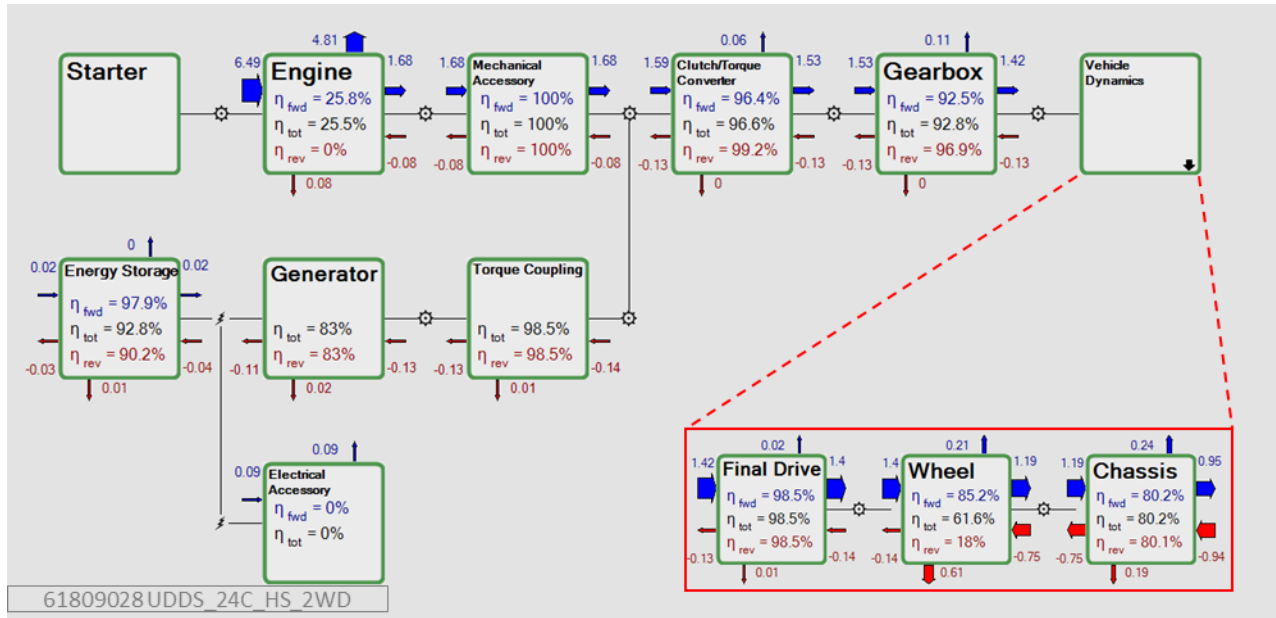


Figure 65: Energy balance diagram on UDDS in Autonomie

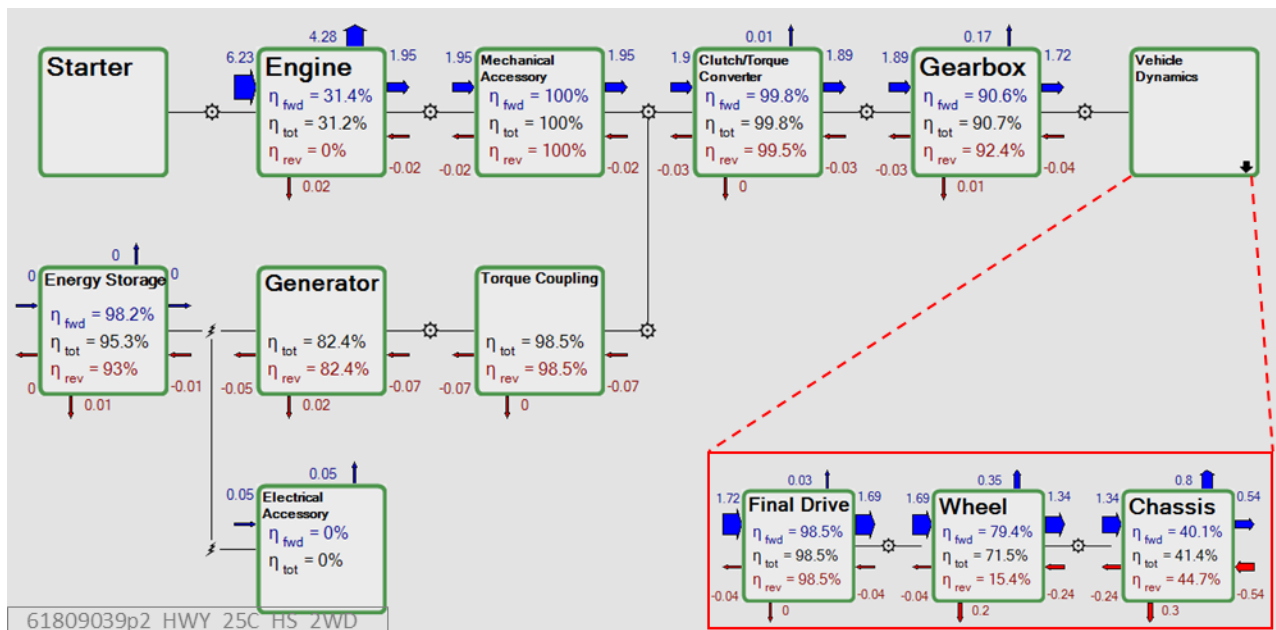


Figure 66: Energy balance diagram on HWFET in Autonomie

## 6. Autonomie Model Validation

An analysis of controls parameters was done based on the test data. Vehicle and component control logics and component models were updated based on this analysis. The component controls include transmission shifting, torque converter lockup, engine fuel cutoff, and so on. The updated component models, including control models, were implemented and integrated in Autonomie to create a vehicle simulation model for the 2018 Honda Accord. The vehicle model is simulated as a “warmed up” vehicle. Since all the simulations considered in this report assume a “hot start,” where the engine coolant temperature is steady at around 95 °C, the cold start condition was not a factor for the simulations.

The validation process for this study is shown in Figure 67.

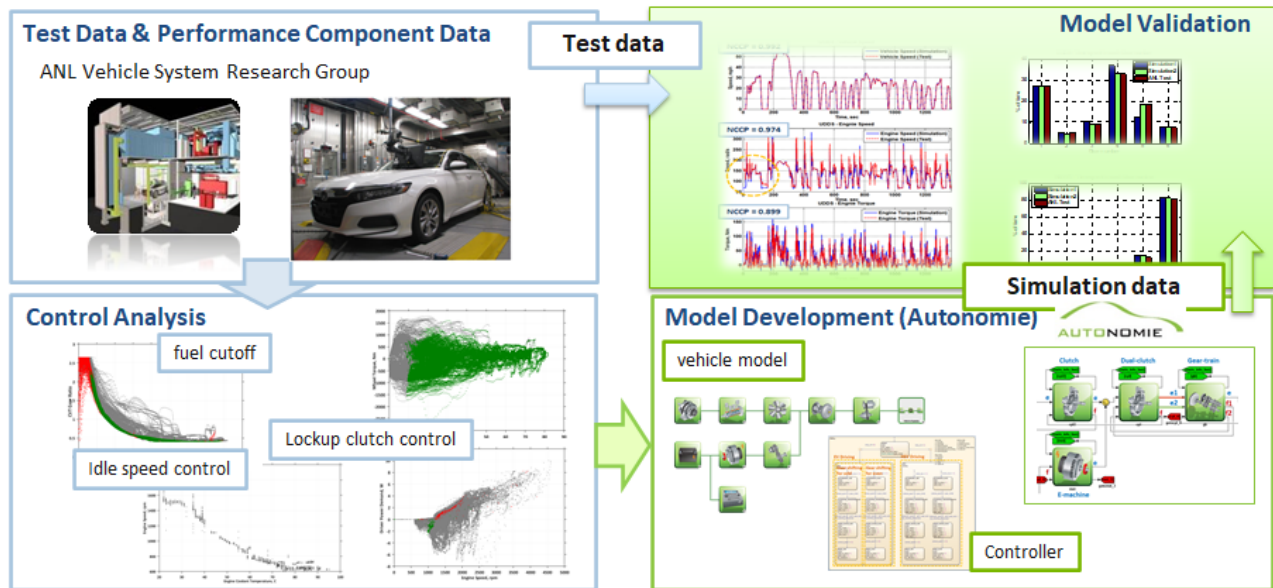


Figure 67: Validation process for 2018 Honda Accord in Autonomie

The simulation was conducted in urban dynamometer driving schedule (UDDS), highway fuel economy test (HWFET), and US06 cycles. Figure 68, Figure 69 and Figure 70 show the vehicle speed, engine speed, engine torque, wheel power, gear number, and fuel integrated of the simulation results and the test data, which match well for each cycle.



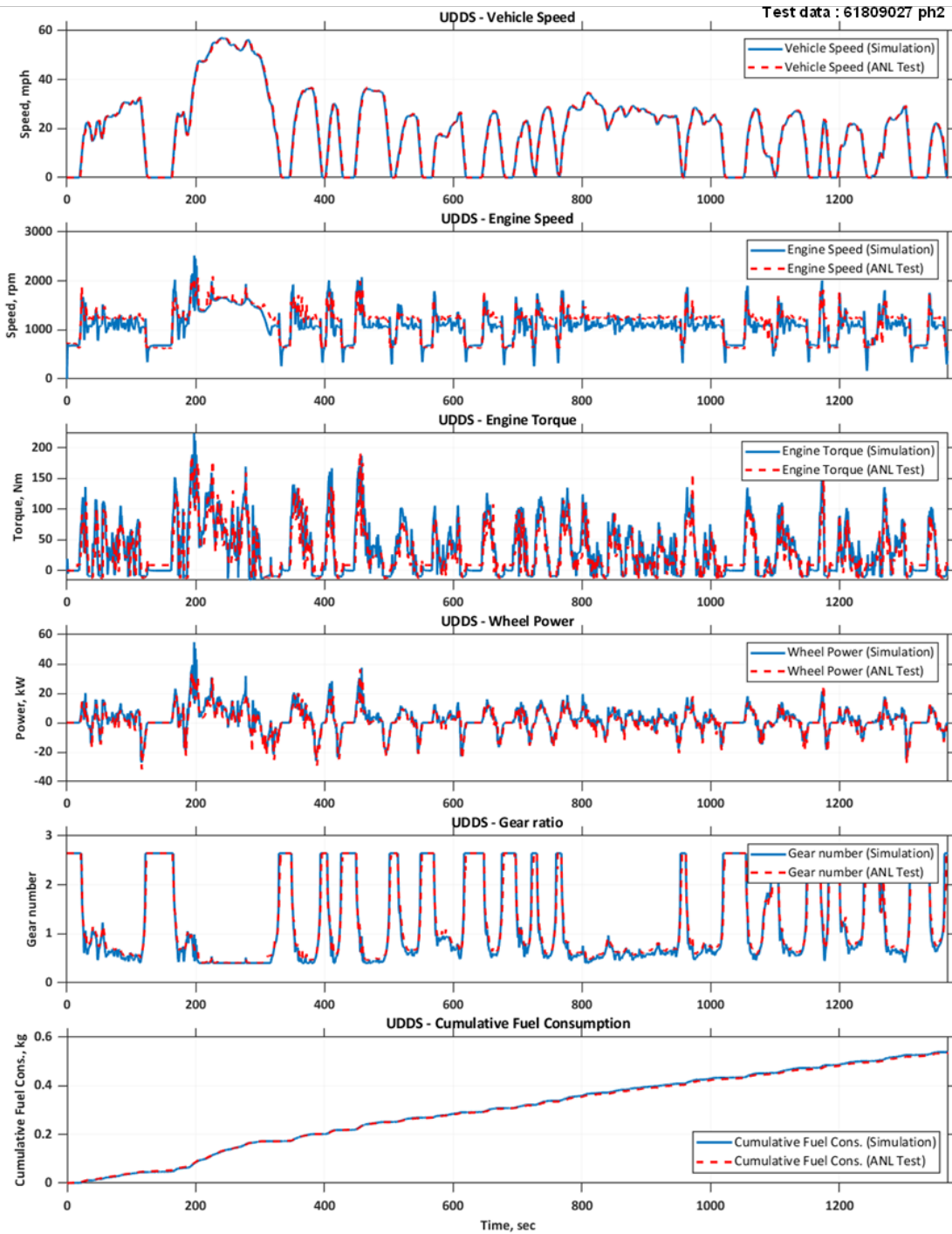


Figure 68: Simulation results and test data for UDDS cycle



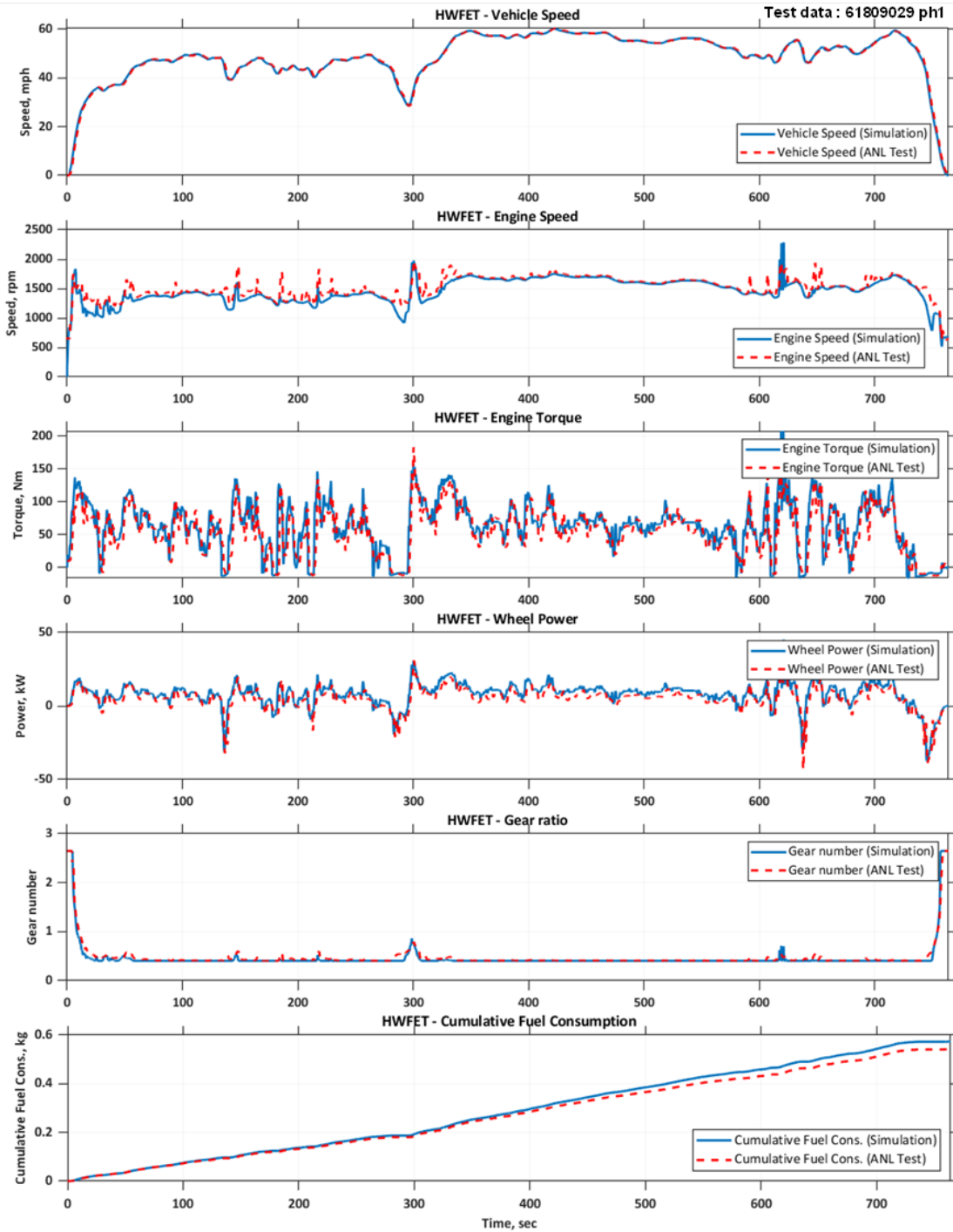


Figure 69: Simulation results and test data for HWFET cycle

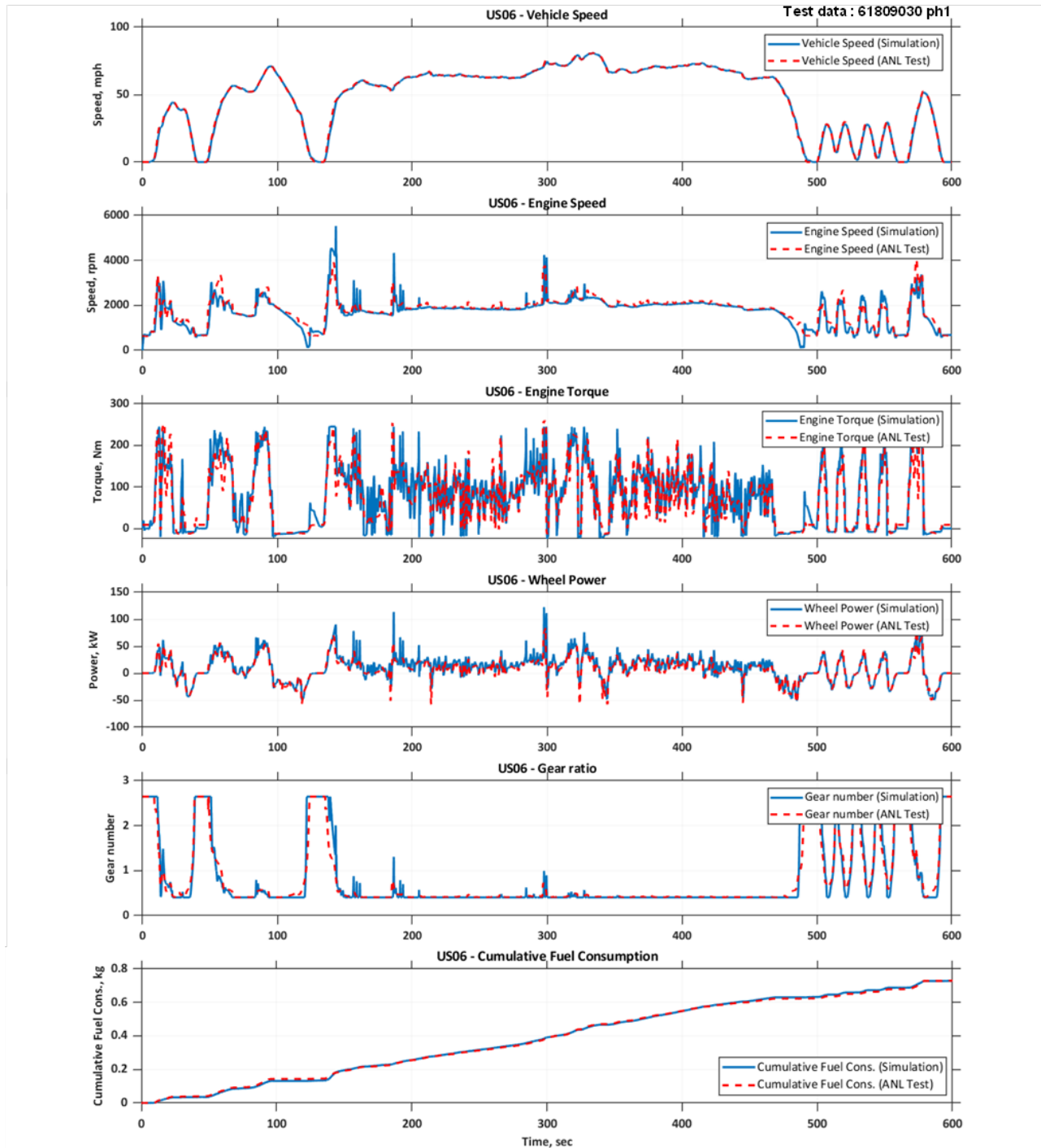


Figure 70: Simulation results and test data for US06 cycle

Normalized cross-correlation power (NCCP) was used to compare second-by-second time-varying signal traces between test and simulation [8]. The NCCP was calculated using Equations 9 and 10 as follows: Here  $x$  and  $y$  represent individual signals.

$$NCCP = \frac{\max\{R_{xy}(\tau)\}}{\max\{R_{xx}(\tau), R_{yy}(\tau)\}}$$

Equation 8

$$R_{xy}(\tau) = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T x(t) \cdot y(t - \tau) dt$$

Equation 9

The NCCP values of simulation results for UDDS, HWFET and US06 cycles are shown in Table 18. It can be seen that the values for vehicle speed, gear number and engine speed, which all exceed 0.9, indicate a high level of correlation, while there is relatively lower correlation in the engine torque.

Table 18: The NCCP values for UDDS, HWFET and US06 cycle

	<b>UDDS</b> (test data: 61809027 Ph2)	<b>HWFET</b> (test data: 61809029 Ph1)	<b>US06</b> (test data: 61809030 Ph1)
<b>Vehicle speed</b>	0.994	0.999	0.997
<b>Gear number</b>	0.932	0.970	0.945
<b>Engine speed</b>	0.925	0.959	0.956
<b>Engine torque</b>	0.837	0.895	0.917

Figure 71 and Figure 72 show the vehicle speed when the torque converter is locked. The torque converter lockup status, based on vehicle speed and engine speed, for the simulation results and test data was compared for UDDS (test data: 61809027 Ph2), HWFET (test data: 61809029 Ph1), and US06 cycles (test data: 61809030 Ph1) in Figure 71, which shows that the simulated torque converter operation was similar to that in the test data. In Figure 72, the engine fuel cutoff status was compared with the test data for UDDS (test data: 61809027 Ph2), HWFET (test data: 61809029 Ph1), and US06 cycles (test data: 61809030 Ph1). The engine fuel cutoff in simulation showed tendencies similar to the test data.

The percentages of times for torque converter lockup and engine fuel cutoff are shown in Table 18.

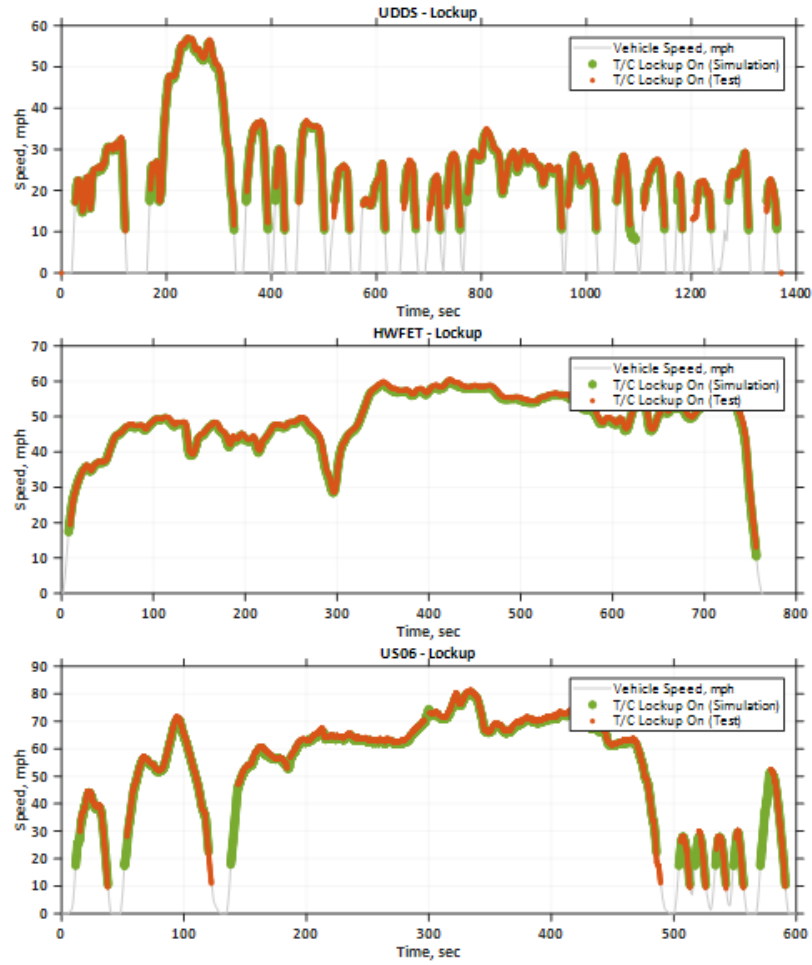


Figure 71: Torque converter locked vs vehicle speed

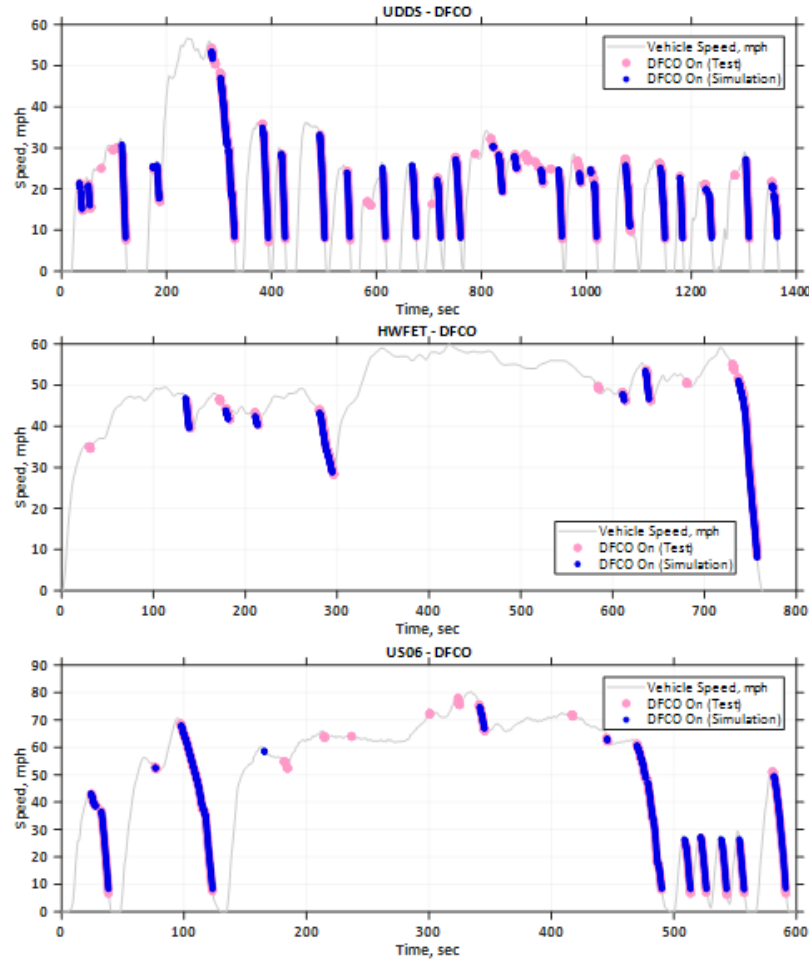


Figure 72: Engine fuel cutoff vs vehicle speed

Table 19: Percentage of times torque converter lockup and DFCO on

		<b>UDDS</b> (test data: 61809027 Ph2)	<b>HWFET</b> (test data: 61809029 Ph1)	<b>US06</b> (test data: 61809030 Ph1)
<b>TC lockup (%)</b>	Test	67.01	97.85	78.02
	Simulation	65.42	98.00	83.05
<b>DFCO on (%)</b>	Test	14.04	8.40	15.95
	Simulation	13.03	6.56	14.57

In Table 19, the simulated fuel consumption is compared to the measured average fuel consumption under hot conditions to validate the simulation performance. The results showed that the fuel consumption of the simulation, 6.4, 4.5 and 7.7 L/100km on UDDS, HWFET & US06 cycles, differed from the test result by 0,2%, 4% and 1%, respectively.

Table 20: Fuel consumption of test data and simulation results

<b>Fuel economy (L/100km)</b>	<b>UDDS</b>	<b>HWFET</b>	<b>US06</b>
<b>Test average</b>	6.1	4.51	7.73
<b>Simulation (error)</b>	6.09 (0.2%)	4.69 (4%)	7.65 (1%)
Test data for UDDS: 61809027 Ph2 Test data for HWFET: 61809029 Ph1 Test data for US06: 61809030 Ph1			

## 7. Conclusions

The vehicle benchmarked in this report is a 2018 Honda Accord equipped with the 1.5 L, I4, “Earth Dreams” engine coupled to a continuously variable automatic transmission. This particular powertrain configuration provided higher than average fuel economy compared to other vehicles in its class, without sacrificing vehicle performance. The focus of the benchmark was to understand the usage of the critical powertrain components and their impact on the vehicle efficiency. The vehicle was instrumented to provide data to support the model development and validation in conjunction to providing the data for the analysis in the report. The vehicle was tested on a chassis dynamometer in the controlled laboratory environment across a range of certification tests. Further tests were performed to map the different powertrain components.

Vehicle control aspects pertaining to CVT, torque converter, DFCO, use of alternator to recuperate part of the braking energy, techniques adopted to manage the operating temperature of the engine were explained. HVAC loads for compressor and fan under both hot and cold ambient temperatures were quantified. This information was used to develop and calibrate the Autonomie models. The fuel economy results and operating conditions of various components predicted by Autonomie models were found to be well correlated to the test data. This effort increases the confidence in the modelling capabilities of Autonomie, and provides us a validated benchmark for a midsize car with a downsized boosted SI engine paired with a CVT. Lessons learned from this effort will be applied to all future work where similar technologies are evaluated.

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**Appendix A: Vehicle Build Sheet**



### 2018 ACCORD 1.5T LX

LX : PLATINUM WHITE P. ENGINE NUMBER: L15BE-2660853  
INT: IVORY

#### STANDARD EQUIPMENT AT NO EXTRA COST

- \* TECHNICAL FEATURES \***
  - 192hp 1.5-Liter Direct Injection Turbo-Charged 4-Cylinder Engine
  - Continuously Variable Transmission (CVT)
  - 4-Wheel Disc Brakes
  - Electric Power Steering
  - Hill Start Assist

- \* SAFETY FEATURES \***
  - Drivers and Front Passenger's Airbags
  - Drivers and Front Passenger's Side Airbags
  - Drivers and Front Passenger's Knee Airbags
  - Side Curtain Airbags with Rollover Sensor
  - Anti-Lock Braking System (ABS)
  - Electronic Brake Distribution (EBD)
  - Vehicle Stability Assist (VSA)
  - Tire Pressure Monitoring System
  - LED Daytime Running Lights
  - LATCH System for Child Seats

- \* INTERIOR FEATURES \***
  - Audio System with 4 Speakers
  - Color LCD Screen and Multi-View Rear Camera
  - Bluetooth HandsFreeLink
  - USB Audio Interface
  - Driver Attention Monitor

- Dual-Zone Automatic Climate Control with Air Filtration System
- Push-Button Start
- Driver's Seat Height Adjustment
- Fold-Down Rear Seat Backer Armrest
- Power Windows and Door Locks
- Front Auto Up/Down Windows
- Illuminated Visor Vanity Mirrors
- Sunglasses Holder
- Exterior Temperature Display
- Fold-Down Rear Seatback
- Rear Mirrors
- 12-Volt Power Outlets
- Electric Parking Brake

- \* EXTERIOR FEATURES \***
  - 17" Alloy Wheels
  - P225/50 R17 All-Season Tires
  - Auto-On/Off Headlights
  - Power Door Mirrors
  - Remote Entry with Security System
  - Capless Fuel Filler

- \* HONDA SENSING \***
  - Adaptive Cruise Control (ACC)
  - 1st Lane-Speed Follow
  - Collision Mitigation Braking System (CMBS)
  - Lane Keeping Assist System (LKAS)
  - Head Departure Mitigation (HDM)

Manufacturer's Suggested Retail Price **\$23,570.00**

Full Tank of Fuel **No Charge**

Honda Roadside Assistance  
3YR/36K Mile Warranty Term

Destination and Handling **890.00**

**TOTAL VEHICLE PRICE**  
(Includes Pre-Delivery Service)  
**\$24,460.00**

License and title fees, state and local taxes and dealer options and accessories are not included in the manufacturer's suggested retail price.

\*HSC 39037.05 Low-Emission Motor Vehicle\*

CONTINENTAL HONDA  
5901 SO. LA GRANGE RD.  
COUNTRYSIDE, IL 60525

PORT OF ENTRY: MARYSVILLE  
DELIVERY POINT: SCHAUWIBURG  
SHIP:  
HOWSPACE: 310-303  
TRANS.METHOD: TRUCK

ORIG. D.H. # : XBM4  
REF. NO.: 42691  
HN CODE: HN-0001  
MISSION: 50 STATE  
CONTROL NO: 590451  
DEALER: 2088\*



### Fuel Economy and Environment

Gasoline Vehicle

**Fuel Economy**

**33** MPG  
combined city/hwy

**30** city  
**38** highway

**3.0** gal/100 mi

These estimates reflect new EPA methods beginning with 2017 models.

Long-Range range from 14 to 104 MPG. The most vehicle score 136 MPG.

**You Save \$1,250**  
in fuel costs over 5 years compared to the average new vehicle.

**Annual fuel cost \$1,100**

Fuel Economy & Greenhouse Gas Rating (EPA) (EPA) Smog Rating (EPA) (EPA)



City: 24 mpg, hwy: 34 mpg, combined: 28 mpg (EPA est.). The best is a 2017 Toyota Camry Hybrid LE (EPA est.) with 52 city and 48 hwy (combined) mpg. The worst is a 2017 Ford Focus SE (EPA est.) with 24 city and 34 hwy (combined) mpg.

Actual results will vary for many reasons, including driving conditions and how you drive and maintain your vehicle. The average new vehicle gets 27 MPG and costs \$8,760 to fuel over 5 years. Cost estimates are based on 15,000 miles per year at \$2.40 per gallon. MPGe is miles per gasoline gallon equivalent. Vehicle emissions are a significant cause of climate change and smog.

**fuel economy.gov**

Calculate personalized estimates and compare vehicles.



#### PARTS CONTENT INFORMATION

FOR VEHICLES IN THIS CARLINE  
U.S./Canadian Parts Content: **60 %**

NOTE: Parts content does not include final assembly, distribution or other non-parts costs.

#### GOVERNMENT 5-STAR SAFETY RATING

**Overall Vehicle Score To Be Rated**

Based on the combined ratings of frontal, side and rollover. Should ONLY be compared to other vehicles of similar size and weight.

<b>Frontal Crash</b>	<b>Driver Passenger</b>	<b>To Be Rated</b>
----------------------	-------------------------	--------------------

Based on the risk of injury in a frontal impact. Should ONLY be compared to other vehicles of similar size and weight.

<b>Side Crash</b>	<b>Front seat Rear seat</b>	<b>To Be Rated</b>
-------------------	-----------------------------	--------------------

Based on the risk of injury in a side impact.

<b>Rollover</b>	<b>To Be Rated</b>
-----------------	--------------------

Based on the risk of rollover in a single vehicle crash.

Star Ratings range from 1 to 5 stars (\*\*\*\*) with 5 being the highest. Source: National Highway Traffic Safety Administration (NHTSA). [www.safercar.gov](http://www.safercar.gov) or 1-888-327-4236

This vehicle is equipped with bumpers that can withstand an impact of 2.5 miles per hour with no damage to the vehicle's body and safety systems, although the bumper and related components may sustain damage. The bumper system on this vehicle conforms to the current federal bumper standard of 2.5 miles per hour.

FOR THIS VEHICLE  
Final Assembly Point:  
**MARYSVILLE, OHIO USA**

Country of Origin: Engine:  
**U.S.A.**  
Transmission:  
**U.S.A.**

## **Appendix B: Subset of Midsized Cars for Comparative Analysis**





## **Appendix C: A018 Honda Accord LX Test Signals**

The following signals were collected at 10Hz for each test, and are publicly posted for reference. Note, signal sampling rate for CAN and diagnostic messages is dependent on the vehicle, and the actual transmission rate may be faster or slower than the 10hz sample rate. Additionally, though most signals are available for the duration of testing, some errors in acquisition can occur.

Facility, dyno, bench, and cell data	Analog data from vehicle	CAN: Broadcast data	CAN: Diagnostic data
Time[s] RawFacilities	Engine Oil Dipstick Temp C	Eng speed CAN7 rpm	12VBatt estimated temp PGM C
Dyno Spd mph	Radiator Air Outlet Temp C	Eng torque trans demand CAN7 Nm	Brake pressure 1 ABS bar
Dyno TractiveForce N	Engine Bay Temp C	Trans secondary shaft spd CAN7 rpm	Eng airfuel FB commanded PGM
Dyno LoadCell N	Cabin Temp C	Eng DFCO signal 1 CAN7	Eng airfuel FB ST fuel trim level PGM
Distance mi	Cabin Upper Vent Temp C	Pedal brake press CAN2	Eng airfuel FB ST LT fuel trim level PGM
Tailpipe Press inH2O	Cabin Lower Vent Temp C	Pedal accel pos CAN2 per	Eng airfuel ratio PGM
Cell Temp C	12VBatt Volt Hioki U1 V	Pedal brake state CAN2	Eng CMP control PGM deg
Cell RH %	Alternator Curr Hioki I1 A	Vehicle speed CAN2 mph	Eng EVAP PC duty PGM per
Cell Press inHg	Alternator Power Hioki P1 kW	Veh odometer displayed BCAN	Eng FSS PGM
Tire Front Temp C	12VBatt Pos Curr Hioki I2 A	Pedal brake press CAN2	Eng fuel injector duration PGM ms
Tire Rear Temp C	12VBatt Pos Power Hioki P2 W	Pedal accel pos CAN2 per	Eng idling PGM
Drive Schedule Time s	12VBatt Power Hioki P3 W	Pedal brake state CAN2	Eng intakeair temp 1 PGM C
Drive Trace Schedule mph	12VBatt Curr Hioki I3 A	Vehicle speed CAN2 mph	Eng knock control PGM
Exhaust Bag	12VBatt Volt Hioki U3 V	Veh odometer displayed BCAN	Eng knock retard PGM deg
Solar Array Ind Temp C	12VBatt Volt Hioki U1 V	Veh fuel use current drive BCAN	Eng MAF sensor PGM gps
AMA Dilute THC[mg/s]	Eng FuelFlow Direct2 gps	Veh fuel use prev drive BCAN	Eng MIL indication PGM
AMA Dilute CH4[mg/s]	Eng FuelFlow Direct ccps	Veh PRNDL drive BCAN	Eng MIL status PGM
AMA Dilute NOx[mg/s]	Eng Fuel Temp Direct C	Veh PRNDL low BCAN	Eng rocker arm oil pressure sensor PGM kPa
AMA Dilute COlow[mg/s]		Veh PRNDL neutral BCAN	Eng run time PGM s
AMA Dilute COmid[mg/s]		Veh PRNDL park BCAN	Eng spark advance PGM deg
AMA Dilute CO2[mg/s]		Veh PRNDL rev BCAN	Eng starting engine coolant temp PGM C
AMA Dilute HFID[mg/s]		Veh PRNDL sport BCAN	Eng TC air bypass sol valve position PGM
AMA Dilute NMHC[mg/s]		Veh econ mode BCAN	Eng TC boost pressure PGM kPa
AMA Dilute Fuel[g/s]		HVAC AC button BCAN	Eng throttle valve PGM deg
		HVAC auto set BCAN	Eng VTEC solenoid valve PGM
		HVAC driver temp BCAN F	Exhaust catalyst temp PGM
		HVAC fan setting BCAN	Fuel level average PGM per
		HVAC outside air BCAN	HVAC AC clutch PGM
		HVAC pass temp BCAN F	HVAC AC pressure sensor PGM kPa
		HVAC rear defrost BCAN	HVAC AC switch PGM
		HVAC sync button BCAN	Trans ATF temp AT C
		HVAC vent pos BCAN	Trans driven pulley pressure AT kPa
		Eng MAP sensor highres CAN7 kPa	Trans input shart drive pulley speed AT rpm
		Trans pulley_ratio_CAN7	Trans LCC_linear_sol_actual_AT_A
			Trans secondary shaft spd 1 AT rpm
			Trans secondary shaft spd AT kph
			Trans torque converter turbine spd AT rpm
			Veh speed PGM KPH
			Veh wheel spd LF ABS kph
			Veh wheel spd LR ABS kph
			Veh wheel spd RF ABS kph
			Veh wheel spd RR ABS kph



## **Appendix D: Test Summary**

(Also available at [www.anl.gov/d3](http://www.anl.gov/d3))

Test ID #	Cycle	Test Time	Start Comments	End Comments	Test Cell Temp (°C)	Test Cell RH (%)	Test Cell Dew (inHg)	Test weight (lb)	Dyne Target A	Dyne Target B	Dyne Target C	Cycle Distance (mi)	Cycle Fuel Consumption (gal)	Cycle Fuel Conversion (lb)	Fuel Used (mod gal)	Electrical Energy Usage												Test Drive	Fuel Batch	Fuel Heating Value (Btu/lb)	APC/M	ASCR	ASC. d	ASC. l	CE. d	CE. l	BBR	BR	RW																		
																Pump #1 (kW)	Pump #2 (kW)	Pump #3 (kW)	Pump #4 (kW)	Pump #5 (kW)	Pump #6 (kW)	Pump #7 (kW)	Pump #8 (kW)	Pump #9 (kW)	Pump #10 (kW)	Pump #11 (kW)	Pump #12 (kW)													Pump #13 (kW)	Pump #14 (kW)	Pump #15 (kW)	Pump #16 (kW)	Pump #17 (kW)	Pump #18 (kW)	Pump #19 (kW)	Pump #20 (kW)	Pump #21 (kW)	Pump #22 (kW)	Pump #23 (kW)	Pump #24 (kW)	Pump #25 (kW)	Pump #26 (kW)	Pump #27 (kW)	Pump #28 (kW)	Pump #29 (kW)	Pump #30 (kW)
<b>Day 1, Coastdowns and UDDS Prep</b>																																																									
63090021	HVY2 w/ coastdowns p1	00/18/18, 02:56:33 PM	HVY2 and coastdowns, 2 bag, w/ 8 Handynum, vehicle charged to 20V mode	All, saved dyno tests acquired after 4 adjustments	35	43	20	3000	43.75	-0.0404	0.02019	10.28	0.174	34.3	0.194	52.9	46.066	5.327	1.027	-0.001	-0.012	0.011	0.001	GA	FR02/HV010	1768	1607.525	12690195	1385.912	1308.862	0.689054	0.683001	0.14782	0.262231	0.278461																						
63090022	HVY2 w/ coastdowns p2	00/18/18, 02:56:33 PM	HVY2 and coastdowns, 2 bag, w/ 8 Handynum, vehicle charged to 20V mode	All, saved dyno tests acquired after 4 adjustments	35	43	20	3000	43.75	-0.0404	0.02019	10.28	0.169	34.4	0.188	54.6	44.246	5.269	1.027	-0.001	-0.012	0.011	0.001	GA	FR02/HV010	1768	1607.941	12690195	1386.682	1309.816	0.687337	0.683001	0.150077	0.262231	0.278461																						
63090023	UDDS prep	00/18/18, 03:56:34 AM	UDDS prep, 1 bag, bags off, w/ 8 dyno new 20V dyno test		23	48	20	3000	43.75	-0.0404	0.02019	7.48	0.168	36.7	0.168	39.6	46.102	13.307	0.611	-1.423	-2.711	1.034	NA	FR02/HV010	1768	1233.012	2.142874	3487.851	5465.891	5.524272	0.286864	0.286864	0.171069	0.630577	0.718609																						
<b>Day 2, Certification cycles in ZWD, closed hood and variable speed fan</b>																																																									
63090024	UDDS cold start p1	00/18/18, 07:55:44 AM	EXOS2, 4 bag (PTC), cold start	IA, some delays seen in PGM diagnostic signal on random basis.	25	43	20	3000	43.75	-0.0404	0.02019	3.38	0.096	37.4	0.097	37.1	50.023	0.828	1.039	-0.282	-0.792	0.000	GA	FR02/HV010	1768	2001.911	7866939	2023.938	2040.047	2.110264	0.301602	0.301602	0.155364	0.255464	0.284634																						
63090025	UDDS cold start p2	00/18/18, 07:55:44 AM	EXOS2, 4 bag (PTC), cold start	IA, some delays seen in PGM diagnostic signal on random basis.	21	54	20	3000	43.75	-0.0404	0.02019	3.84	0.102	37.4	0.100	38.5	41.422	13.379	0.844	-1.052	-1.151	1.071	GA	FR02/HV010	1768	1233.804	0.020564	3416.473	3416.569	2.973655	0.257063	0.257063	0.160201	0.287240	0.652008																						
63090026	UDDS hot start p1	00/18/18, 07:55:44 AM	EXOS2, 4 bag (PTC), cold start	IA, some delays seen in PGM diagnostic signal on random basis.	23	49	20	3000	43.75	-0.0404	0.02019	3.20	0.097	37.6	0.098	41.1	50.001	0.470	0.815	-0.229	-0.255	0.000	GA	FR02/HV010	1768	2001.911	7866939	2023.938	2040.047	2.110264	0.301602	0.301602	0.155364	0.255464	0.284634																						
63090027	UDDS hot start p2	00/18/18, 07:55:44 AM	EXOS2, 4 bag (PTC), cold start	IA, some delays seen in PGM diagnostic signal on random basis.	22	49	20	3000	43.75	-0.0404	0.02019	3.20	0.103	37.6	0.101	38.5	43.683	13.826	1.021	0.333	-1.191	0.495	GA	FR02/HV010	1768	1233.544	1.381118	3483.802	3418.588	2.928884	0.256984	0.256984	0.178626	0.240740	0.652008																						
63090028	UDDS hot start p3	00/18/18, 07:55:44 AM	EXOS2, 4 bag (PTC), cold start	IA, some delays seen in PGM diagnostic signal on random basis.	23	48	20	3000	43.75	-0.0404	0.02019	3.20	0.098	37.6	0.098	40.4	50.000	0.454	0.815	-0.229	-0.255	0.000	GA	FR02/HV010	1768	2001.911	7866939	2023.938	2040.047	2.110264	0.301602	0.301602	0.155364	0.255464	0.284634																						
63090029	UDDS hot start p4	00/18/18, 07:55:44 AM	EXOS2, 2 bag (PTC)	IA, some delays seen in PGM diagnostic signal on random basis.	22	49	20	3000	43.75	-0.0404	0.02019	3.88	0.103	37.6	0.102	38.2	46.320	15.017	0.776	-0.061	-1.056	0.474	GA	FR02/HV010	1768	1233.544	1.381118	3483.802	3418.588	2.928884	0.256984	0.256984	0.178626	0.240740	0.652008																						
63090030	HVY2 p1	00/18/18, 09:30:05 AM	HVY 2	IA, some delays seen in PGM diagnostic signal on random basis.	26	40	20	3000	43.75	-0.0404	0.02019	10.28	0.190	54.1	0.190	54.0	59.717	5.819	-1.763	-0.903	-4.863	0.474	GA	FR02/HV010	1768	1607.525	12690195	1384.078	1336.878	0.679261	0.683231	0.130004	0.255464	0.278461																							
63090031	HVY2 p2	00/18/18, 09:30:05 AM	HVY 2	IA, some delays seen in PGM diagnostic signal on random basis.	23	41	20	3000	43.75	-0.0404	0.02019	10.28	0.196	55.2	0.196	55.2	60.426	5.401	-1.709	-0.160	-1.001	0.746	GA	FR02/HV010	1768	1608.201	2.683034	1345.966	1306.872	0.680104	0.683231	0.130004	0.255464	0.278461																							
63090032	UM04 p1	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	28	48	20	3000	43.75	-0.0404	0.02019	10.78	0.087	20.5	0.087	20.5	14.640	0.245	0.566	-0.320	-0.310	0.411	GA	FR02/HV010	1768	1607.525	12690195	1384.078	1336.878	0.679261	0.683231	0.130004	0.255464	0.278461																							
63090033	UM04 p2	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	27	40	20	3000	43.75	-0.0404	0.02019	8.24	0.194	37.9	0.190	37.7	29.820	4.000	-0.210	-0.046	-0.068	0.010	GA	FR02/HV010	1768	1608.201	1.642301	1189.092	1193.987	0.680793	0.683231	0.081887	0.267963	0.342818	0.80724																						
63090034	UM04 p3	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	28	48	20	3000	43.75	-0.0404	0.02019	10.78	0.087	20.5	0.087	20.5	14.640	0.245	0.566	-0.320	-0.310	0.411	GA	FR02/HV010	1768	1607.525	12690195	1384.078	1336.878	0.679261	0.683231	0.130004	0.255464	0.278461																							
63090035	UM04 p4	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	23	45	20	3000	43.75	-0.0404	0.02019	8.24	0.194	37.9	0.190	37.7	29.820	4.000	-0.210	-0.046	-0.068	0.010	GA	FR02/HV010	1768	1608.201	1.642301	1189.092	1193.987	0.680793	0.683231	0.081887	0.267963	0.342818	0.80724																						
63090036	UM04 p1	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	28	48	20	3000	43.75	-0.0404	0.02019	10.78	0.087	20.5	0.087	20.5	14.640	0.245	0.566	-0.320	-0.310	0.411	GA	FR02/HV010	1768	1607.525	12690195	1384.078	1336.878	0.679261	0.683231	0.130004	0.255464	0.278461																							
63090037	UM04 p2	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	27	40	20	3000	43.75	-0.0404	0.02019	8.24	0.194	37.9	0.190	37.7	29.820	4.000	-0.210	-0.046	-0.068	0.010	GA	FR02/HV010	1768	1608.201	1.642301	1189.092	1193.987	0.680793	0.683231	0.081887	0.267963	0.342818	0.80724																						
63090038	UM04 p3	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	28	48	20	3000	43.75	-0.0404	0.02019	10.78	0.087	20.5	0.087	20.5	14.640	0.245	0.566	-0.320	-0.310	0.411	GA	FR02/HV010	1768	1607.525	12690195	1384.078	1336.878	0.679261	0.683231	0.130004	0.255464	0.278461																							
63090039	UM04 p4	00/18/18, 10:27:30 AM	UM04, 4 bag	IA, some delays seen in PGM diagnostic signal on random basis.	28	48	20	3000	43.75	-0.0404	0.02019	10.78	0.087	20.5	0.087	20.5	14.640	0.245	0.566	-0.320	-0.310	0.411	GA	FR02/HV010	1768	1608.201	1.642301	1189.092	1193.987	0.680793	0.683231	0.081887	0.267963	0.342818	0.80724																						
63090040	UDDS prep	00/18/18, 03:04:26 PM	UDDS prep		28	41	20	3000	43.75	-0.0404	0.02019	7.24	0.198	37.9	0.198	37.5	29.880	4.110	-0.410	-0.006	-1.142	0.023	GA	FR02/HV010	1768	1608.201	2.683034	1345.966	1306.872	0.680104	0.683231	0.130004	0.255464	0.278461																							
<b>Day 3, Certification cycles in ZWD, closed hood and variable speed fan</b>																																																									
63090041	UDDS cold start p1	00/18/18, 07:48:26 AM	EXOS2, 4 bag (PTC), cold start, Repeat Day #1	IA, some delays seen in PGM diagnostic signal on random basis.	22	44	20	3000	43.75	-0.0404	0.02019	3.58	0.098	37.4	0.098	37.4	51.220	0.713	1.138	-0.975	1.148	0.878	NA	FR02/HV010	1768	1605.264	2.104145	2028.243	2040.025	2.607652	0.210437	0.210437	0.144801	0.135881	0.56205																						
63090042	UDDS cold start p2	00/18/18, 07:48:26 AM	EXOS2, 4 bag (PTC), cold start, Repeat Day #1	IA, some delays seen in PGM diagnostic signal on random basis.	21	54	20	3000	43.75	-0.0404	0.02019	3.82	0.098	38.1	0.098	38.1	61.000	13.862	0.822	0.643	0.401	0.606	NA	FR02/HV010	1768	1612.917	0.807726	2413.947	2414.202	2.931720	0.281208	0.281208	0.137825	0.137825	0.652008																						
63090043	UDDS hot start p1	00/18/18, 07:48:26 AM	EXOS2, 4 bag (PTC), cold start, Repeat Day #1	IA, some delays seen in PGM diagnostic signal on random basis.	22	49	20	3000	43.75	-0.0404	0.02019	3.80	0.108	38.0	0.108	38.0	61.222	11.993	0.770	0.6	0.897	0.8	NA	FR02/HV010	1768	2009.700	0.279876	2054.475	2040.018	2.728922	0.275402	0.275402	0.158970	0.158970	0.56205																						
63090044	UDDS hot start p2	00/18/18, 07:48:26 AM	EXOS2, 4 bag (PTC), cold start, Repeat Day #1	IA, some delays seen in PGM diagnostic signal on random basis.	22	49	20	3000	43.75	-0.0404	0.02019	3.80	0.108	38.1	0.108	38.1	61.222	11.993	0.770	0.6	0.897	0.8	NA	FR02/HV010	1768	2009.700	0.279876	2054.475	2040.018	2.728922	0.275402	0.275402	0.158970	0.158970	0.56205																						
63090045	UDDS hot start p3	00/18/18, 07:48:26 AM	EXOS2, 2 bag (PTC), cold start, Repeat Day #1	IA, some delays seen in PGM diagnostic signal on random basis.	23	47	20	3000	43.75	-0.0404	0.02019	3.60	0.106	38.0	0.106	38.0	60.426	15.017	0.864																																						

Text ID (R)	Cycle	Test Time	Start Comments	End Comments	Liquid Fuel Usage										Electrical Energy Use										SAE J2951 Metrics								
					Test Cell Temp (C)	Test Cell P (MPa)	Test Cell Flow (g/s)	Test Weight (lbs)	Dyno Target A	Dyno Target B	Dyno Target C	Cycle Fuel Consumption (g/s)	Cycle Fuel Economy (mpg)	Cycle Fuel Economy (liters/kg)	Fuel used (mol)	Fuel Economy (mpg)	Alternator A (W)	Alternator B (W)	12V Pwr Del (A W)	12V Pwr Del (W)	12V Reg Del (A W)	12V Reg Del (W)	Test Drive	Fuel Batch	Fuel Heating Value (Btu/lbs)	APCMA	ASCR	ASC A	ASC J	CEJ	EER	ER	RW
<b>Day 7, LA92, JC08, additional performance cycles</b>																																	
6380953	LA92 p1	06/21/18, 09:27 AM	HX002- 2 bag, cool start		24	46	20	3000	43.75	-0.8042	0.0019	3.00	0.228	3.25	33.2	37.94	9.356	1.339	0.341	1.663	0.172	MKA	GH2LTL70	1794	3074.620	4.9037920	734.0977	2922.277	1.007350	8.970293	5.25504	1.077245	0.83200
6380954	LA92 p2	06/21/18, 09:27 AM	HX002- 2 bag, prep + hot start		21	51	20	3000	43.75	-0.8042	0.0019	3.01	0.234	34.4	35.0	9.209	1.114	0.317	1.463	0.166	0.164	MKA	GH2LTL70	1794	3071.360	4.9122000	735.0082	2922.191	0.929161	8.974176	5.196284	1.077245	0.83200
6380955	JC08-2 p1	06/21/18, 09:27 AM	JC08-2 2 bag, prep + hot start		23	47	20	3000	43.75	-0.8042	0.0019	5.00	0.368	31.6	30.8	1.138	36.7	11.283	0.938	-0.247	-0.446	GTAM	GH2LTL70	1794	113.830	4.310627	3775.3360	361.2423	1.531625	3.470351	1.701728	1.568766	0.86080
6380956	JC08-2 p2	06/21/18, 09:27 AM	JC08-2 2 bag, prep + hot start		21	52	20	3000	43.75	-0.8042	0.0019	5.00	0.368	31.6	30.8	1.138	36.7	11.283	0.938	-0.247	-0.446	GTAM	GH2LTL70	1794	113.0004	4.308701	3699.8332	361.2279	1.476184	3.470351	1.568766	0.86080	0.86080
6380957	SSS 0-8@ 30rpm Mode - warmup @ 35mph	06/21/18, 10:47:08 AM	Steady State Speeds: 2 bag, 0.5@-0.5 bagp OFF, vehicle in sport mode		43	2	20	3000	43.75	-0.8042	0.0019	9.40	0.618	43.2	43.2	6.101	6.462	11.481	-0.462	-0.462	1.033	KS	GH2LTL70	1794	134.8680	4.992071	320.7331	401.744	6.487898	6.419262	2.103320	0.270225	0.719135
6380958	SSS 0-8@ 30rpm Mode - slava	06/21/18, 10:47:08 AM	Steady State Speeds: 2 bag, 0.5@-0.5 bagp OFF, vehicle in sport mode		25	42	20	3000	43.75	-0.8042	0.0019	6.22	0.381	31.2	31.1	36.444	6.864	3.505	-0.352	-0.352	1.032	KS	GH2LTL70	1794	111.1557	4.319601	3881.1664	374.2684	1.544964	4.068033	2.127047	1.200047	0.206478
6380959	Passing Maneuver w/ 0% Sport Mode	06/21/18, 11:05:21 AM	Passing maneuver for 1 bag, bagp OFF, vehicle in sport mode		24	44	20	3000	43.75	-0.8042	0.0019	3.37	0.241	23.9	23.8	17.864	5.043	-0.443	-0.443	-0.393	-0.393	KS	GH2LTL70	1794	1937.720	23.53324	3809.3491	1177.6	4.163859	4.547787	0.274400	0.480279	0.452974
6380960	UDS Prep	06/21/18, 09:28:40 AM	UDS: 1 bag, prep for CDT COV test		24	44	20	3000	43.75	-0.8042	0.0019	3.34	0.231	23.2	23.2	17.864	5.043	-0.443	-0.443	-0.393	-0.393	KS	GH2LTL70	1794	1937.330	23.53324	3809.3491	1177.6	4.163859	4.547787	0.274400	0.480279	0.452974
<b>Day 8, Certification cycles in ZWD, 20F</b>																																	
6380961	UDS cool start p1	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380962	UDS cool start p2	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380963	UDS cool start p3	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380964	UDS cool start p4	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380965	UDS cool start p5	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380966	UDS cool start p6	06/21/18, 07:38:11 AM	HX002- 4 bag (Cool CO <sub>2</sub> ), in cool test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	JK	GH2LTL70	1794	2001.361	4.797771	2044.657	2040.013	2.06667	2.19570	0.46524	0.720464	0.55325
6380967	UDS cool start p7	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-6	18	20	3000	43.75	-0.8042	0.0019	3.58	0.180	26.4	26.3	46.592	12.254	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380968	UDS cool start p8	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-7	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380969	UDS cool start p9	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-7	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380970	UDS cool start p10	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-7	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380971	UDS cool start p11	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-6	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380972	UDS cool start p12	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-6	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380973	UDS cool start p13	06/21/18, 07:38:11 AM	HX002- 4 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F; second UDS to verify fully stable temp.		-6	18	20	3000	43.75	-0.8042	0.0019	3.58	0.178	26.3	26.2	46.478	12.078	0.844	0.200	-0.100	-0.028	GT	GH2LTL70	1794	2000.538	4.800689	2008.090	2040.014	2.089314	2.19570	0.488541	0.68128	0.53820
6380974	HWV1 p1	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	10.28	0.203	50.0	51.4	78.802	7.807	1.392	0.325	-0.400	-0.088	KS	GH2LTL70	1794	1605.025	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380975	HWV1 p2	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-5	17	20	3000	43.75	-0.8042	0.0019	10.27	0.203	51.4	51.0	78.802	7.807	1.392	0.325	-0.400	-0.088	KS	GH2LTL70	1794	1605.847	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380976	HWV1 p3	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	10.28	0.203	50.0	51.4	78.802	7.807	1.392	0.325	-0.400	-0.088	KS	GH2LTL70	1794	1605.025	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380977	HWV1 p4	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	10.27	0.203	51.4	51.0	78.802	7.807	1.392	0.325	-0.400	-0.088	GT	GH2LTL70	1794	1605.847	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380978	HWV1 p5	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-6	17	20	3000	43.75	-0.8042	0.0019	10.28	0.203	50.0	51.4	78.802	7.807	1.392	0.325	-0.400	-0.088	GT	GH2LTL70	1794	1605.025	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380979	HWV1 p6	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-7	18	20	3000	43.75	-0.8042	0.0019	10.27	0.203	51.4	51.0	78.802	7.807	1.392	0.325	-0.400	-0.088	GT	GH2LTL70	1794	1605.847	4.972264	1394.894	1308.849	6.71468	6.802120	4.249654	1.973280	0.27840
6380980	HWV1 p7	06/21/18, 10:15:59 AM	HWV1 3 bag, warm start in cold test cell @ 20F; HVAC:ON AUTO 72F		-7	18	20	3000	43.75	-0.80																							

**Appendix E: Cert Fuel Specifications**

Table 21: Certificate of Analysis for Tier 3 test fuel used in tests 61809017-61809052

**haltermannsolutions**  
 Telephone: (800) 969-2542

**Certificate of Analysis**

FAX: (281) 457-1489

*REC'D 2 DRUMS  
2-6-2018*

**PRODUCT:** EPA Tier 3 EEE  
Emission Certification Fuel,  
General Testing - Regular  
**Specification No.:** HF2021

Batch No.: FH3021HW10  
 Tank No.: DRUMS  
 Date: 7/24/2017

TEST	METHOD	UNITS	SPECIFICATIONS			RESULTS
			MIN	TARGET	MAX	
Distillation - IBP	ASTM D86 <sup>2</sup>	°F				100.5
5%		°F				124.1
10%		°F	120		140	130.4
20%		°F				138.8
30%		°F				147.1
40%		°F				154.5
50%		°F	190		210	203.9
60%		°F				236.4
70%		°F				258.5
80%		°F				283.3
90%		°F	315		335	321.1
95%		°F				339.8
Distillation - EP		°F	380		420	383.6
Recovery		%		Report		97.8
Residue		ml			2.0	1.1
Loss		%		Report		1.1
Gravity @ 60 ° F	ASTM D4052 <sup>2</sup>	°API		Report		58.50
Density @ 15.56 ° C	ASTM D4052 <sup>2</sup>	kg/l		Report		0.7440
Reid Vapor Pressure EPA Equation	ASTM D5191 <sup>2</sup>	psi	8.7		9.2	8.9
Carbon	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.827
Hydrogen	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.138
Hydrogen/Carbon ratio	ASTM D5291 <sup>2</sup>	mole/mole		Report		1.989
Oxygen	ASTM D4815 <sup>2</sup>	wt %		Report		3.85
Ethanol content	ASTM D5999-00 <sup>2</sup>	vol %	9.6		10.0	10.0
Total oxygenates other than ethanol	ASTM D4815 <sup>2</sup>	vol %			0.1	None Detected
Sulfur	ASTM D5453 <sup>2</sup>	mg/kg	8.0		11.0	8.2
Phosphorus	ASTM D3231 <sup>2</sup>	g/l			0.0013	None Detected
Lead	ASTM D3237 <sup>2</sup>	g/l			0.0026	None Detected
Composition, aromatics	ASTM D5769 <sup>2</sup>	vol %	21.0		25.0	23.8
C6 aromatics (benzene)	ASTM D5769 <sup>2</sup>	vol %	0.5		0.7	0.5
C7 aromatics (toluene)	ASTM D5769 <sup>2</sup>	vol %	5.2		6.4	6.2
C8 aromatics	ASTM D5769 <sup>2</sup>	vol %	5.2		6.4	6.1
C9 aromatics	ASTM D5769 <sup>2</sup>	vol %	5.2		6.4	5.6
C10+ aromatics	ASTM D5769 <sup>2</sup>	vol %	4.4		5.6	5.5
Composition, olefins	ASTM D6550 <sup>2</sup>	wt %	4.0		10.0	5.5
Oxidation Stability	ASTM D525 <sup>2</sup>	minutes	1000			1000+
Copper Corrosion	ASTM D130 <sup>2</sup>				1	1a
Existent gum, washed	ASTM D381 <sup>2</sup>	mg/100mls			3.0	<0.5
Existent gum, unwashed	ASTM D381 <sup>2</sup>	mg/100mls		Report		1.5
Research Octane Number	ASTM D2699 <sup>2</sup>			Report		91.9
Motor Octane Number	ASTM D2700 <sup>2</sup>			Report		83.3
R+M/2	D2699/2700 <sup>2</sup>		87.0		88.4	87.6
Sensitivity	D2699/2700 <sup>2</sup>		7.5			8.6
Net Heat of Combustion	ASTM D240 <sup>2</sup>	BTU/lb		Report		17958

Quality Assurance Technician 

<sup>1</sup> Haltermann Solutions is accredited to ISO/IEC 17025 by A2LA for the tests referred to with this footnote.

<sup>2</sup> Tested by ISO/IEC 17025 accredited subcontractor.

Gasoline and diesel specialty fuels from Haltermann Solutions shall remain within specifications for a minimum of 3 years from the date on the COA so long as the drums are sealed and unopened in their original container and stored in a warehouse at ambient conditions. Specialty fuels that have been intentionally modified for aggressive or corrosive properties are excluded.

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Main Lab, 15600 West Hardy Rd., Houston, TX 77060 USA

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Table 22: Certificate of Analysis for Tier 3 test fuel used in tests 61809053 -61809066



Certificate of Analysis

FAX: (281) 457-1469

PRODUCT: **EPA Tier 3 EEE Emission Certification Fuel, General Testing - Regular**      Batch No.: GH1621LT10  
 Specification No.: **HF2021**      Tank No.: Drums      Date: 8/18/2018

TEST	METHOD	UNITS	SPECIFICATIONS			RESULTS
			MIN	TARGET	MAX	
Distillation - IBP	ASTM D86 <sup>2</sup>	°F				96.0
5%		°F				120.0
10%		°F	120		140	128.2
20%		°F				138.7
30%		°F				147.0
40%		°F				154.4
50%		°F	190		210	198.1
60%		°F				231.7
70%		°F				254.2
80%		°F				282.4
90%		°F	315		335	320.9
95%	°F				343.0	
Distillation - EP		°F	380		420	384.9
Recovery		%		Report		97.3
Residue		ml			2.0	1.2
Loss		%		Report		1.4
Gravity @ 60° F	ASTM D4052 <sup>2</sup>	°API		Report		58.13
Density @ 15.56° C	ASTM D4052 <sup>2</sup>	kg/l		Report		0.7454
Reid Vapor Pressure EPA Equation	ASTM D5191 <sup>2</sup>	psi	8.7		9.2	9.1
Carbon	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.8252
Hydrogen	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.1384
Hydrogen/Carbon ratio	ASTM D5291 <sup>2</sup>	mole/mole		Report		1.999
Oxygen	ASTM D4815 <sup>2</sup>	wt %		Report		3.64
Ethanol content	ASTM D5599-00 <sup>2</sup>	vol %	9.6		10.0	9.7
Total oxygenates other than ethanol	ASTM D4815 <sup>2</sup>	vol %			0.1	None Detected
Sulfur	ASTM D5453 <sup>2</sup>	mg/kg	8.0		11.0	8.8
Phosphorus	ASTM D3231 <sup>2</sup>	g/l			0.0013	None Detected
Lead	ASTM D3237 <sup>2</sup>	g/l			0.0026	None Detected
Composition, aromatics	ASTM D5769 <sup>1</sup>	vol %	21.0		25.0	23.2
C6 aromatics (benzene)	ASTM D5769 <sup>1</sup>	vol %	0.5		0.7	0.5
C7 aromatics (toluene)	ASTM D5769 <sup>1</sup>	vol %	5.2		6.4	6.0
C8 aromatics	ASTM D5769 <sup>1</sup>	vol %	5.2		6.4	5.9
C9 aromatics	ASTM D5769 <sup>1</sup>	vol %	5.2		6.4	5.7
C10+ aromatics	ASTM D5769 <sup>1</sup>	vol %	4.4		5.6	5.1
Composition, olefins	ASTM D6550 <sup>2</sup>	wt %	4.0		10.0	7.6
Oxidation Stability	ASTM D525 <sup>2</sup>	minutes	1000			1000+
Copper Corrosion	ASTM D130 <sup>2</sup>				1	1a
Existent gum, washed	ASTM D381 <sup>2</sup>	mg/100mls			3.0	<0.5
Existent gum, unwashed	ASTM D381 <sup>2</sup>	mg/100mls		Report		1.5
Research Octane Number	ASTM D2699 <sup>2</sup>			Report		92.0
Motor Octane Number	ASTM D2700 <sup>2</sup>			Report		84.3
R+M/2	D2699/2700 <sup>2</sup>		87.0		88.4	88.2
Sensitivity	D2699/2700 <sup>2</sup>		7.5			7.7
Net Heat of Combustion	ASTM D240 <sup>2</sup>	BTU/lb		Report		17994

Quality Assurance Technician

<sup>1</sup> Haltermann Solutions is accredited to ISO/IEC 17025 by ANAB for the tests referred to with this footnote.  
<sup>2</sup> Tested by ISO/IEC 17025 accredited subcontractor.

Gasoline and diesel specialty fuels from Haltermann Solutions shall remain within specifications for a minimum of 3 years from the date on the COA so long as the drums are sealed and unopened in their original container and stored in a warehouse at ambient conditions. Specialty fuels that have been intentionally modified for aggressive or corrosive properties are excluded.

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Table 23: Certificate of Analysis for Tier 2 test fuel used in tests 61809067-61810004

**haltermannsolutions**  
 Telephone: (800) 969-2542

**Certificate of Analysis**  
 FAX: (281) 457-1469

*REC'D 5 DRUMS  
 8-18-17  
 3-APRF  
 2-Supervisor*

PRODUCT: **EPA TIER II EEE  
 FEDERAL REGISTER  
 HF0437**

Batch No.: FC2421BE10

PRODUCT CODE:

Tank No.: Drums  
 Date: 6/23/2017

TEST	METHOD	UNITS	HALTERMANN Specs			RESULTS
			MIN	TARGET	MAX	
Distillation - IBP	ASTM D86 <sup>2</sup>	°F	75		95	87
5%		°F				111
10%		°F	120		135	125
20%		°F				145
30%		°F				167
40%		°F				195
50%		°F	200		230	218
60%		°F				231
70%		°F				240
80%		°F				258
90%		°F	305		325	312
95%		°F				339
Distillation - EP		°F			415	393
Recovery		vol %			Report	97.2
Residue		vol %			Report	1.1
Loss		vol %			Report	1.7
Gravity	ASTM D4052 <sup>1</sup>	°API	58.7		61.2	58.9
Density	ASTM D4052 <sup>1</sup>	kg/l	0.734		0.744	0.743
Reid Vapor Pressure	ASTM D5191 <sup>1</sup>	psi	8.7		9.2	9.0
Carbon	ASTM D3343 <sup>2</sup>	wt fraction		Report		0.8658
Carbon	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.8678
Hydrogen	ASTM D5291 <sup>2</sup>	wt fraction		Report		0.1322
Hydrogen/Carbon ratio	ASTM D5291 <sup>2</sup>	mole/mole		Report		1.815
Stoichiometric Air/Fuel Ratio				Report		14.533
Oxygen	ASTM D4815 <sup>2</sup>	wt %			0.05	None Detected
Sulfur	ASTM D5453 <sup>2</sup>	wt %	0.0025		0.0035	0.0028
Lead	ASTM D3237 <sup>2</sup>	g/gal			0.01	None Detected
Phosphorous	ASTM D3231 <sup>2</sup>	g/gal			0.005	None Detected
Silicon	ASTM 5184	mg/kg			4	None Detected
Composition, aromatics	ASTM D1319 <sup>2</sup>	vol %			35	29
Composition, olefins	ASTM D1319 <sup>2</sup>	vol %			10	1
Composition, saturates	ASTM D1319 <sup>2</sup>	vol %		Report		70
Particulate matter	ASTM D5452 <sup>2</sup>	mg/l			1	0
Oxidation Stability	ASTM D525 <sup>2</sup>	minutes	240			1000+
Copper Corrosion	ASTM D130 <sup>2</sup>				1	1a
Gum content, washed	ASTM D381 <sup>2</sup>	mg/100mls			5	<0.5
Fuel Economy Numerator/C Density	ASTM D5291 <sup>2</sup>		2401		2441	2436
C Factor	ASTM D5291 <sup>2</sup>			Report		1.0004
Research Octane Number	ASTM D2699 <sup>2</sup>		96.0			96.8
Motor Octane Number	ASTM D2700 <sup>2</sup>			Report		89.1
Sensitivity	D2699/2700 <sup>2</sup>		7.5			7.7
Net Heating Value, btu/lb	ASTM D3338 <sup>1</sup>	btu/lb		Report		18460
Net Heating Value, btu/lb	ASTM D240 <sup>2</sup>	btu/lb		Report		18627
Color	VISUAL			Report		Undyed

APPROVED BY: *[Signature]*

<sup>1</sup> Haltermann Solutions is accredited to ISO/IEC 17025 by A2LA for the tests referred to with this footnote.  
<sup>2</sup> Tested by ISO/IEC 17025 accredited subcontractor.

Gasoline and diesel specialty fuels from Haltermann Solutions shall remain within specifications for a minimum of 3 years from the date on the COA so long as the drums are sealed and unopened in their original container and stored in a warehouse at ambient conditions. Specialty fuels that have been intentionally modified for aggressive or corrosive properties are excluded.

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## **Appendix F: Test ID to Figure Matrix**



This appendix specifies which test IDs were used to make the figures on the report.

<b>Figure #</b>	<b>Test IDs</b>
Figure 1: Fuel economy trends: car 3,500 lb weight class	Not applicable
Figure 2: Summary distributions of weight and horsepower of mid-size cars reviewed	Not applicable
Figure 3: FTP unadjusted fuel economy of 2018 midsize vehicles	Not applicable
Figure 4: FTP unadjusted fuel economy of 2018 midsize vehicles by vehicle	Not applicable
Figure 5: Highway unadjusted fuel economy of 2018 midsize vehicles	Not applicable
Figure 6: Highway unadjusted fuel economy of 2018 midsize vehicles by vehicle selected	Not applicable
Figure 7: Drive cycle developed from on-road data for on-dyno mileage accumulation	Not applicable
Figure 8: Vehicle mounted for mileage accumulation on the AMTL 2WD chassis dynamometer	Not applicable
Figure 9: Vehicle mounted for full testing inside the AMTL 4WD chassis dynamometer	Not applicable
Figure 10: Overview of general instrumentation for conventional vehicle	Not applicable
Figure 11: Instrumentation overview of direct fuel injection system on 2018 Honda Accord	Not applicable
Figure 12: Direct fuel flow measurements via fuel scale and Coriolis flow meters	Not applicable
Figure 13: Wiring of Hioki power analyzer on the 2018 Honda Accord test vehicle	Not applicable
Figure 14: CAN breakout on the 2018 Honda Accord LX	Not applicable
Figure 15: Daily drive cycle test sequence executed in the morning	61809044
Figure 16: Overview of steady state drive cycle with preparation	61809044
Figure 17: Vehicle acceleration with varying constant pedal inputs	61810002
Figure 18: Constant acceleration ramp cycles with varying accelerator pedal inputs	61810003
Figure 19: Honda Accord test vehicle mounted to the chassis dynamometer inside the test cell.	Not applicable
Figure 20: Raw fuel economy results: UDDS and HWFET certification cycles from EPA and Argonne	TS#1: 61809027, 61809029 TS#2: 61809032, 61809034 TS#3: 61809037, 61809039 Tier 2 – 93 AKI: 61809072, 61809076
Figure 21: Test to test repeatability (UDDS AND HWFET raw fuel economy results)	61809032, 61809033, 61809034, 61809035
Figure 22: Honda Accord powertrain operation on cold start UDDS	61809032
Figure 23: Raw, uncorrected, fuel economy results for certification cycles across different temperature conditions	23C avg: • TS#1: 61809027, 61809028, 61809029, 61809030 • TS#2: 61809032, 61809033, 61809034, 61809035

Figure #	Test IDs
	<ul style="list-style-type: none"> <li>• TS#3: 61809037, 61809038, 61809039, 61809040</li> <li>-7C: <ul style="list-style-type: none"> <li>• 61809058, 61809059, 61809060, 61809061</li> </ul> </li> <li>35C: <ul style="list-style-type: none"> <li>• 61809048, 61809049, 61809050, 61809052</li> </ul> </li> </ul>
Figure 24: Engine operation on the UDDS across different temperatures	61809058, 61809032, 61809048
Figure 25: Powertrain and cabin temperature profiles across different temperature	<ul style="list-style-type: none"> <li>• -7C: 61809058, 61809059, 61809060, 61809061</li> <li>• 23C: 61809032, 61809033, 61809034, 61809035</li> <li>• 35C: 61809048, 61809049 (SC03), 61809050, 61809052</li> </ul>
Figure 26: Steady state speed operation at 23 °C, 0% grade and Tier 3 low-octane fuel	61809042
Figure 27: Steady state speed operation at 23 °C, 0% grade and Tier 2 high-octane fuel	61809068
Figure 28: Steady state speed operation at 35 °C and 0% grade	61809051
Figure 29: Steady state speed operation at -7 °C and 0% grade	61809062
Figure 30: Steady state speed operation at 23 °C and 0% grade in Sport mode	61809055
Figure 31: Steady state speed operation at 23 °C and 3% grade	61809042
Figure 32: Steady state speed operation at 23 °C and 6% grade	61809043
Figure 33: Powertrain operation during the 55 mph to 80 mph passing maneuver with low-octane fuel in Drive at 0% grade	61809045
Figure 34: Powertrain operation during the 55 mph to 80 mph passing maneuver with low-octane fuel in Sport at 0% grade	61809056
Figure 35: Powertrain operation during the 55 mph to 80 mph passing maneuver with high-octane fuel in Drive at 0% grade	61809070
Figure 36: Powertrain operation during maximum acceleration, with a focus area highlighted	61809069
Figure 37: Honda Accord continuous power test on simulated 25% grade, with a focus area highlighted	61809046
Figure 38: Powertrain operation during maximum acceleration with simulated constant gear ratios, with a focus area highlighted	61809041
Figure 39: Initial 120 s of the idle fuel flow test	61809018
Figure 40: Idle fuel flow test—full duration	61809018
Figure 41: Knock feedback signals on UDDSx2, cold start cycles	61809027, 61809032, 61809037, 61809072
Figure 42: Spark advance comparison between Tier 2 and Tier 3 fuels	Tier 3 – 88 AKI: 61809032, 61809033, 61809034, 61809035, 61809042, 618090345 Tier 2 – 93 AKI: 61809072, 61809073, 61809074, 61809075, 61809069, 61809070, 61810002

<b>Figure #</b>	<b>Test IDs</b>
Figure 43: Schematic of the vehicle configuration	Not applicable
Figure 44: Calculation of missing signals for component	Not application
Figure 45: Time spent in each gear ratio segment for UDDS/HWFET/US06 cycles	Not application
Figure 46: Torque converter lockup operation – wheel torque vs vehicle speed	Autonomie
Figure 47: Torque converter lockup operation — vehicle speed vs CVT gear ratio	Autonomie
Figure 48: Torque converter operation points for clutch engaging vs. disengaging	Autonomie
Figure 49: Operation of DFCO — vehicle speed vs wheel torque	Autonomie
Figure 50: Operation of DFCO — vehicle speed vs CVT gear ratio	Autonomie
Figure 51: Brake energy regeneration system points — engine power vs alternator power	Autonomie
Figure 52: Regenerative braking points — vehicle speed vs wheel torque	Autonomie
Figure 53: Mechanical braking power and alternator power on UDDS cycle	Autonomie
Figure 54: Engine fuel rate map according to engine speed and torque	Autonomie
Figure 55: Torque pedal map	Autonomie
Figure 56: Engine operation at vehicle start-up differs according to the engine coolant temperature	Autonomie
Figure 57: Engine idle speed is controlled by the coolant temperature	Autonomie
Figure 58: Behaviors of engine coolant temperatures on UDDS in different test conditions	Autonomie
Figure 59: Fuel rate of engine by engine power for different coolant temperatures	Autonomie
Figure 60: Accumulated fuel consumption trajectories on UDDS under different test conditions	Autonomie
Figure 61: Engine power loss and engine coolant temperature according to driving conditions	Autonomie
Figure 62: Engine output power when the vehicle is fully stopped	Autonomie
Figure 63: Electrical consumption when the vehicle is fully stopped	Autonomie

<b>Figure #</b>	<b>Test IDs</b>
Figure 64: Example of energy calculation for one component on Autonomie	Autonomie
Figure 65: Energy balance diagram on UDDS in Autonomie	Autonomie
Figure 66: Energy balance diagram on HWFET in Autonomie	Autonomie
Figure 67: Validation process for 2018 Honda Accord in Autonomie	Autonomie
Figure 68: Simulation results and test data for UDDS cycle	Autonomie
Figure 69: Simulation results and test data for HWFET cycle	Autonomie
Figure 70: Simulation results and test data for US06 cycle	Autonomie
Figure 71: Torque converter locked vs vehicle speed	Autonomie
Figure 72: Engine fuel cutoff vs vehicle speed	Autonomie

## **Appendix G: Guide to Test IDs for Analysis**

Table 24: Recommended tests for analysis

Ambient Temperature	Engine Start Condition	Cycle	Test ID (#)
Normal (23°C)	Cold	UDDS	61809032 (1 <sup>st</sup> UDDS)
		LA92	61809053 (1 <sup>st</sup> cycle)
		30 minute idle test	61809018
		Octane adjuster	61809067-61809071
	Hot	UDDS	61809032 (2 <sup>nd</sup> UDDS), 61809033
		HWFET	61809034
		US06	61809035
		Steady state speed	61809042
		LA92	61809053 (2 <sup>nd</sup> cycle)
		JC08	61809054
		Passing maneuvers @ 0, 3 & 6% grade	61809045
		25% grade test	61809046
		Wide open throttle	61809041
		Transmission mapping, constant pedal tip ins	61810002
		Transmission mapping with ramps	61810003
		Accessory load test	61810004
Hot (35°C)	Cold	UDDS	61809048 (1 <sup>st</sup> UDDS)
	Hot	UDDS	61809048 (2 <sup>nd</sup> UDDS)
		SC03	61809049
		US06	61809052
		Steady state speed	61809051
Cold (-7°C)	Cold	UDDS	61809058 (1 <sup>st</sup> UDDS)
	Hot	UDDS	61809058 (2 <sup>nd</sup> UDDS)
		HWFET	61809060
		US06	61809051

## **Appendix H: Comments From External Reviewers**

This document contains the comments from external reviewers on the vehicle testing and validation reports for the following 4 vehicles

1. Infiniti QX50, 2L Turbo VCR, CVT
2. 2019 Acura MDX Sport Hybrid, 3L V6 VTEC, 7 spd DCT
3. Toyota Camry, 2.5L I4, 8 spd AT
4. Honda Accord, 1.5L turbo VTEC, CVT

Reviewer 1

**Prof. Giorgio Rizzoni**

*Ford Motor Company Chair in ElectroMechanical Systems, is a Professor of Mechanical and Aerospace Engineering and of Electrical and Computer Engineering at Ohio State University (OSU).*

Argonne National Lab (ANL) has operated the Advanced Mobility Technology Laboratory (AMTL, formerly Advanced Powertrain Research Facility, APRF) for over 20 years. This reviewer is quite familiar with the operation and characteristics of the AMTL, having served as an Associate Technical Team Member of the Vehicle Systems Analysis Technical Team of the U.S. DRIVE Partnership between 2013 and 2016. During this time, I had the opportunity to participate in numerous program reviews of the work done by ANL-APRF in characterizing and evaluating the fuel economy, energy efficiency and emissions of a number of vehicles, mostly with focus on alternative fuels and powertrains. During the course of these reviews, it became apparent that the test capabilities and instrumentation of the AMTL are of the highest quality, and far exceed the minimum requirements for certification testing. The four-wheel-drive chassis dynamometer is operated in an environmental chamber capable of low- and high-temperature testing, and the available instrumentation permits both non-intrusive and intrusive testing to evaluate not only the fuel economy and emissions of the vehicle, but also to perform distinct and specific tests to evaluate the energy efficiency and power consumption of specific subsystems and components in the vehicle. In addition, the APRF team has developed considerable software analysis capabilities that allow the team to present results in comprehensive and carefully thought-out graphical and tabular forms. In my 35-year career as an automotive researcher, I have not come across a public-domain test facility of this kind that matches the capabilities of the AMTL. The work presented in this report is of the highest quality.

The test plan is quite comprehensive, designed to address specific questions related to the fuel economy impact of the operation of various automotive subsystems, and far exceeds the minimum requirements of certification testing. I have no suggestions for further improvement.

The tests conducted in the study were comprehensive and evaluated vehicle fuel economy under different environmental conditions (72, 20, and 95 °F, the last with solar radiation emulation), and with fuels with different octane ratings (regular and premium). In addition to performing fuel economy tests following regulatory driving cycles (UDDS, HWFET, US06, and SC03, LA92 and JCo8), the testing included steady speed tests at different grades, tests during passing maneuvers, and wide-open throttle and idle fuel consumption tests. The test program is as comprehensive as one could expect to implement in a chassis dynamometer test cell. The comparison with EPA CAFE test results is very valuable.



The graphical and tabular summary of the test results give a clear and concise representation of the results. I made some recommendations on minor improvements that I believe will be incorporated in the final report. The only item that is important to note is the lack of consistency in the units used throughout the report. This is an industry-wide problem, wherein SI and English units are both used and not always both shown next to one another.

The energy analysis, including both fuel economy and overall efficiency, is comprehensive and includes consideration of thermal environment (both ambient temperature as well as cold and hot start conditions), and of different vehicle modes of operation (accel/decel, cruise, stop). The visual presentation of these results is excellent and gives the reader the opportunity to understand the results of complex tests.

As part of the peer review process, I took the time to carefully review the report, and made a number of editorial suggestions that, in my opinion, further enhanced the already excellent quality of the report. I believe that the final product is a well-organized, readable, clear and accurate report.

### **Vehicle specific comments:**

Infinity QX50:

This report provides testing results for a 2019 Infiniti QX50 equipped with a turbocharged 2.0 liter in-line four-cylinder Variable Compression Ratio (VCR) Atkinson cycle-capable engine with dual fuel injection, coupled to the driveline by a CVT. The combination of features in this powertrain is novel, to best of this reviewer's knowledge, and is a very appropriate choice for testing and analysis at Argonne.

The additional analysis presented in the report on: details of VCR engine operation; dual fuel injection strategies; transmission operating strategy; torque converter lock-up strategies; vehicle performance (acceleration and passing maneuvers); fuel cut-off strategies; cycle thermal test conditions; comparison of fuels with different AKI ratings; and accessory load operation further enhances the quality and completeness of the report. The Autonomie Model Validation section is a valuable addition to the testing results and is very well executed.

Acura MDXSH

This report provides testing results for a 2019 Acura MDX Sport Hybrid equipped with a 3.0 V6 Variable Valve Timing and Lift Electronic Control (VTEC) engine coupled through a 7-speed dual clutch transmission (DTC) and a three-motor hybrid system. The 2019 Acura MDX sport hybrid "super-handling" all-wheel drive (SH-AWD) system includes a 143-kW engine coupled to a 7-speed dual clutch transmission (DCT) and a 35-kW electric motor in the front and two 27-kW electric motors on the rear axle, capable of driving each wheel independently, thus replacing the rear differential. The 3.0L V6 engine is port fuel injected and can perform cylinder deactivation for each bank to achieve higher low-load efficiencies. The configuration of the rear electric machines permits the implementation of torque-vectoring strategies and enable superior vehicle handling. This choice of this vehicle is appropriate as it represents a trend towards achieving improved fuel economy while also providing improved performance.

Camry:

The vehicle tested in this report is equipped with a 2.5 L, in-line, four-cylinder engine coupled to an 8-speed automatic transmission. The engine is a high-expansion-ratio Atkinson cycle engine

with very high peak thermal efficiency (40%), dual variable valve timing, cooled EGR. The 8-speed transmission is a new development that replaces the previously employed 6-speed transmission. The vehicle is claimed to offer outstanding fuel economy while delivering impressive performance. The results presented in the report clearly support these statements and suggest that the technologies embodied in this vehicle are representative of future trends for conventional (i.e., non-hybrid) powertrains in mid-size sedans.

#### Accord

The vehicle tested in this report is equipped with a best-in-class powertrain featuring a turbocharged 1.5 L, in-line, four-cylinder engine with variable valve timing and lift electronic control (VTEC) paired with a direct injection system and a continuously variable transmission. The Honda's VTEC turbo technology is marketed as part of the powertrain technologies marketed by Honda as "Earth Dreams Technology." The vehicle is claimed to offer outstanding fuel economy while delivering impressive performance. The results presented in the report clearly support these statements and suggest that the technologies embodied in this vehicle are representative of future trends for conventional (i.e., non-hybrid) powertrains in mid-size sedans.

The additional analysis presented in the report on: transmission and torque converter operating strategy (including different transmission operating modes); vehicle performance (acceleration and passing maneuvers); start-stop operation; vehicle fuel injection strategies; fuel cut-off strategies; cycle thermal test conditions; comparison of fuels with different AKI ratings; and accessory load operation further enhances the quality and completeness of the report. The Autonomie Model Validation section is a valuable addition to the testing results, and is very well executed.

Reviewer 2

**Prof. David Foster**

*Phil and Jean Myers Professor Emeritus,  
Department of Mechanical Engineering, University of Wisconsin-Madison*

The experimental protocols and quality of the data taken is very good. It was also nice to see the extra dyno test runs that were developed to probe the vehicle control systems and performance for a more extensive range of operating conditions than the standardized certification tests. The use of this data to fit the Autonomie simulation was impressive as were the correlations between the simulation predictions and the certification cycle test data. Very nice work.

I have made many comments throughout the four reports. Some were generic to the descriptions of the experimental procedure and simulation tuning. Relative to these comments, I sometimes repeated them in the individual reports and other times merely said I had made a comment on the item being described in one of the reports previously reviewed. I hope that the individual teams will share the generic comments about operating procedure, etc. with each other.

Finally, I also had suggestions which I thought would increase the impact of this work. I think that the detail of the operating characteristics of the specific components of each vehicles powertrain contained in Autonomie puts you are in a position to quantify the incremental improvement each of the advanced powertrain technologies makes in the vehicles' fuel economy and performance relative to previous model vehicles as well as competitor vehicles. This is what I expected as part of the discussion on the insights gained from vehicle testing. I inferred this from reading the contract statement: "The focus of the evaluation was to understand the use of critical powertrain components and their impact on the vehicle efficiency," given in the introduction and/or conclusion of each report. In conclusion of each report I made an extended comment further detailing this thought – usually with specific reference to the technologies used in the vehicle reported on in the report.

Below is a copy of my conclusive comment from the Acura Performance Report:

*"This is a similar comment to that made in the reports I have previously reviewed.*

*This is very good work. The experimental protocol, procedures and data taking techniques are of high quality. The component data extracted from the tests were used to tune Autonomie which was then used to simulate the vehicle with excellent results.*

*The reporting of the data in this report was pretty much just that; here is the data we got; we can see the different aspect of the powertrain engaging and disengaging; here are the results for the two different octane fuels that were tested, etc. However, there was very little discussion of, or attempts to quantify, the impact on fuel economy and performance improvement of the individual advanced technologies used in the vehicle. Also, to me it was disconcerting that when the testing showed no difference between the manufacturer's recommended high-octane fuel and the less expensive low octane fuel almost no discussion ensued. To me this was a significant finding.*

*I think you are well situated to make these assessments. The Autonomie simulation has energy flows and performance evaluation criteria for most, if not all, of the components and subsystems of the vehicle. I thought it would be possible to use the simulation, which reproduces the data well, to partition the energy flow from the fuel to the wheels for the various driving conditions tested and quantify the impact of the different technologies on fuel economy and performance.*

*By doing this for the different vehicles tested you would be able to offer a look-up type categorization of the potential benefits of different technologies, used either separately or synergistically, on overall vehicle performance.*

*Such an analysis would be a tremendous contribution to the technical and regulatory community, and it is what I inferred what the NHTSA was interested in. It is why I offer this comment on the highlighted phrase.”*

The testing of the impact of the fuels octane number was particularly surprising. In general the octane number did not make a significance difference in the vehicles performance. In fact in the Acura, where the manufacturer recommends high octane gasoline, the low octane gasoline showed better performance. This is a significant finding which I do not understand. It was not discussed in any detail in the report.

There is no reason to discount the data in your tests. However, if this is true, why would the manufacturers recommend high octane gasoline when better performance could be obtained with a less expensive fuel? I made comments of this nature in the different vehicle reports because I think this is a significant finding. It is also one that your laboratory should make absolutely sure that nothing is strange with the data. I even suggested asking Honda about this. To that end, I think one needs to be sure that there are no caveats to this data before it is disseminated more widely in the public arena. This result is significant!

For more detail on this I am also including the extended comment I made in the fuels testing section of the Acura Performance report:

*“Considering these tests relative to the fuel test results given in the Infinity makes me more confused. It seems to me that the most important test to perform for this evaluation is the one using the manufacturer’s recommended octane rating fuel – which should be the focus of your results.*

*If the manufacturer recommends the lower octane fuel isn’t it safe to assume that they have optimized the engine for the lower-octane fuel, and have not included technologies that would optimize for higher octane? For example, the range of spark advance might be limited, the chosen compression ratio might not be optimal if a higher-octane fuel were used, .... In other words, using a high-octane fuel could very well result in significant knock margin being ‘left on the table’ because of this non optimal operation. In which case it would be easy to interpret results of such tests out of context and come to a more general conclusion that higher octane is not worth very much.*

*I commented in the Infinity testing that an opportunity may have been missed by not running a lower octane fuel in the vehicle which specifies high-octane. It might more clearly inform us on the magnitude of performance improvements that are available through the use of a high-octane fuel in a vehicle which has been optimized of that fuel. Or conversely, it could inform us of the performance degradation that will be experienced from using a low octane fuel in a vehicle designed for high octane fuel.*

*For this vehicle it appears that you are doing what I suggested in the Infinity report. (Although because of confusion in how the fuel specifications are given in Appendix D, I got confused trying to interpret the results.) I was hoping your data, when combined with the fuel testing data from the other vehicle performance evaluations, would show the performance detriments that may occur when an engine optimized for higher octane fuel is run on low octane fuel. It could*

*also give information about using a lower octane fuel in an engine optimized for high octane relative to the performance of an engine/vehicle optimized for a lower octane number fuel using the low octane fuel. And finally, it could assess if there is any benefit to using a high-octane fuel in an engine optimized for low octane.*

*Partitioning these efficiency contributions of both engine technology and fuel specifications would be a significant contribution to the larger technical community, regulatory agencies, and the public in general.”*

Perhaps the level of energy flow partitioning I was hoping for is outside of the scope of the contract with NHTSA. If it is, fine, but I still think these data and the subsequent Autonomie simulation capabilities give ANL a unique opportunity to offer some quantification of the efficiency improvement potential for a wide array of advanced technology components that are being incorporated into new vehicles.

Reviewer 3

**Prof. Douglas Nelson**

*Department of Mechanical Engineering, Virginia Tech*

**Comments on Toyota Camry report:**

The ANL report documents vehicle testing and model development for the 2018 Toyota Camry XLE 2.5L

PFI/DI engine coupled to an eight-speed automatic transmission. This vehicle was selected to evaluate these technologies and to develop models in support of NHTSA's CAFE work. Overall, the report is of high quality and achieves the objectives set out in the report. The following comments are intended to help improve the report.

The report should add an Executive Summary that clearly states the results of the report. The Conclusions should also be revised and extended to include what is significant about the results; does the work provide new and better data, models, and control? Does this engine have improved efficiency beyond previous versions of direct and port fuel injection engines? Does the Atkinson cycle used in a conventional vehicle rather than a hybrid have any issues with operation of the engine?

The given reference [8] does not seem to be available (yet?) to the public. The data provided in the report is of very high quality and high value, but the errors and uncertainty are not adequately addressed. The excellent repeatability of some data has been shown. Even if the details are provided in [8] a brief summary of the overall testing data quality/uncertainty should be included in the report.

**Comments on Infiniti QX50 report**

The ANL report documents vehicle testing and model development for the 2019 Infiniti QX50 2.0L variable compression ratio (VCR) turbocharged engine coupled to a continuously variable transmission (CVT). This vehicle was selected to evaluate these technologies and to develop models in support of NHTSA's CAFE work. Overall, the report is of high quality and achieves the objectives set out in the report. The following comments are intended to help improve the report.

The Executive Summary should clearly state the results of the modeling and validation sections of the report. The Conclusions should also be revised and extended to include what is significant about the results; does the work provide new and better data, models, and control? Does this engine have improved efficiency beyond previous versions of direct and port fuel injection engines? Does the Atkinson cycle used in this conventional vehicle rather than a hybrid have any issues with operation of the engine? What are the advantages of VCR for efficiency vs performance? The given reference [4] does not seem to be available (yet?) to the public. The data provided in the report is of very high quality and high value, but the errors and uncertainty are not adequately addressed. The excellent repeatability of some data has been shown. Even if the details are provided in [4] a brief summary of the overall testing data quality/uncertainty should be included in the report.

Overall, the testing sections have good documentation and presentation of the complex interactions of VCR, boost, DI and ignition timing. The following comments are provided in the order of the report, and are not in any order of significance. In several places in the vehicle

comparison, the term “adjusted” fuel economy is used. The fuel economy results available from the EPA test car list (tcl) data (as referenced) are broadly understood to be unadjusted values that correspond to specific drive cycles and phases, while the label fuel economy available from fueconomy.gov are adjusted. CAFE is based on unadjusted fuel economy directly available from the EPA test car list data. That tcl data does have a header that says RND\_ADJ\_FE, but that ADJ is not in the same context. If you use the term adjusted with respect to the tcl data, please very specifically define what the adjustment means in this context. Is it the weighting of the cold start and hot start phases 1 and 3 of the UDDS test results to get the FTP? Then why are HwFET results also (sometimes) referenced as adjusted? Please just be very clear about this term as there is a lot of confusion about CAFÉ vs Label fuel economy.

The mix of using superscripted numbers for both footnotes and references is a bit confusing – suggest using references in [#] format as in the other reports.

### **Comments on the Accord report**

The ANL report documents vehicle testing and model development for the 2018 Honda Accord LX 1.5L turbocharged engine coupled to a continuously variable transmission (CVT). This vehicle was selected to evaluate these technologies and to develop models in support of NHTSA’s CAFE work. Overall, the report is of high quality and achieves the objectives set out in the report. The following comments are intended to help improve the report.

The report should add an Executive Summary that clearly states the results of the report. The conclusions should also be revised and extended to include what is significant about the results; does the work provide new and better data, models, and control? Does this engine have improved efficiency beyond previous versions of turbocharged four-cylinder engines? Does the CVT have reduced losses in addition to improving the operation of the engine?

The given reference [8] does not seem to be available (yet?) to the public. The data provided in the report is of very high quality and high value, but the errors and uncertainty are not adequately addressed. The excellent repeatability of some data has been shown. Even if the details are provided in [8] a brief summary of the overall testing data quality/uncertainty should be included in the report.

### **Comments on Acura MDXSH**

The ANL report documents vehicle testing and model development for the 2019 Acura MDX SH 3.0L VTEC engine coupled to a 7-speed dual clutch transmission and a 3-motor hybrid electric system. This AWD hybrid vehicle was selected to evaluate these technologies and to develop models in support of NHTSA’s CAFE work. Overall, the report is of high quality and achieves the objectives set out in the report. The following comments are intended to help improve the report.

The Executive Summary should clearly state the results of the modeling and validation sections of the report. The Conclusions should also be revised and extended to include what is significant about the results; does the work provide new and better data, models, and control? Does this hybrid vehicle have improved engine efficiency beyond previous hybrids? Does the DCT with integrated motor have significant fuel consumption benefits? What are the advantages of rear motors for efficiency vs performance?

The given reference [4] does not seem to be available (yet?) to the public. The data provided in the report is of very high quality and high value, but the accuracy and uncertainty are not

adequately addressed. The excellent repeatability of some data has been shown. Even if the details are provided in [4] a brief summary of the overall testing data quality/uncertainty should be included in the report.

Overall, the testing sections have good documentation and presentation of the complex interactions of hybrid strategy and components.



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